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THEORY AND APPLICATION OF MATHEMATICAL MODELING
OF SHOULDER-FIRED WEAPONS

PART I: M16A1 RIFLE

BY

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FINAL REPORT



RESEARCH DIRECTORATE

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Now define the following linear operations:

$$L_0 = -C \frac{d(\)}{dt} - K$$

$$L_1 = m \frac{d^2(\)}{dt^2} + (C+C_\infty) \frac{d(\)}{dt} + (K+K_\infty)$$

$$L_2 = m \frac{d^2(\)}{dt^2} + 2C \frac{d(\)}{dt} + 2K$$

Then equations (C-7), (C-8), and (C-9) can be rewritten as:

$$\begin{aligned} L_1 X_1 + L_0 X_2 + 0X_3 + 0X_4 + \dots + 0X_n &= K_\infty (X_{FP} - X_{FP_0}) + C_\infty \dot{X}_{FP} \\ L_0 X_1 + L_2 X_2 + L_0 X_3 + 0X_4 + \dots + 0X_n &= 0 \\ 0X_1 + L_0 X_2 + L_2 X_3 + L_0 X_4 + \dots + 0X_n &= 0 \\ \vdots & \\ \vdots & \\ 0X_1 + 0X_2 + 0X_3 + 0X_4 + L_0 X_{n-1} + L_1 X_n &= K_\infty (X_{MG} - X_{MG_0}) + C_\infty \dot{X}_{MG} \end{aligned} \tag{C-10}$$

Now take Laplace transforms of $L_0 X(t)$, $L_1 X(t)$, $L_2 X(t)$

$$\mathcal{L}\{L_0[X(t)]\} = -C [S\bar{X}(S) - X(0) - K\bar{X}(S)]$$

$$\begin{aligned} \mathcal{L}\{L_1[X(t)]\} &= m [S^2\bar{X}(S) - SX(0) - \frac{dx}{dt}(0)] \\ &+ (C+C_\infty)[S\bar{X}(S) - X(0)] + (K+K_\infty) \bar{X}(S) \end{aligned}$$

$$\begin{aligned} \mathcal{L}\{L_2[X(t)]\} &= m [S^2\bar{X}(S) - SX(0) - \frac{dx(0)}{dt}] \\ &+ 2C [S\bar{X}(S) - X(0)] + 2K\bar{X}(S) \end{aligned}$$

Assume $X(0) = \dot{X}(0) = 0$

$$\mathcal{L}\{L_0[X(t)]\} = -C [S + \frac{K}{C}] \bar{X}(S) \equiv \alpha_0(S) \bar{X}(S)$$

$$\mathcal{L}\{L_1[X(t)]\} = [mS^2 + (C+C_\infty)S + (K+K_\infty)] \bar{X}(S) \equiv \alpha_1(S) \bar{X}(S)$$

$$\mathcal{L}\{L_2[X(t)]\} = [mS^2 + 2CS + 2K] \bar{X}(S) \equiv \alpha_2(S) \bar{X}(S)$$

Also define

$$\mathcal{L}\{K_\infty[X_{FP} - X_{FP_0}] + C_\infty[\dot{X}_{FP}]\} \equiv g_1(S)$$

$$\mathcal{L}\{K_\infty[X_{MG} - X_{MG_0}] + C_\infty[\dot{X}_{MG}]\} \equiv g_n(S)$$

Now take the Laplace transforms of Equation (C-10)

$$\begin{aligned} \alpha_1(s) \bar{X}_1(s) + \alpha_0(s) \bar{X}_2(s) + 0\bar{X}_3(s) + 0\bar{X}_4(s) + \dots + 0\bar{X}_n(s) &= g_1(s) \\ \alpha_0(s) \bar{X}_1(s) + \alpha_2(s) \bar{X}_2(s) + \alpha_0(s) \bar{X}_3(s) + 0\bar{X}_4(s) + \dots + 0\bar{X}_n(s) &= 0 \\ 0 \bar{X}_1(s) + \alpha_0(s) \bar{X}_2(s) + \alpha_2(s) \bar{X}_3(s) + \alpha_0(s) \bar{X}_4(s) + \dots + 0\bar{X}_n(s) &= 0 \\ \vdots & \\ 0 \bar{X}_1(s) + 0 \bar{X}_2(s) + 0 \bar{X}_3(s) + 0 \bar{X}_4(s) + \dots + \alpha_0(s) \bar{X}_{n-1}(s) + \alpha_1(s) \bar{X}_n(s) &= g_2(s) \end{aligned}$$

Consider $g_1(s)$ and $g_2(s)$ as known. Such an assumption implies that X_{FP} and X_{MG} are given.

$$\begin{bmatrix} \alpha_1 & \alpha_0 & 0 & 0 & \dots & 0 \\ \alpha_0 & \alpha_2 & \alpha_0 & 0 & \dots & 0 \\ 0 & \alpha_0 & \alpha_2 & \alpha_0 & \dots & 0 \\ \vdots & \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & 0 & 0 & \dots & \alpha_0 \alpha_1 \end{bmatrix} \begin{bmatrix} \bar{X}_1(s) \\ \bar{X}_2(s) \\ \bar{X}_3(s) \\ \vdots \\ \bar{X}_n(s) \end{bmatrix} = \begin{bmatrix} g_1(s) \\ 0 \\ 0 \\ \vdots \\ g_2(s) \end{bmatrix}$$

Solve for $\bar{X}_1(s)$ using Cramer's rule. The denominator of the resulting expression for $\bar{X}_1(s)$ is:

$$\det[\alpha] = \alpha_1 \left[\alpha_1 \frac{U^{n-1} - V^{n-1}}{U-V} - \alpha_0^2 \frac{U^{n-2} - V^{n-2}}{U-V} \right] - \alpha_0^2 \left[\alpha_1 \frac{U^{n-2} - V^{n-2}}{U-V} - L_0^2 \frac{U^{n-3} - V^{n-3}}{U-V} \right]$$

$$\det[\alpha] = \frac{U^{n-3}}{U-V} [U\alpha_1 - \alpha_0^2]^2 - \frac{V^{n-3}}{U-V} [\alpha_1 V - \alpha_0^2]$$

$$\text{where } U \equiv \frac{\alpha_2 + \sqrt{\alpha_2^2 - 4\alpha_0^2}}{2} \quad v \equiv \frac{\alpha_2 - \sqrt{\alpha_2^2 - 4\alpha_0^2}}{2}$$

In Reference ¹¹, it is shown that the determinate of a MXM triple-banded matrix is:

$$\begin{vmatrix} A_2 & A_0 & 0 & 0 & 0 & \cdots & 0 & 0 \\ A_0 & A_2 & A_0 & 0 & 0 & \cdots & 0 & 0 \\ 0 & A_0 & A_2 & A_0 & 0 & \cdots & 0 & 0 \\ \vdots & \vdots & \vdots & \vdots & \vdots & \ddots & \vdots & \vdots \\ 0 & 0 & 0 & 0 & \cdots & A_0 & A_2 & A_0 & 0 \\ 0 & 0 & 0 & 0 & \cdots & 0 & A_0 & A_2 & A_0 \\ 0 & 0 & 0 & 0 & \cdots & 0 & 0 & A_0 & A_2 \end{vmatrix} = \frac{U^{m+1} - V^{m+1}}{U-V}$$

where U and V are the roots of $X^2 - A_2 X - A_0^2 = 0$

By this fact, the surprisingly simple result for det [α] was made possible.

The numerator of the Cramer's rule solution for $\bar{X}_1(S)$ is:

$$g_1 [\alpha_1 K_{n-2} - \alpha_0^2 K_{n-3}] + (-1)^{n+1} g_2 \alpha_0^{n-1}$$

$$\text{where } K_i \equiv \frac{U^{i+1} - V^{i+1}}{U-V}$$

thus

$$\bar{X}_1(S) = \frac{(-1)^{n+1} g_2 \alpha_0^{n-1} + g_1 \left[\alpha_1 \frac{U^{n-1} - V^{n-1}}{U-V} - \alpha_0^2 \frac{U^{n-2} - V^{n-2}}{U-V} \right]}{\frac{U^{n-3}}{U-V} [U \alpha_1 - \alpha_0^2]^2 - \frac{V^{n-3}}{U-V} [\alpha_1 V - \alpha_0^2]^2}$$

$$\bar{X}_1(S) = \frac{(-1)^{n+1} g_2 \alpha_0^{n-1} (U-V) + g_1 U^{n-2} (\alpha_1 U - \alpha_0^2) - V^{n-2} g_1 (\alpha_1 V - \alpha_0^2)}{U^{n-3} [\alpha_1 U - \alpha_0^2]^2 - V^{n-3} [\alpha_1 V - \alpha_0^2]^2}$$

¹¹Muir, Thomas, A Treatise on the Theory of Determinants, Dover Publication, pp 564.

One finds a similar expression for the last coil. If these are rewritten in the K_i -notation, the result is:

$$\bar{X}_1(s) = \frac{(-1)^{n+1} g_2 \alpha_0^{n-1} + g_1 (\alpha_1 K_{n-2} - \alpha_0^2 K_{n-3})}{\alpha_1^2 K_{n-2} - 2\alpha_1 \alpha_0^2 K_{n-3} + \alpha_0^4 K_{n-4}}$$

$$\bar{X}_n(s) = \frac{(-1)^{n+1} g_1 \alpha_0^{n-1} + g_2 (\alpha_1 K_{n-2} - \alpha_0^2 K_{n-3})}{\alpha_1^2 K_{n-2} - 2\alpha_1 \alpha_0^2 K_{n-3} + \alpha_0^4 K_{n-4}}$$

The great difficulty is in taking the inverse transforms. If one can find the inverse transform of $\bar{X}_1(s)$, one can then easily evaluate the spring force acting on the firing pin if the motion of the firing pin is given. Many methods for finding inverse transforms were used. The following expression is shown in Reference¹².

$$f(t) = \mathcal{L}^{-1} [\bar{f}(s)] = e^{-t/2} \sum_{k=0}^{\infty} \frac{C_k L_k(t)}{k!}$$

where $C_k = \sum_{j=0}^k \binom{k}{j} \frac{1}{j!} \bar{f}^j \left(\frac{1}{2}\right) \quad k=0, 1, 2, \dots$

$$L_k(t) = e^t \frac{d^k}{dt^k} (t^k e^{-t}) = (-1)^k [X^k - k^2 X^{k-1} + \frac{k^2(k-1)^2}{2!} X^{k-2} + \dots]$$

The $K_k(t)$ are Laguerre polynomials.

The above method did not appear to converge. Because of time consideration, the transfer function approach was temporarily abandoned, and a direct solution of the equation was sought. However, the finding of a transfer function for such a mechanical system is of such importance to weapon modeling that efforts should be resumed; perhaps Fourier transforms could also be tried.

¹²Korn, G.A. and Korn, T.M., Mathematical Handbook for Scientists and Engineers; definitions, theorems, and formulas for reference and review, McGraw-Hill, New York, N.Y. (1961), pg 226.

In solving the equations directly, the approach was to assume that the right side of the equation for the i th coil,

$$M\ddot{X}_i + 2C\dot{X}_i + 2KX_i = K(X_{i-1} + X_{i+1}) + C(\dot{X}_{i-1} + \dot{X}_{i+1}),$$

is constant within any given time increment of integration. It can, of course, vary from increment-to-increment. With the right side thus held constant temporarily, homogeneous and particular solutions are easily obtained for the i th coil, and one can evaluate its position and velocity at the end of each time interval so that these become initial conditions for the next time increment. The method was found to give more accurate results than the fourth-order Runge-Kutta algorithm in less than one-half the computation time. Comparisons were made between these two methods, and the exact solutions for two and three masses.

A derivation of the method now will be presented. Assume, for simplicity, that $c_\infty = c$ and $K_\infty = K$, and rewrite the equations as follows:

$$M\ddot{X}_1 + 2KX_1 + 2C\dot{X}_1 = KX_2 + C\dot{X}_2 + K(X_{FP} - X_{FP0}) + C\dot{X}_{FP} = h_1 \quad (C-11)$$

$$\vdots$$

$$M\ddot{X}_i + 2KX_i + 2C\dot{X}_i = K(X_{i+1} + X_{i-1}) + C(\dot{X}_{i+1} + \dot{X}_{i-1}) = h_i \quad (C-12)$$

$$\vdots$$

$$M\ddot{X}_n + 2KX_n + 2C\dot{X}_n = KX_{n-1} + K(X_{MG} - X_{MG0}) + C(\dot{X}_{MG} + \dot{X}_{n-1}) = h_n \quad (C-13)$$

Integrate these equations from $t=t_1$ to $t=t_1 + \Delta t$. During this interval, assume that the right side of each equation does not change. The homogeneous equations are of the form

$$M\ddot{X}_h + \alpha \dot{X}_h + \beta X_h = 0,$$

where

$$\alpha = 2C, \beta = 2K.$$

Assume homogeneous solutions of the form $X = e^{rt}$.

Substitution of the assumed solution into the homogeneous equation yields the following:

$$mr^2 + \alpha r + \beta = 0$$

$$r = \frac{-\alpha \pm \sqrt{\alpha^2 - 4m\beta}}{2m}$$

Three cases exist that are analogous to the underdamped, overdamped, and critically-damped cases for a single mass-spring-dashpot system.

CASE I

$\alpha^2 - 4m\beta < 0$ or $\frac{2K}{m} - \left(\frac{C}{m}\right)^2 > 0$ This case corresponds to underdamping.

$$r = -\frac{\alpha}{2m} \pm i \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2}$$

$$X_h = e^{-\frac{\alpha}{2m}t} \left[A e^{i \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t} + B e^{-i \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t} \right]$$

$$X_h = e^{-\frac{\alpha}{2m}t} \left[(A+B) \cos \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t + i (A-B) \sin \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t \right]$$

$$X_h = e^{-\frac{\alpha}{2m}t} \left[C_1 \cos \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t + C_2 \sin \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t \right]$$

Now, find a particular solution to:

$$m\ddot{X}_j + \alpha \dot{X}_j + \beta X_j = h_j = \text{constant} = h_j(t=0).$$

$$X_{Pj} = \frac{h_j}{\beta}$$

Thus, the complete solution to equations (C-11), (C-12), (C-13) is:

$$X_j = e^{-\frac{\alpha}{2m}t} \left[C_{1j} \cos \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t + C_{2j} \sin \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t \right] + \frac{h_j(t=0)}{\beta} \quad (C-14)$$

C_{1j} and C_{2j} are determined from the initial conditions

$$X_j(t_1) = P_j \quad (C-15)$$

and

$$\dot{X}_j(t_1) = V_j. \quad (C-16)$$

For simplicity, equation (C-14) will be applied individually over each time increment. Thus, t_1 will be equated to zero; and P_j and V_j will be equated to the respective values at the end of the previous time increment. A separate accounting is made of the true value of t as the solution progresses.

Application of the initial condition (C-15) yields:

$$P_j = C_{1j} + \frac{h_j(t=0)}{\beta} \quad (C-17)$$

$$C_{1j} = P_j - \frac{h_j(t=0)}{\beta} \quad (C-18)$$

Now,

$$\begin{aligned} \dot{x}_j = & -\frac{\alpha}{2m} e^{-\frac{\alpha}{2m} t} \left[C_{1j} \cos \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t \right. \\ & \left. + C_{2j} \sin \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t \right] \\ & + e^{-\frac{\alpha}{2m} t} \left[-C_{1j} \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} \sin \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t \right. \\ & \left. + C_{2j} \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} \cos \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t \right] \end{aligned}$$

Application of (C-16) yields:

$$V_j = -\frac{\alpha}{2m} C_{1j} + C_{2j} \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2}$$

$$V_j = -\frac{\alpha}{2m} \left[P_j - \frac{h_j(t=0)}{\beta} \right] + C_{2j} \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2}$$

$$C_{2j} = \frac{1}{\sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2}} \left[V_j + \frac{\alpha}{2m} \left(P_j - \frac{h_j(t=0)}{\beta} \right) \right] \quad (C-19)$$

From (C-14), (C-18), and (C-19)

$$\begin{aligned}
 x_j = e^{-\frac{\alpha}{2m} t} & \left\{ \left(P_j - \frac{h_1(t=0)}{\beta} \right) \cos \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t \right. \\
 & \left. + \frac{1}{\sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2}} \left[v_j + \frac{\alpha}{2m} \left(P_j - \frac{h_1(t=0)}{\beta} \right) \right] \sin \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t \right\} \\
 & + \frac{h_1(t=0)}{\beta}
 \end{aligned} \tag{C-20}$$

$$\begin{aligned}
 x_j(t = t_1 + \Delta t) = e^{-\frac{C\Delta t}{m}} & \left\{ \left(P_j - \frac{h_1(t=t_1)}{2K} \right) \cos \left(\sqrt{\frac{2K}{m} - \left(\frac{C}{m}\right)^2} \Delta t \right) \right. \\
 & \left. + \frac{1}{\sqrt{\frac{2K}{m} - \left(\frac{C}{m}\right)^2}} \left[v_j + \frac{C}{m} \left(P_j - \frac{h_1(t=t_1)}{2K} \right) \right] \sin \left(\sqrt{\frac{2K}{m} - \left(\frac{C}{m}\right)^2} \Delta t \right) \right\} \\
 & + \frac{h_1(t=t_1)}{2K}
 \end{aligned} \tag{C-21}$$

P_j and v_j are position and velocity of the j th mass as calculated at the end of the previous time interval.

Since the value of \dot{x}_j at the end of each time interval is needed as an initial condition for the next increment, \dot{x}_j will now be calculated:

$$\dot{x}_j = e^{-\frac{\alpha}{2m} t} \left\{ \left[-\frac{\alpha}{2m} c_{1j} + c_{2j} \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} \right] \cos \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t \right. \tag{C-22}$$

$$\begin{aligned}
 & \left. - \left[\frac{\alpha}{2m} c_{2j} + c_{1j} \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} \right] \sin \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t \right\} \\
 \dot{x}_j = e^{-\frac{\alpha}{2m} t} & \left\{ \left[-\frac{\alpha}{2m} \left(P_j - \frac{h_1}{\beta} \right) + \frac{1}{\sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2}} \left(v_j + \frac{\alpha}{2m} \left(P_j - \frac{h_1}{\beta} \right) \right) \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} \right] \right. \\
 & \cos \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t + \left[-\frac{\alpha}{2m} \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} \left(v_j + \frac{\alpha}{2m} \left(P_j - \frac{h_1}{\beta} \right) \right) \right. \\
 & \left. - \left(P_j - \frac{h_1}{\beta} \right) \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} \right] \sin \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t \left. \right\}
 \end{aligned} \tag{C-23}$$

$$\dot{x}_j = e^{-\frac{\alpha}{2m} t} \left\{ v_j \cos \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t - \left[\frac{\alpha v_j + 2\beta(P_j - \frac{h_j}{\beta})}{2m \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2}} \right] \sin \sqrt{\frac{\beta}{m} - \left(\frac{\alpha}{2m}\right)^2} t \right\} \quad (C-24)$$

$$\dot{x}_j(t=t_1 + \Delta t) = e^{-\frac{C}{m} \Delta t} \left\{ v_j \cos \left(\sqrt{\frac{2K}{m} - \left(\frac{C}{m}\right)^2} \Delta t \right) - \left[\frac{Cv_j + 2K(P_j - \frac{h_j}{2K})}{m \sqrt{\frac{2K}{m} - \left(\frac{C}{m}\right)^2}} \right] \sin \left(\sqrt{\frac{2K}{m} - \left(\frac{C}{m}\right)^2} \Delta t \right) \right\} \quad (C-25)$$

P_j and V_j are position and velocity calculated at the end of the previous time interval.

CASE II

$$\alpha^2 - 4m\beta > 0$$

or

$$\frac{2K}{m} - \left(\frac{C}{m}\right)^2 < 0$$

This case corresponds to overdamping.

$$r = -\frac{\alpha}{2m} \pm \sqrt{\left(\frac{\alpha}{2m}\right)^2 - \frac{\beta}{m}}$$

$$x_h = e^{-\frac{\alpha}{2m} t} \left[A e^{\sqrt{\left(\frac{\alpha}{2m}\right)^2 - \frac{\beta}{m}} t} + B e^{-\sqrt{\left(\frac{\alpha}{2m}\right)^2 - \frac{\beta}{m}} t} \right]$$

$$x_p_j = \frac{h_j}{\beta}$$

$$x_j = e^{-\frac{\alpha}{2m} t} \left[c_{1j} e^{\sqrt{\left(\frac{\alpha}{2m}\right)^2 - \frac{\beta}{m}} t} + c_{2j} e^{-\sqrt{\left(\frac{\alpha}{2m}\right)^2 - \frac{\beta}{m}} t} \right] + \frac{h_j}{\beta} \quad (C-26)$$

$$\text{at } t = 0, x_j = P_j \quad (C-27)$$

$$\text{and } \dot{x}_j = V_j \quad (C-28)$$

Let

$$\left(\frac{\alpha}{2m}\right)^2 - \frac{\beta}{m} \equiv D$$

Then

$$\dot{x}_j = e^{-\frac{\alpha}{2m}t} \left[c_{1j} \sqrt{D} e^{\sqrt{D}t} - c_{2j} \sqrt{D} e^{-\sqrt{D}t} \right]$$

$$- \frac{\alpha}{2m} \left[c_{1j} e^{\sqrt{D}t} + c_{2j} e^{-\sqrt{D}t} \right]$$

$$\dot{x}_j = e^{-\frac{\alpha}{2m}t} \left[e^{\sqrt{D}t} \left(c_{1j} \sqrt{D} - c_{1j} \frac{\alpha}{2m} \right) + e^{-\sqrt{D}t} \left(-c_{2j} \sqrt{D} - \frac{\alpha}{2m} c_{2j} \right) \right]$$

$$\dot{x}_j = e^{-\frac{\alpha}{2m}t} \left\{ e^{\sqrt{D}t} c_{1j} \left[\sqrt{D} - \frac{\alpha}{2m} \right] + e^{-\sqrt{D}t} (-c_{2j}) \left[\sqrt{D} + \frac{\alpha}{2m} \right] \right\}$$

Application of (C-27) and (C-28) yields

$$\begin{cases} v_j = c_{1j} \left(\sqrt{D} - \frac{\alpha}{2m} \right) - c_{2j} \left(\sqrt{D} + \frac{\alpha}{2m} \right) \\ p_j = c_{1j} + c_{2j} + \frac{h_j(t=0)}{\beta} \end{cases}$$

$$\begin{cases} c_{1j} \left[\sqrt{D} - \frac{\alpha}{2m} \right] + c_{2j} \left[-\sqrt{D} - \frac{\alpha}{2m} \right] = v_j \\ c_{1j} [1] + c_{2j} [1] = p_j - \frac{h_j}{\beta} \end{cases}$$

Now apply Cramer's Rule.

$$c_{1j} = \frac{\begin{vmatrix} v_j & -\sqrt{D} - \frac{\alpha}{2m} \\ p_j - \frac{h_j}{\beta} & 1 \end{vmatrix}}{\begin{vmatrix} \sqrt{D} - \frac{\alpha}{2m} & -\sqrt{D} - \frac{\alpha}{2m} \\ 1 & 1 \end{vmatrix}}$$

$$C_{1j} = \frac{v_j + \left(p_j - \frac{h_1}{\beta}\right) \left(\sqrt{D} + \frac{\alpha}{2m}\right)}{\sqrt{D} - \frac{\alpha}{2m} + \sqrt{D} + \frac{\alpha}{2m}}$$

$$C_{1j} = \frac{v_j + \left(p_j - \frac{h_1}{\beta}\right) \left(\sqrt{D} + \frac{\alpha}{2m}\right)}{2\sqrt{D}} \quad (C-29)$$

$$C_{2j} = \frac{\begin{vmatrix} \sqrt{D} - \frac{\alpha}{2m} & v_j \\ 1 & p_j - \frac{h_1}{\beta} \end{vmatrix}}{2\sqrt{D}} = \frac{\left(\sqrt{D} - \frac{\alpha}{2m}\right) \left(p_j - \frac{h_1}{\beta}\right) - v_j}{2\sqrt{D}} \quad (C-30)$$

Substitute (C-29) and (C-30) into (C-26)

$$x_j = e^{-\frac{\alpha}{2m}t} \left[\frac{v_j + \left(p_j - \frac{h_1}{\beta}\right) \left(\sqrt{D} + \frac{\alpha}{2m}\right) e^{\sqrt{D}t}}{2\sqrt{D}} + \frac{\left(\sqrt{D} - \frac{\alpha}{2m}\right) \left(p_j - \frac{h_1}{\beta}\right) - v_j}{2\sqrt{D}} e^{-\sqrt{D}t} \right] + \frac{h_1}{\beta} \quad (C-31)$$

$$x_j(t=t_1 + \Delta t) = e^{-\frac{c}{m} \Delta t} \left[\frac{v_j + \left(p_j - \frac{h_j(t=t_1)}{2K} \right) \left(\sqrt{\left(\frac{c}{m}\right)^2 - \frac{2K}{m}} + \frac{c}{m} \right) \sqrt{\left(\frac{c}{m}\right)^2 - \frac{2K}{m}} \Delta t}{2 \sqrt{\left(\frac{c}{m}\right)^2 - \frac{2K}{m}}} e^{\sqrt{\left(\frac{c}{m}\right)^2 - \frac{2K}{m}} \Delta t} \right. \\ \left. + \frac{\sqrt{\left(\frac{c}{m}\right)^2 - \frac{2K}{m}} \left(p_j - \frac{h_j(t=t_1)}{2K} \right) - v_j}{2 \sqrt{\left(\frac{c}{m}\right)^2 - \frac{2K}{m}}} e^{-\sqrt{\left(\frac{c}{m}\right)^2 - \frac{2K}{m}} \Delta t} \right] + \frac{h_j(t=t_1)}{2K} \quad (C-32)$$

p_j and v_j are position and velocity calculated at the end of the previous time interval.

Now, calculate \dot{x}_j .

$$\dot{x}_j = e^{-\frac{\alpha t}{2m}} \left[c_{1j} \sqrt{D} e^{\sqrt{D} t} - c_{2j} \sqrt{D} e^{-\sqrt{D} t} \right] \\ - \frac{\alpha}{2m} e^{-\frac{\alpha t}{2m}} \left[c_{1j} e^{\sqrt{D} t} + c_{2j} e^{-\sqrt{D} t} \right] \\ \dot{x}_j = e^{-\frac{\alpha t}{2m}} \left[e^{\sqrt{D} t} \left(c_{1j} \sqrt{D} - \frac{\alpha}{2m} c_{1j} \right) + e^{-\sqrt{D} t} \left(-c_{2j} \sqrt{D} - \frac{\alpha}{2m} c_{2j} \right) \right] \\ \dot{x}_j = e^{-\frac{\alpha t}{2m}} \left[c_{1j} \left(\sqrt{D} - \frac{\alpha}{2m} \right) e^{\sqrt{D} t} + c_{2j} \left(-\sqrt{D} - \frac{\alpha}{2m} \right) e^{-\sqrt{D} t} \right]$$

$$\dot{x}_j(t=t_1 + \Delta t) = e^{-\frac{c}{m} t} \left\{ \left[\frac{v_j + \left(P_j - \frac{h_j(t=t_1)}{2K} \right) \left(\sqrt{\left(\frac{c}{m}\right)^2 - \frac{2K}{m}} + \frac{c}{m} \right)}{2 \sqrt{\left(\frac{c}{m}\right)^2 - \frac{2K}{m}}} \right] \right. \quad (C-33)$$

$$\times \left[\sqrt{\left(\frac{c}{m}\right)^2 - \frac{2K}{m}} - \frac{c}{m} \right] e^{\sqrt{\left(\frac{c}{m}\right)^2 - \frac{2K}{m}} \Delta t}$$

$$+ \left[\frac{\left(\sqrt{\left(\frac{c}{m}\right)^2 - \frac{2K}{m}} - \frac{c}{m} \right) \left(P_j - \frac{h_j(t=t_1)}{2K} \right) - v_j}{2 \sqrt{\left(\frac{c}{m}\right)^2 - \frac{2K}{m}}} \right] \left[-\sqrt{\left(\frac{c}{m}\right)^2 - \frac{2K}{m}} - \frac{c}{m} \right]$$

$$\times e^{-\sqrt{\left(\frac{c}{m}\right)^2 - \frac{2K}{m}} \Delta t}$$

CASE III

$$\alpha^2 = 4m\beta$$

or

$$\frac{2K}{m} = \left(\frac{c}{m}\right)^2$$

This case corresponds to critical damping.

$$r = -\frac{\alpha}{2m}$$

Repeated roots.

$$x_h = A e^{-\frac{\alpha}{2m} t} + B t e^{-\frac{\alpha}{2m} t}$$

$$x_p = \frac{h_1}{\beta}$$

$$x_j = A_j e^{-\frac{\alpha}{2m} t} + B_j t e^{-\frac{\alpha}{2m} t} + \frac{h_1}{\beta}$$

$$\dot{x}_j = -A_j \frac{\alpha}{2m} e^{-\frac{\alpha}{2m} t} + B_j \left[e^{-\frac{\alpha}{2m} t} - \frac{\alpha}{2m} t e^{-\frac{\alpha}{2m} t} \right]$$

$$\dot{x}_j = e^{-\frac{\alpha}{2m} t} \left[-A_j \frac{\alpha}{2m} + B_j - B_j \frac{\alpha}{2m} t \right]$$

$$\text{At } t = 0, x_j = P_j, \dot{x}_j = V_j$$

$$\therefore P_j = A_j + \frac{h_j}{\beta}$$

$$V_j = -A_j \frac{\alpha}{2m} + B_j$$

$$A_j = P_j - \frac{h_j}{\beta}$$

$$B_j = V_j + A_j \frac{\alpha}{2m} = V_j + \frac{\alpha}{2m} \left(P_j - \frac{h_j}{\beta} \right)$$

$$X_j(t=t_1 + \Delta t) = e^{-\frac{c}{m} \Delta t} \left[P_j - \frac{h_j(t=t_1)}{2K} + \Delta t \left(V_j + \frac{c}{m} \left(P_j - \frac{h_j(t=t_1)}{2K} \right) \right) \right] + \frac{h_j(t=t_1)}{2K} \quad (C-34)$$

$$\dot{X}_j(t=t_1 + \Delta t) = e^{-\frac{c}{m} \Delta t} \left\{ - \left(P_j - \frac{h_j(t=t_1)}{2K} \right) \frac{c}{m} + \left[V_j + \frac{c}{m} \left(P_j - \frac{h_j(t=t_1)}{2K} \right) \right] \left[1 - \frac{c}{m} \Delta t \right] \right\} \quad (C-35)$$

P_j and V_j are positions and velocities calculated at the end of the previous time interval.

The choice of which pair of equations ((C-21) (C-25) or (C-32) (C-33) or (C-34) (C-35)) should be used for calculating the motions of the masses is determined by the value of $\frac{2K}{m} - \left(\frac{c}{m}\right)^2$.

Using the latest available calculated parameter values, one evaluates h_j at each time interval. All equations for X_j ($j = 1, 2, 3, \dots$) are evaluated sequentially for one time interval before the next time interval is examined.

NOTE: The above solutions can be generalized to a system with all different masses, spring constants, and dampers if the following substitutions are made:

$$\beta = K_i + K_{i+1}, \quad \alpha = c_i + c_{i+1}, \quad m = m_i$$

Spring surging and coil clashing can also be analyzed from a continuum rather than a finite element. A derivation of the appropriate equations appears in Reference¹³. This work was done under contract to the Army Weapons Command, and a computer program is available. This approach appears to be more economical than the finite-element approach unless a transfer function can be found. The search for such a function is continuing.

A simpler treatment of inertial effects is to assume that an ideal spring is acting on a mass equal to $\frac{1}{3}$ the mass of the spring. Because the underlying assumptions are not immediately apparent, a derivation is shown below:

Assume that the velocity of any point on the spring varies linearly with spring mass between that point and the fixed end; that is,

$$V = \frac{m}{M_{\text{SPRING}}} V_{\text{end}} .$$

Thus, the velocity is zero at the fixed end ($m=0$) and is equal to V_{end} at the forced end ($m = M_{\text{SPRING}}$). Now, calculate the total kinetic energy (KE) of all mass points along the spring.

$$\begin{aligned} \text{KE} &= \frac{1}{2} \int_0^{M_{\text{SPRING}}} v^2 dm = \frac{1}{2} \int_0^{M_{\text{SPRING}}} \left(\frac{m}{M_{\text{SPRING}}} \right)^2 v_{\text{end}}^2 dm \\ &= \frac{1}{2} \left(\frac{v_{\text{end}}}{M_{\text{SPRING}}} \right)^2 \frac{1}{3} m^3 \Big|_0^{M_{\text{SPRING}}} = \frac{1}{2} \left[\frac{1}{3} M_{\text{SPRING}} \right] v_{\text{end}}^2 \end{aligned}$$

Thus, a mass equal to $\frac{1}{3} M_{\text{SPRING}}$ that is forced against a massless spring with velocity V_{end} will have the same kinetic energy as the spring with the distributed mass that is forced with the same velocity. The key assumption is linearity of velocity along the spring. As the spring is forced with greater velocity, this assumption becomes less accurate.

¹³Phillips, J.W. and Costello, G.A., Large Deflections of Impacted Helical Springs, Journal of the Acoustical Society of America, VOL 51, No. 3 (Part 2), pg 967 (1972)

APPENDIX D

EXTENSION OF DETERMINISTIC MODELS TO PROBABILISTIC REGIME

The M16A1 Rifle model described in this report is deterministic. That is, one is given certain single-valued input data, and the model provides single-valued output information. Such a model describes only one particular weapon under one particular set of firing conditions, and with one particular round of ammunition. However, input data in reality have a range of values. Not all weapon parts will have the same weights and dimensions because of certain manufacturing tolerances. Not all rounds of ammunition will produce the same pressure-time curve. Not all mounts will have the same stiffness. Not all crosswinds will have the same velocity. Not all shooters will hold the weapon the same way. Not all weapons will have the same amount of lubricant and contaminant. Not all weapons are fired at the same ambient temperature. Such a list is virtually endless; however, the problem is not as hopeless as it may seem. Most of these quantities have reasonably well-defined bounds, that is, wind velocity does not vary from plus infinity to minus infinity. A reasonable estimate of mean value and standard deviation is possible. One can then construct a reasonable probability distribution function. The more experimental data that are available, the more accurate this curve is likely to be. Similarly for the other input data, one can approximate probability distribution functions; of course, some will be more difficult than others. The next problem is the determination of how to process this information. The computer should accept this stochastic input information, operate with it, and provide stochastic output.

Two possible methods are discussed by which a model could be designed to carry out this process. The first is the Monte Carlo technique and the second is a generalization of the method of partial derivatives. An alternative to statistics is also presented. This is a perturbation technique.

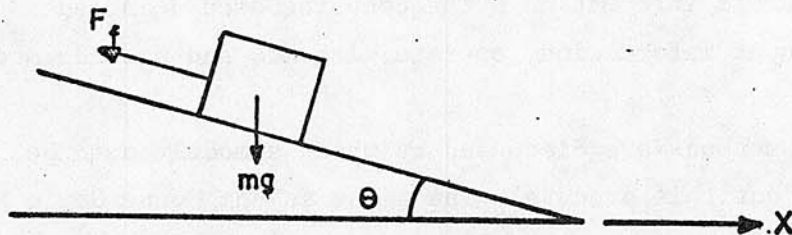
The Monte Carlo approach requires that one run the deterministic model for many input values. These values are chosen to reflect the probability distribution of the appropriate quantity. The method of partial derivatives attempts to approximate the output means and standard deviation given the input function. The perturbation technique is simply a way to examine the derivative of various quantities, with respect to the others, to determine how sensitive the first variable is to changes in the second. One can then use this knowledge of the range of values for the second variable to determine the significance of the derivative. The Monte Carlo method has a serious disadvantage in that it requires a large number of computer runs. For a complex model, this shortcoming causes the computer costs to become excessive.

The method of partial derivatives requires only one computer run and would save computer costs, but this method is more difficult to set up. The technique does not provide the probability density function; it provides a measure of the variability in output that could be expected. The perturbation technique provides even less information directly.

The three techniques will be discussed in some technical detail in the following paragraphs.

D.1 Introductory Example

First, an introductory problem will be presented to illustrate some main ideas of this discussion about probabilistic models. The problem is atypical in its lack of complexity. Let "m" be a mass sliding on an inclined plane, as shown below:



F_f is a friction force equal to $\mu mg \cos \theta$, where μ is the coefficient of sliding friction. The equation of motion for the mass is

$$\ddot{x} = mg \sin \theta - \mu mg \cos \theta, \theta \neq 0$$

$$\text{I.C. } x(0) = \dot{x}(0) = 0$$

The solution is

$$x(t) = [g \sin \theta - \mu g \cos \theta] \frac{t^2}{2}, \dot{x}(t) = [g \sin \theta - \mu g \cos \theta] t$$

or

$$x(t) = A + B \mu, \dot{x}(t) = A_1 + B_1 \mu$$

where

$$A = [g \sin \theta] \frac{t^2}{2}, \quad B = -[g \cos \theta] \frac{t^2}{2}$$

$$A_1 = [g \sin \theta] t, \quad B_1 = -[g \cos \theta] t$$

Suppose that one selects the inclined plane and the sliding mass from a stockpile of these objects where the coefficient of friction has a normal distribution. Certainly, the surfaces will have some differences in finish and lubrication, and not all the coefficients can be expected to be the same. Suppose that the distribution has a mean of .3 and a standard deviation of .05. Then, from probability theory, $x(t)$ and $\dot{x}(t)$ have a normal distribution with means of $\bar{x} = A + B \bar{\mu} = A + .3B$,

$$\bar{\dot{x}} = A_1 + B_1 \bar{\mu} = A_1 + .3B_1$$

and with standard deviations of

$$\sigma_x = [B^2 (.05)^2]^{1/2}$$

and

$$\sigma_{\dot{x}} = [B_1^2 (.05)^2]^{1/2}, \text{ respectively.}$$

At any given time, t_s , one can determine the probability that $x(t_s)$ or $\dot{x}(t_s)$ will lie within certain limits.

For example, let us find the values of $x(.5)$ that lie within two standard deviations of the mean of $x(.5)$.

Let

$$\theta = 45^\circ, \quad g = 32.2 \text{ ft/sec}^2, \quad t_s = .5 \text{ sec}$$

Then

$$A = 2.8461 \text{ ft}, \quad B = -2.8461 \text{ ft}$$

and

$$\bar{x} (.5) = 2.8461 - (.3)(2.8461) = 1.9923$$

$$\sigma_x = [(2.8461)^2 (.05)^2]^{\frac{1}{2}} = .1423$$

$$P(1.9923 - 2\sigma_x \leq x(.5) \leq 1.9923 + 2\sigma_x) = .95$$

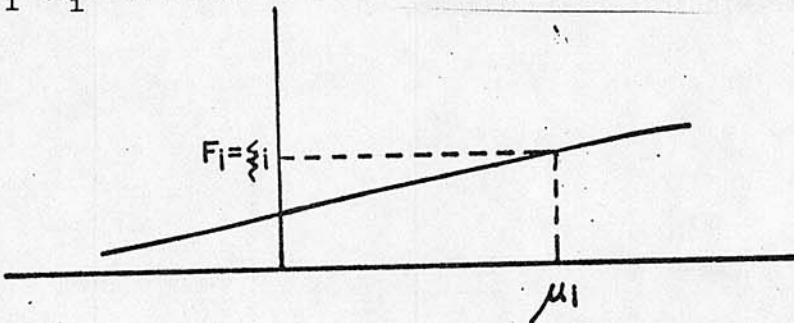
$$P(1.7077 \leq x(.5) \leq 2.2769) = .95$$

The previous results are dependent upon the fact that the differential equation can be solved analytically, and that the distribution is normal and completely determined. If μ had a log-normal distribution, it would be difficult to find the probability distribution for $x(t)$. With a complex model, analytical solutions become virtually impossible.

D.2 Monte Carlo

To attack problems associated with complex models, other techniques are required. The first to be discussed will be the Monte Carlo technique. It will be applied to the simple problem of a mass sliding on an inclined plane. First, a preliminary discussion of the technique will be made.

Let $\mu_1, \mu_2, \dots, \mu_n$ be a sequence of random variables that are constant in time. Let $F_1(\mu_1), F_2(\mu_2), \dots, F_n(\mu_n)$ be the corresponding cumulative probability distributions. These may be known in terms of elementary functions or from test data. For each random variable μ_i , random numbers ξ_i where $0 \leq \xi_i \leq 1$ are generated from a uniform distribution. The numbers ξ_i are equated to $F(\mu_i)$ for some μ_i . Then many values for μ_i are computed from $\mu_i = F_i^{-1}(\xi_i)$ as shown in the following diagram:



The differential equations are then solved many times with the use of the μ_i values. The output values are then distributed in some fashion not necessarily similar to the input values.

Recall that the inclined plane problem is described by:

$\ddot{x} = g \sin \theta - \mu g \cos \theta$, $x(0) = \dot{x}(0) = 0$, $\theta = 45^\circ$, and $g = 32.2 \text{ ft/sec}^2$ where μ is normally distributed with a mean of .3 and a standard deviation of .05.

Let $Z \equiv \frac{\mu - .3}{.05}$. Then "Z" has a standard normal distribution. Let $F(x)$ be the cumulative probability distribution function of the standard normal.

$$F(Z) = \int_{-\infty}^Z f(s) ds. \text{ Generate random } \xi_1 \text{ and compute } Z = F^{-1}(\xi_1).$$

Then $\mu = .05 Z + .3$. Table 11 contains the values obtained.

By taking a sufficiently large sample, one can find approximate probabilities that $x(t)$ at a given time will lie within certain limits. For example,

$$P(29 \leq x(2) \leq 31) \approx \frac{\# \text{trials } (29 \leq x(2) \leq 31)}{\text{total \# trials}} \approx \frac{12}{15} = .80$$

or

$$P(1.7077 \leq x(.5) \leq 2.2769) \approx \frac{\# \text{trials } (1.7077 \leq x(.5) \leq 2.2769)}{\text{total \# trials}} \approx \frac{14}{15} = .93$$

From the previously derived exact solution, $P(1.7077 \leq x(.5) \leq 2.2769) = .95$

It is possible to place confidence limits on the estimate that the probability is .93 that $x(.5)$ lies between 1.7077 and 2.2769. Such information is tabulated. From pg. 147 in Reference¹⁴, a 95% confidence interval about .93 is (.67, 1). This interval could be reduced if a larger sample were used.

An estimate of the sample size required for a $1 - \alpha$ confidence interval ($P - E$, $P + E$) for the probability "P" is

$$n = \frac{P(1-P) \left[Z_{1 - \frac{\alpha}{2}} \right]^2}{E^2} \cdot \left(Z_{1 - \frac{\alpha}{2}} - \frac{\alpha}{2} \right)$$

indicates that "Z" is a function of $1 - \frac{\alpha}{2}$.)

¹⁴Haugen, E., Probabilistic Approaches to Design, John Wiley & Sons, Inc., New York (1968)

TABLE 11 SAMPLE MONTE CARLO CALCULATIONS

	ξ	Z	μ	$x(.5)$	$x(2)$
1.	.816	.90	.345	1.864	29.82
2.	.763	.72	.336	1.889	30.22
3.	.061	-1.54	.323	1.927	30.83
4.	.988	2.28	.4125	1.672	26.75
5.	.174	-.94	.253	2.126	30.02
6.	.709	.55	.3275	1.914	30.62
7.	.889	1.22	.361	1.819	29.10
8.	.772	.75	.3375	1.885	30.16
9.	.893	1.24	.362	1.816	29.05
10.	.232	.73	.3365	1.888	30.21
11.	.091	1.33	.3665	1.803	28.85
12.	.133	1.11	.3555	1.834	29.34
13.	.197	.85	.3425	1.871	29.94
14.	.469	.10	.305	1.978	31.65
15.	.061	-1.55	.2225	2.213	35.10

For the case $(1.7077 \leq x(.5) \leq 2.2769)$, $E = .05$, $\alpha = .05$, $P = .93$
and

$$n = \frac{.93(1 - .93)(1.96)^2}{(.05)^2} = 100$$

Thus, a sample size of 100 would be needed to get a 95% confidence interval of (.88, .98).

D.3 Partial Derivatives

Now, consider the method of partial derivatives. This method provides estimates for the means and standard deviations of the output given the means and standard deviations of the input data. It does not provide a probability density function for the output. The technique is applied to algebraic equations in Haugen's book.¹⁴ Here, the technique is extended to ordinary differential equations.

$$\text{Consider the differential equation } \dot{x}(t) = F(x, a, t) \quad (\text{D-1})$$

where $x(0) = x_0$. The variable "a" is random and has a mean " \bar{a} " and a standard deviation " σ ".

¹⁴Haugen, E., Probabilistic Approaches to Design, John Wiley & Sons, Inc., New York (1968)

The solution of the equation is $x(t) = \int_0^t F(x, a, \tau) d\tau + x_0$. (D-2)

Thus, the variable $x(t)$ is also random.

Given that x_1, x_2, \dots, x_n are random variables with means $\bar{x}_1, \bar{x}_2, \dots, \bar{x}_n$ and standard deviations $S_{x_1}, S_{x_2}, \dots, S_{x_n}$, define $z = \Psi(x_1, x_2, \dots, x_n)$.

Approximations to the mean and standard deviation of "z" are:

$$\bar{z} \approx \Psi(\bar{x}_1, \bar{x}_2, \dots, \bar{x}_n) \quad (D-3)$$

$$S_z \approx \left[\sum_{j=1}^n \left(\frac{\partial z}{\partial x_j} \right)^2 S_{x_j}^2 \right]^{1/2} \quad (D-4)$$

NOTE: In the approximation of the mean, higher-order terms in a Taylor series expansion of "z" about \bar{x}_i are neglected. Some could be included for greater accuracy. The partial derivatives in the approximation to the standard deviation are evaluated at the mean values of the x_i 's. These approximations are quite good if the variances are not too large. The remaining development of the method of partial derivatives will be based on these approximations.

Now apply (D-3) to (D-2)

$$\bar{x}(t, a) = \int_0^t F(x, \bar{a}, \tau) d\tau + x_0$$

This equation means that the mean of $x(t, a)$ can be approximated by the solution of the differential equation where " \bar{a} " has been substituted for "a".

Now

$$S_{x(t, a)} = \left[\left(\frac{\partial x(t, a)}{\partial a} \right)^2 S_a^2 \right]^{1/2}$$

From (D-2)

$$\frac{\partial x(t, a)}{\partial a} \Big|_{a = \bar{a}} = \int_0^t \frac{\partial F(x, a, \tau)}{\partial a} \Big|_{a = \bar{a}} d\tau$$

$$S_a = \sigma$$

$$\therefore S_{x(t, a)} = \int_0^t \frac{\partial F(x, a, \tau)}{\partial a} \Big|_{a = \bar{a}} d\tau \sigma$$

Consider the example

$$\dot{x}(t) = a e^{-t}$$

$$x(0) = 0$$

The variable "a" has a mean " \bar{a} " and a standard deviation σ .

$$\bar{x}(t, a) \approx \int_0^t F(x, \bar{a}, \tau) d\tau + x_0 = \int_0^t \bar{a} e^{-\tau} d\tau + 0 = \bar{a} (1 - e^{-t})$$

$$\frac{\partial F(x, a, t)}{\partial a} \Big|_{a = \bar{a}} = e^{-t}$$

$$\frac{\partial x(t, a)}{\partial a} \Big|_{a = \bar{a}} = \int_0^t \frac{\partial F(x, a, \tau)}{\partial a} \Big|_{a = \bar{a}} d\tau = \int_0^t e^{-\tau} d\tau = 1 - e^{-t}$$

$$S_{x(t, a)} \approx \left| \frac{\partial x(t, a)}{\partial a} \right| \sigma = \sigma (1 - e^{-t})$$

For this example, the exact result can be obtained.

$$x(t, a) = a (1 - e^{-t})$$

$$\bar{x}(t, a) = \bar{a} (1 - e^{-t})$$

$$S_x(t, a) = \sigma (1 - e^{-t})$$

Thus, the approximate and exact results are equal.

This scheme will now be generalized to a system of first-order differential equations. Since one or more higher-order differential equations can be transformed to a system of first-order equations, this is a powerful generalization.

The system of equations is described by:

$$\dot{x}(t) = F(x, a, t)$$

$$x(0) = g(a)$$

where

$$\dot{\mathbf{x}}(t) = \begin{bmatrix} \dot{x}_1(t) \\ \dot{x}_2(t) \\ \vdots \\ \dot{x}_n(t) \end{bmatrix} \quad \mathbf{F}(x, a, t) = \begin{bmatrix} F_1(x, a, t) \\ F_2(x, a, t) \\ \vdots \\ F_n(x, a, t) \end{bmatrix}$$

$$\mathbf{a} = \begin{bmatrix} a_1 \\ a_2 \\ \vdots \\ a_m \end{bmatrix} \quad \mathbf{g}(\mathbf{a}) = \begin{bmatrix} g_1(\mathbf{a}) \\ g_2(\mathbf{a}) \\ \vdots \\ g_n(\mathbf{a}) \end{bmatrix} \quad \bar{\mathbf{a}} = \begin{bmatrix} \bar{a}_1 \\ \bar{a}_2 \\ \vdots \\ \bar{a}_m \end{bmatrix}$$

then

$$\mathbf{x}(t, \mathbf{a}) = \int_0^t \mathbf{F}(x, \mathbf{a}, \tau) d\tau + \mathbf{g}(\mathbf{a})$$

$$\bar{\mathbf{x}}(t, \mathbf{a}) \approx \int_0^t \mathbf{F}(x, \bar{\mathbf{a}}, \tau) d\tau + \mathbf{g}(\bar{\mathbf{a}})$$

$$\left[\begin{pmatrix} \left(\frac{\partial x_1(t, \mathbf{a})}{\partial a_1} \right)^2 & \dots & \left(\frac{\partial x_1(t, \mathbf{a})}{\partial a_m} \right)^2 \\ \vdots & & \vdots \\ \left(\frac{\partial x_n(t, \mathbf{a})}{\partial a_1} \right)^2 & \dots & \left(\frac{\partial x_n(t, \mathbf{a})}{\partial a_m} \right)^2 \end{pmatrix} \begin{pmatrix} \sigma_{a_1}^2 \\ \vdots \\ \sigma_{a_m}^2 \end{pmatrix} \right]^{1/2} \quad (\text{D-5})$$

The square root of the column vector is the column vector formed from the square root of each term. Partial derivatives are evaluated at $\bar{a}_1, \bar{a}_2, \dots, \bar{a}_n$.

$$\frac{\partial \mathbf{x}(t, \mathbf{a})}{\partial a_j} \Big|_{\mathbf{a}=\bar{\mathbf{a}}} = \int_0^t \frac{\partial \mathbf{F}(x, \mathbf{a}, \tau)}{\partial a_j} \Big|_{\mathbf{a}=\bar{\mathbf{a}}} d\tau + \frac{\partial \mathbf{g}(\mathbf{a})}{\partial a_j} \Big|_{\mathbf{a}=\bar{\mathbf{a}}} \quad (\text{D-6})$$

The following is a simple example for which exact results can also be obtained. Consider a free-falling mass. $\ddot{x} = -g$

$$\begin{aligned} x(0) &= a_1 \\ \dot{x}(0) &= a_2 \end{aligned}$$

Assume a_1 has a mean of zero and a standard deviation of σ_{a_1} .

Assume a_2 has a mean of zero and a standard deviation of σ_{a_2} .

Make the following transformation:

$$\begin{cases} x_1 \equiv x \\ x_2 \equiv \dot{x} \end{cases}$$

$$\begin{aligned} \therefore x_1(0) &= a_1 \\ x_2(0) &= a_2 \\ \begin{pmatrix} \dot{x}_1 \\ \dot{x}_2 \end{pmatrix} &= \begin{pmatrix} x_2 \\ -g \end{pmatrix} \end{aligned}$$

$$\begin{pmatrix} \frac{\partial x_1}{\partial a_1} \\ \frac{\partial x_2}{\partial a_1} \end{pmatrix} = \begin{pmatrix} \int_0^t \frac{\partial x_2}{\partial a_1} dt \\ - \int_0^t \frac{\partial g}{\partial a_1} dt \end{pmatrix} + \begin{pmatrix} 1 \\ 0 \end{pmatrix} = \begin{pmatrix} 1 \\ 0 \end{pmatrix}$$

$$\begin{pmatrix} \frac{\partial x_1}{\partial a_2} \\ \frac{\partial x_2}{\partial a_2} \end{pmatrix} = \begin{pmatrix} \int_0^t \frac{\partial x_2}{\partial a_2} dt \\ - \int_0^t \frac{\partial g}{\partial a_2} dt \end{pmatrix} + \begin{pmatrix} 0 \\ 1 \end{pmatrix} = \begin{pmatrix} t \\ 1 \end{pmatrix}$$

Insert these expressions into (D-5).

$$S_x(t, a) \equiv \left[\begin{pmatrix} 1 & t^2 \\ 0 & 1 \end{pmatrix} \begin{pmatrix} \sigma_{a_1}^2 \\ \sigma_{a_2}^2 \end{pmatrix} \right]^{\frac{1}{2}} = \left[\begin{pmatrix} \sigma_{a_1}^2 + \sigma_{a_2}^2 t^2 \\ \sigma_{a_2}^2 \end{pmatrix} \right]^{\frac{1}{2}}$$

Now, find the exact results for $S_x(t, a)$ and compare with the approximate expression above.

$$\ddot{x} = -g$$

$$x(0) = a_1 \quad \dot{x}(0) = a_2$$

$$x(t) = -\frac{1}{2} g t^2 + a_2 t + a_1$$

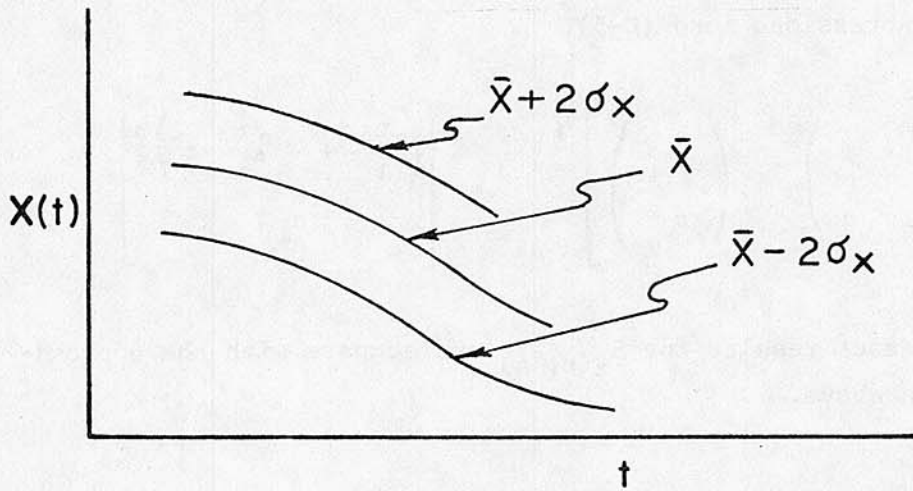
$$S_x = \left[\begin{pmatrix} \sigma_{a_2}^2 t^2 + \sigma_{a_1}^2 \\ \sigma_{a_2}^2 \end{pmatrix} \right]^{\frac{1}{2}}$$

$$\dot{x}(t) = -g t + a_2$$

$$S_{\dot{x}} = \sigma_{a_2}$$

Thus, the approximate and exact results for the standard deviations are equal. It can also be shown that the approximate and exact mean values are equal.

Using this approximate method, one could get a good idea of the nature of the output without resorting to the time-consuming Monte Carlo technique. By calculating approximate output means and standard deviations, one can "bracket" the solution by graphing the results as follows:



Using the method of partial derivatives, one can avoid transformation to a system of first-order equations and can find partial derivatives directly. For example, consider a projectile that leaves a barrel with a random muzzle velocity having a mean v_0 and a standard deviation σ_{v_0} . Assume that the angle of elevation θ is a random variable with mean $\bar{\theta}$ and standard deviation σ_θ . Assume that the only force on the projectile is that of gravity.

Then

$$\begin{aligned} \ddot{y} &= -g & y(0) &= 0, & \dot{y}(0) &= v_0 \sin \theta \\ \ddot{x} &= 0 & x(0) &= 0, & \dot{x}(0) &= v_0 \cos \theta \end{aligned}$$

Do not transform the system to first-order, but directly take partial derivatives of the equations with respect to the random variables.

Note that

$$y_\alpha \equiv \frac{\partial y}{\partial \alpha}, \quad x_\alpha \equiv \frac{\partial x}{\partial \alpha}$$

$$m \ddot{y}_{v_0} = 0$$

$$y_{v_0}(0) = 0$$

$$\dot{y}_{v_0}(0) = \sin \theta$$

$$\therefore \dot{y}_{v_0} = \alpha$$

$$y_{v_0} = \alpha t + \beta$$

$$\begin{cases} y_{v_0} = (\sin \theta) t \\ \dot{y}_{v_0} = \sin \theta \end{cases}$$

$$m \ddot{x}_{v_0} = 0$$

$$x_{v_0}(0) = 0$$

$$\dot{x}_{v_0}(0) = \cos \theta$$

$$\therefore \dot{x}_{v_0} = \alpha$$

$$x_{v_0} = \alpha t + \beta$$

$$\begin{cases} x_{v_0} = (\cos \theta) t \\ \dot{x}_{v_0} = \cos \theta \end{cases}$$

$$m \ddot{y}_\theta = 0$$

$$y_\theta(0) = 0$$

$$\dot{y}_\theta(0) = v_0 \cos \theta$$

$$\therefore \dot{y}_\theta = \alpha$$

$$y_\theta = \alpha t + \beta$$

$$\begin{cases} y_\theta = v_0 (\cos \theta) t \\ \dot{y}_\theta = v_0 \cos \theta \end{cases}$$

$$m \ddot{x}_\theta = 0$$

$$x_\theta(0) = 0$$

$$\dot{x}_\theta(0) = -v_0 \sin \theta$$

$$\therefore \dot{x}_\theta = \alpha$$

$$x_\theta = \alpha t + \beta$$

$$\begin{cases} x_\theta = -(v_0 \sin \theta) t \\ \dot{x}_\theta = -v_0 \sin \theta \end{cases}$$

Equation (D-4) is

$$S_Z \approx \left[\sum_{j=1}^n \left(\frac{\partial Z}{\partial x_j} \right)^2 S_{x_j}^2 \right]^{1/2}$$

$$\therefore S_{x(t)} \equiv [\sigma_{v_0}^2 \cos^2 \bar{\theta} + \sigma_{\theta}^2 (\bar{v}_0 \sin \bar{\theta})^2]^{1/2} t$$

$$S_{x^*}(t) \approx [\sigma_{v_0}^2 \cos^2 \bar{\theta} + \sigma_{\theta}^2 (\bar{v}_0 \sin \bar{\theta})^2]^{1/2}$$

$$S_{y(t)} \approx [\sigma_{v_0}^2 \sin^2 \bar{\theta} + \sigma_{\theta}^2 (\bar{v}_0 \cos \bar{\theta})^2]^{1/2} t$$

$$S_{y^*}(t) \approx [\sigma_{v_0}^2 \sin^2 \bar{\theta} + \sigma_{\theta}^2 (\bar{v}_0 \cos \bar{\theta})^2]^{1/2}$$

The following is an illustration of the types of conclusions that can be drawn from the results of this method.

Suppose

$$\bar{\theta} = 0$$

Then

$$S_{x(t)} \equiv \sigma_{v_0} t$$

$$S_{x^*}(t) \equiv \sigma_{v_0}$$

$$S_{y(t)} \equiv \sigma_{\theta} \bar{v}_0 t$$

$$S_{y^*}(t) \equiv \sigma_{\theta} \bar{v}_0$$

Thus, when the average elevation is zero, σ_{θ} has virtually no effect on $S_{x(t)}$ or $S_{x^*}(t)$ if σ_{θ} is small.

One can, by maximizing $S_{x(t)}$, find the average elevation at which $x(t)$ has a maximum variation. Thus, one must find $\bar{\theta}$ that maximizes

$$S_{x(t)} = [\sigma_{v_0}^2 \cos^2 \bar{\theta} + \sigma_{\theta}^2 v_0^2 \sin^2 \bar{\theta}]^{1/2} t$$

Substitute $\sin^2 \bar{\theta} = 1 - \cos^2 \bar{\theta}$

$$S_{x(t)} = [(\sigma_{v_0}^2 - \sigma_{\theta}^2 v_0^2) \cos^2 \bar{\theta} + \sigma_{\theta}^2 v_0^2]^{1/2} t$$

If

$$\theta = 0,$$

then

$S_{x(t)}$ is a maximum.

D.4 Perturbation

The following perturbation method is an alternative to the statistical approach. This method is not as powerful, but it can yield much useful information.

Consider the equation $\dot{x} = f(x, t, a)$, where "a" is a random variable, and $x(0) = x_0$.

The solution of this equation is

$$x(t) = \int_0^t f(x, t, a) dt + x_0$$

The fundamental objective of this method is to evaluate the effect on $x(t)$ caused by a small change in "a".

The change in $x(t)$ can be written $\Delta x(t) = \frac{\partial x(t)}{\partial a} \Delta a$, if only the first two terms in a Taylor series expansion of "x" about "a" are used.

From (D-1)

$$\frac{\partial x(t)}{\partial a} = \int_0^t \frac{\partial f(x, t, a)}{\partial a} dt$$

As a simple illustration, consider the following:

$$\dot{x}(t) = e^{-at}$$

$$x(0) = 0$$

$$\frac{\partial f(x, t, a)}{\partial a} = -t e^{-at}$$

$$\frac{\partial x(t)}{\partial a} = \int_0^t -t e^{-at} dt = \frac{t e^{-at}}{a} + \frac{1}{a^2} (e^{-at} - 1)$$

$$\Delta x(t) = \left[\frac{t e^{-at}}{a} + \frac{1}{a^2} (e^{-at} - 1) \right] \Delta a$$

NOTE: If the equation cannot be solved analytically, $\frac{\partial x}{\partial a}$ can still be found numerically by integrating $\frac{\partial f(x, t, a)}{\partial t}$ over time with zero initial conditions. The basic approach can be extended to systems of differential equations.

Now, consider the case of a free-falling mass and find the effects of slight variations in initial velocity and displacement on velocity and displacement at some time "t".

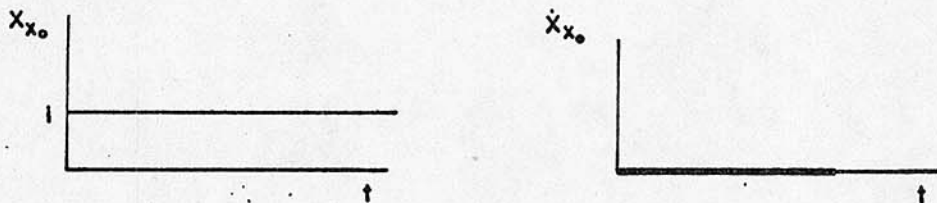
$$\begin{aligned} m\ddot{x} &= -g \\ x(0) &= x_0 \\ \dot{x}(0) &= \dot{x}_0 \end{aligned}$$

Take partial derivatives of each equation with respect to x_0 and \dot{x}_0 and solve the resulting equations.

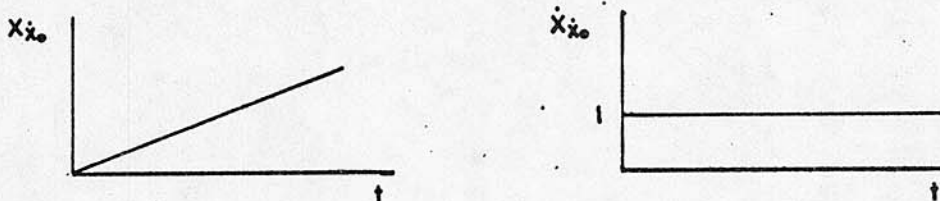
$$\begin{aligned} m\ddot{x}_{x_0} &= 0 \\ x_{x_0}(0) &= 1 \\ \dot{x}_{x_0}(0) &= 0 \\ x_{x_0} &= \alpha t + \beta \\ \begin{cases} x_{x_0}(t) = 1 \\ \dot{x}_{x_0}(t) = 0 \end{cases} \\ \Delta x(t) &= x_{x_0}(t) \Delta x_0 + \dot{x}_{x_0}(t) \Delta \dot{x}_0 \\ \therefore \Delta x(t) &= \Delta x_0 + t \Delta \dot{x}_0 \end{aligned}$$

$$\begin{aligned} m\ddot{x}_{\dot{x}_0} &= 0 \\ \dot{x}_{\dot{x}_0}(0) &= 1 \\ x_{\dot{x}_0}(0) &= 0 \\ x_{\dot{x}_0} &= \alpha t + \beta \\ \begin{cases} \dot{x}_{\dot{x}_0} = 1 \\ x_{\dot{x}_0} = 0 \end{cases} \\ \Delta \dot{x}(t) &= \dot{x}_{\dot{x}_0}(t) \Delta x_0 + x_{\dot{x}_0}(t) \Delta \dot{x}_0 \\ \Delta \dot{x}(t) &= \Delta \dot{x}_0 \end{aligned}$$

A graph can often be helpful in determining which variables should be altered to achieve a desired change in displacement or velocity.



Thus, one can see that a small change in initial displacement changes the displacement at each point in time by an equal amount. Velocity can be seen to be unaffected by a small change in displacement.



Thus, a change in initial velocity affects the displacement by an amount that increases with time. A change in initial velocity is simply added to the velocity at any other time.

APPENDIX E
"BROAD SPECTRUM OF AMMUNITION" STUDY

The "Broad Spectrum of Ammunition" Study initiated under the Weapons Laboratory (WECOM) is designed to investigate an important, but relatively unexplored area of weapon-ammunition interaction. Emphasis is on the determination of how various ammunition parameters affect weapon operation and how these parameters are distributed in normal ammunition production. The term "broad spectrum" refers to the wide range in parameter values that occurs in production. Potentially, many useful results exist from such an investigation. For example, by understanding how gas behavior is affected by nonuniformity of propellant grain shape and, in turn, how gas behavior affects mechanism performance and reliability, one could then either stiffen or relax manufacturing tolerances on uniformity of grain shape. Also, having a more complete understanding of the weapon-ammunition interface, one could better design weapons so that they are relatively insensitive to ammunition tolerances difficult to maintain. Reference ¹⁵ provides additional background.

The general approach of this study is to answer the following three basic questions: (1) What measurable weapon characteristics should be used as indicators of weapon performance? (2) How do ammunition tolerances affect these performance indicators? (3) In normal production, what are the distributions of ammunition tolerance that are important to weapon performance? Mathematical models of weapons can be highly useful in the study of the first two questions. Statistical analyses of acceptance test data on breech and port pressures have been performed in support of the third question. See Reference ¹⁶.

¹⁵"Ammunition Selection for Verification Testing of the 5.56mm, XM207, Belt-Fed Machine Gun", Technical Report 70-103, Rock Island, Illinois (Aug 1969)

¹⁶"Investigation of the Interaction of Weapon-Ammunition Subsystems", Technical Report SWERR-TR-72-30, AD 742723, Rock Island, Illinois (May 1972)

This task is very difficult, but it is also important. The problem it addresses cannot be solved in a straight-forward manner. Although a long-term effort is required, interim results should prove highly useful. This work interfaces with the probabilistic modeling described in Appendix D.

APPENDIX F
METHODS OF TREATING IMPACT

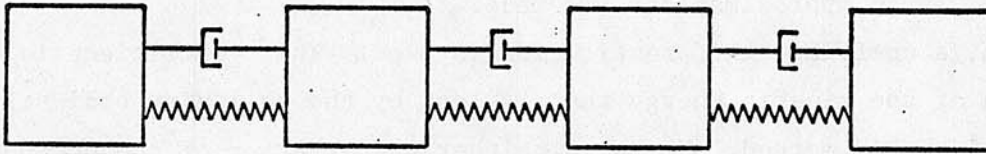
The impact between moving bodies in a weapon mechanism is very difficult to model accurately. Knowledge in this field is important for stress, wear, and fatigue analysis. In reality, time-dependent elastoplastic effects are involved. The nature of the impact depends on the motions of the masses just prior to impact, the mechanical properties of the material, the area of contact, the angle of contact, the boundary conditions on the colliding bodies, the physical dimensions of the masses, etc. However, in the interest of reasonable computation time for an already complex weapon model and in view of the lack of current theory to handle accurately impact situations, large approximations are made.

Often, a coefficient of restitution is used. This coefficient is an indication of the kinetic energy that is lost by the colliding bodies. However, with this method, one knows neither the duration of contact nor the forces involved. A better approximation would be to assume that, at the time of impact, a stiff spring and dashpot hypothetically appear between the two colliding masses. Coulomb, rather than viscous damping, could be assumed, but the question of which should be used is somewhat academic since the collision process is far too complex to be accurately modeled by either one.

Two major difficulties exist in the spring approach. The first difficulty is that it is difficult to properly choose the most appropriate spring and damping values. To some extent, the spring constant can be based on Young's modulus if one assumes that the constant is approximately AE/L . However, then a question arises as to the choice of length L . Possibly, one could let L vary and be equal to the instantaneous distance that the stress wave has traveled in the body. Another way to arrive at spring and damping constants is to correlate them with a coefficient of restitution. These coefficients are easy to measure and are often tabulated. Mr. Robert Coberly, Research Directorate, Weapons Laboratory, WECOM, has derived a relationship that will permit one to choose spring and dashpot values if the coefficient of restitution is known. The spring will then account for the force history of impact; this feat cannot be accomplished with the use of a coefficient of restitution without the spring.

The second difficulty with the spring concept is that, when a digital computer is used to solve the equations, an extremely small time increment is needed when a stiff spring is present. Otherwise, the momenta of the colliding bodies will, in the space of a single time-increment, carry these bodies so deeply into each other that the displacements represent extremely high force levels when multiplied by a large spring constant.

Another method of treating impact is to consider that the colliding bodies are hypothetically broken into a series of point masses, springs, and dashpots as follows:



This mechanical system is studied in Appendix C. However, the analysis is still too time-consuming on a computer to be practical in a large weapon model. What is needed is a transfer function that will give a direct relation between input displacement and output force. Such a model could account for another facet of impact not considered by the previously described methods. That facet is the propagation and reflection of stress waves. Parts of a colliding body, not close to the impact point, do not receive information that contact has been made until a finite length of time has elapsed. How the stress wave is reflected depends on the boundary conditions. This approach by which the body is divided into a number of segments is called a lumped-mass or finite-element approach. Simplification of this approach may be possible by replacement of the many masses and springs with a single mass-spring system where the spring constant is a function of time. This constant would be $K = \ddot{M}X/X$, where the position X of the single mass would be equal to the position history of the first mass in the large finite-element model. This ratio has been found to remain constant even if the initial conditions are changed. Thus, for a particular problem, one might establish $K(t)$ and expect it to hold regardless

of the impact velocities.

A continuum approach might also be considered. This can also be used to account for the propagation and reflection of stress waves.

An example of how it might be used follows:

Navier's equation is:

$$G \nabla^2 U_i + (\lambda + G) \frac{\partial \Delta}{\partial x_i} = \rho \ddot{U}_i$$

Assume

$$\omega = \nu = 0$$

$$U = U(x, t)$$

Substitute into Navier's equation to obtain

$$\frac{\partial^2 U}{\partial x^2} = \frac{\rho}{\lambda + 2G} \frac{\partial^2 U}{\partial t^2}$$

$$U_{tt} = \frac{\lambda + 2G}{\rho} U_{xx}$$

$$U_{tt} = c^2 U_{xx}$$

$$c = \sqrt{\frac{\lambda + 2G}{\rho}}$$

$$U_{tt} = c^2 U_{xx}$$

$$U(0, t) = f(t)$$

$$U(L, t) = 0$$

$$U(x, 0) = 0$$

$$U_t(x, 0) = h(x)$$

Let

$$U = v + g(x) f(t)$$

$$v(0, t) + g(0) f(t) = f(t)$$

$$v(L, t) + g(L) f(t) = 0$$

Require

$$v(0, t) = v(L, t) = 0$$

$$g(0) = 1$$

$$g(L) = 0$$

Δ = dilatation

λ = Lamé's constant

G = Shear modulus

$$v_{tt} + g(x) f''(t) = c^2 v_{xx} + g'(x) f(t)$$

$$v_{tt} - c^2 v_{xx} = g'(x) f(t) - g(x) f''(t)$$

Let

$$g(x) = 1 - \frac{x}{L}$$

$$v_{tt} - c^2 v_{xx} = \left(\frac{x}{L} - 1\right) f''(t)$$

$$v(0, t) = 0$$

$$v(L, t) = 0$$

$$v(x, 0) = -\left(1 - \frac{x}{L}\right) f(0)$$

$$= 0,$$

Since $f(0) = 0$

$$v_t(x, 0) = h(x) - g(x) f'(0)$$

Let

$$v = \sum \omega_n(t) \sin \frac{n\pi x}{L}$$

$$\text{I.C.} \Rightarrow \sum \omega_n(0) \sin \frac{n\pi x}{L} = 0$$

$$\therefore \omega_n(0) = 0$$

Write equation as

$$v_{tt} - c^2 v_{xx} = \sum Q_n(t) \sin \frac{n\pi x}{L},$$

where

$$Q_n(t) = \frac{2}{L} \int_0^L \left(\frac{x}{L} - 1\right) f''(t) \sin \frac{n\pi x}{L} dx.$$

By substitution,

$$\sum \omega_n'' \sin \frac{n\pi x}{L} + \sum c^2 \frac{n^2 \pi^2}{L^2} \omega_n \sin \frac{n\pi x}{L} = \sum Q_n \sin \frac{n\pi x}{L}$$

$$\omega_n''(t) + c^2 \frac{n^2 \pi^2}{L^2} \omega_n(t) = Q_n(t)$$

$$U_t(x, 0) = h(x)$$

$$v_t(x, 0) + g(x) f'(0) = h(x)$$

$$v_t(x, 0) = h(x) - f'(0) \left(1 - \frac{x}{L}\right)$$

$$v_t(x, 0) = \sum \beta_n \sin \frac{n\pi x}{L}$$

where

$$\beta_n = \frac{2}{L} \int_0^L \left[h(x) - f'(0) \left(1 - \frac{x}{L}\right) \sin \frac{n\pi x}{L} \right] dx$$

$$v_t(x, 0) = \sum \omega'_n(0) \sin \frac{n\pi x}{L}$$

$$\therefore \omega'_n(0) = \beta_n$$

$$\omega''_n(t) + \frac{c^2 n^2 \pi^2}{L^2} \omega_n(t) = G_n(t)$$

$$\omega_n(0) = 0$$

$$\omega'_n(0) = \beta_n$$

One can solve for ω_n .

Then,

$$v = \sum \omega_n(t) \sin \frac{n\pi x}{L}$$

$$U = v + \left(1 - \frac{x}{L}\right) f(t)$$

$$U = \sum \omega_n(t) \sin \frac{n\pi x}{L} + \left(1 - \frac{x}{L}\right) f(t)$$

Thus,

for a given input displacement $f(t)$ at $x=0$, one knows $U(x, t)$

Thus,

$$\text{one can find } \epsilon = \frac{\partial U}{\partial x}.$$

The generalized Hook's Law reduces to

$$\sigma = 2 G \epsilon + \lambda \epsilon = (2 G + \lambda) \epsilon$$

$$\sigma = (2 G + \lambda) \frac{\partial U}{\partial x}$$

What is needed is

$$-\sigma A = K U + c \dot{U} \doteq K f(t), \text{ neglecting damping}$$

$$K = \frac{-\sigma A}{f(t)}$$

$$K = \frac{-(2 G + \lambda) \frac{\partial U}{\partial x} A}{f(t)}$$

$$K = -\frac{2 G + \lambda}{f(t)} \left[\frac{\pi}{L} \sum_{n=1}^{\infty} n \omega_n(t) - \frac{1}{L} f(t) \right] A \quad (X=L)$$

where

$$\omega_n''(t) + \frac{c^2 n^2 \pi^2}{L^2} \omega_n(t) = Q_n(t)$$

$$\omega_n(0) = 0$$

$$\omega_n'(0) = \beta_n$$

$$Q_n(t) = \frac{2}{L} \int_0^L \left(\frac{x}{L} - 1 \right) f''(t) \sin \frac{n\pi x}{L} dx$$

$$\beta_n = \frac{2}{L} \int_0^L (h(x) - f'(0) (1 - \frac{x}{L})) \sin \frac{n\pi x}{L} dx$$

λ = Lamé's constant

G = Shear modulus

$f(t)$ = displacement history at $x=0$

$h(x) = U_t(x, 0)$ = rate of change of U along body at $t=0$.

Probably $U_t(0, 0) = f'(0)$

$$U_t(x > 0, 0) = 0$$

Let

$$f(t) = a t$$

$$Q_n(t) = \frac{2}{L} \int_0^L \left(\frac{x}{L} - 1 \right) (0) \sin \left(\frac{n\pi x}{L} \right) dx = 0$$

$$h(x) = \begin{cases} f'(0) & \text{at } x=0 \\ 0 & \text{at } x > 0 \end{cases}$$

$$\beta_n = \frac{2}{L} \int_0^L \left(-f'(0) \left(1 - \frac{x}{L} \right) \sin \frac{n\pi x}{L} \right) dx$$

$$\beta_n = -\frac{2a}{L} \int_0^L \left(1 - \frac{x}{L} \right) \sin \frac{n\pi x}{L} dx$$

$$= -\frac{2a}{L} \left[\int_0^L \sin \frac{n\pi x}{L} dx - \int_0^L \frac{x}{L} \sin \frac{n\pi x}{L} dx \right]$$

$$= -\frac{2a}{L} \left[-\frac{L}{n\pi} \cos \frac{n\pi x}{L} \Big|_0^L - \frac{1}{L} \left(\frac{L^2}{n^2\pi^2} \sin \frac{n\pi x}{L} - \frac{L}{n\pi} x \cos \frac{n\pi x}{L} \right) \Big|_0^L \right]$$

$$= -\frac{2a}{L} \left[-\frac{L}{n\pi} (-1)^n + \frac{L}{n\pi} - \frac{1}{L} \left(-\frac{L^2}{n\pi} (-1)^n \right) \right]$$

$$= -\frac{2a}{L} \left[\frac{L}{n\pi} \right] = -\frac{2a}{n\pi}$$

$$\beta_n = \frac{-2a}{n\pi}$$

$$\omega_n'' + \gamma^2 \omega_n = 0$$

$$\omega_n(0) = 0$$

$$\omega_n'(0) = -\frac{2a}{n\pi}$$

$$\omega_n = A_n \cos \gamma_n t + B_n \sin \gamma_n t$$

$$\omega_n(0) = 0 \Rightarrow A_n = 0$$

$$\omega_n' = B_n \gamma \cos \gamma_n t$$

$$\omega_n'(0) = B_n \gamma_n = -\frac{2a}{n\pi}$$

$$B_n = -\frac{2a}{\gamma_n n\pi}$$

$$B_n = -\frac{2a}{n\pi} \frac{L}{cn\pi} = -\frac{2aL}{cn^2\pi^2}$$

$$\omega_n = -\frac{2aL}{cn^2\pi^2} \sin \frac{cn\pi}{L} t$$

$$\therefore K = -A \frac{2G + \lambda}{f(t)} \left[\sum_{n=1}^{\infty} \left(-\frac{2a}{cn\pi} \sin \frac{cn\pi}{L} t \right) - \frac{f(t)}{L} \right]$$

$$K = -A \frac{2G + \lambda}{f(t)} \left[\sum_{n=1}^{\infty} \left(-\frac{2a}{cn} \sin \frac{cn\pi}{L} t \right) - \frac{f(t)}{L} \right]$$

$$f(t) = at$$

$$K = A(2G + \lambda) \left[+\frac{1}{L} + \frac{1}{t} \sum_{n=1}^{\infty} \frac{2}{cn} \sin \frac{cn\pi}{L} t \right]$$

$$K = +A(2G + \lambda) \left[\frac{1}{L} + \frac{1}{t} \sum_{n=1}^{\infty} \frac{2}{cn} \sin \frac{cn\pi}{L} t \right]$$

$$K = +AE \left[\frac{1}{L} + \frac{1}{t} \sum_{n=1}^{\infty} \frac{2}{cn} \sin \frac{cn\pi}{L} t \right]$$

Spring Constant

This K can be shown to be

$$c = \sqrt{\frac{E}{\rho}}$$

$$K = AE \left[\frac{1}{L} + \frac{\pi}{ct} \left(1 - \frac{c}{L} t \right) \right], \quad 0 < t < \frac{2L}{c}, \text{ periodically repeated.}$$

APPENDIX G
COUPLING OF COMPLEX MODELS

Many occasions arise when it would be desirable to couple complex mathematical models. Often a very large effort is needed to successfully combine the computer programs. Sometimes it is better "to start from scratch" than to attempt to make the programs compatible that have been written by two different people. This problem arises in weapon dynamic studies where an important consideration is the interaction between a mount-vehicle system and a weapon system. For example, if someone has a model of a certain weapon mechanism, much information could be gained by coupling it with someone else's model of a certain helicopter. The basic problem is an extremely important one and has applications far beyond weapons. However, the problem is also very difficult. The coupled system behaves as a completely new system and often bears little relation to the independent behavior of either system. A review of the literature reveals very little information concerning this problem.

One approach is to simplify one or both of the complex models. In effect, one models the model. It is possible to simplify the model to the point where it is a single analytic expression relating the force exerted by the mount to the position, velocity, and acceleration of the weapon. (See Reference³) However, considerable deterioration in accuracy can easily occur. Another approach is outlined below. It is much more successful in its ability to preserve the true natures of the interacting systems. However, it is more difficult to apply; but for many problems, the application of the method is simpler than simultaneous solution of all equations in both models.

³Ehle, P.E., "Mathematical Model of the Stoner 5.56mm Medium Machine Gun, XM207," WECOM Technical Report 70-114, AD 862081L, Research and Engineering Directorate, Rock Island, Ill. (Oct 1969)

Let \vec{x}_v locate the position of the point on the vehicle that is connected to the gun. Let \vec{x}_G locate the position of the point on the gun that is connected to the vehicle. Assume that some unknown interaction force exists at this common point that causes the vehicle and gun to move together there. Assume that this force can be represented by a Fourier sine or cosine series in $(0, t_0)$.

That is,

$$\vec{F}(t) = \sum_{n=1}^K \vec{A}_n \sin \frac{n\pi t}{t_0}$$

where

$$\vec{A}_n = \frac{2}{t_0} \int_0^{t_0} \vec{F}(t) \sin \frac{n\pi t}{t_0} dt$$

and t_0 is the maximum time for which the solution is desired. The assumption that the masses move together at the coupling point is equivalent to the constraint equation $\vec{x}_v - \vec{x}_G = \vec{\alpha}$, where $\vec{\alpha}$ accounts for any difference in the origins of the two coordinate systems. The procedure is as follows:

1. Make an initial guess for the \vec{A}_n values. Denote these by $\vec{A}_{n(0)}$.
2. Run the vehicle and gun programs independently with a forcing function equal to the Fourier series established with $\vec{A}_{n(0)}$ values. The result will be $\vec{x}_{v(0)}(t)$ and $\vec{x}_{G(0)}(t)$. However, these functions will almost certainly not satisfy the constraint equation.
3. Expand \vec{x}_v and \vec{x}_G in a Taylor series about $\vec{A}_{n(0)}$.

$$\vec{x}_v = \vec{x}_{v(0)} + \sum_{n=1}^K \left[\frac{\partial \vec{x}_v(0)}{\partial \vec{A}_n(0)} \right] \Delta \vec{A}_n + \sum_{n=1}^K \left[\frac{\partial^2 \vec{x}_v(0)}{\partial \vec{A}_n^2} \right] (\Delta \vec{A}_n)^2 + \dots$$

$$\vec{x}_G = \vec{x}_{G(0)} + \sum_{n=1}^K \left[\frac{\partial \vec{x}_G(0)}{\partial \vec{A}_n(0)} \right] \Delta \vec{A}_n + \sum_{n=1}^K \left[\frac{\partial^2 \vec{x}_G(0)}{\partial \vec{A}_n^2} \right] (\Delta \vec{A}_n)^2 + \dots$$

4. Substitute the results of (3.) into the constraint equation. Neglect second and higher order terms.

$$\vec{x}_v(t) - \vec{x}_G(t) - \vec{\alpha} = \sum_{n=1}^K \left[\frac{\partial \vec{x}_G(0)}{\partial \vec{A}_n(0)} - \frac{\partial \vec{x}_v(0)}{\partial \vec{A}_n(0)} \right] \Delta \vec{A}_n$$

5. From the gun model determine $\frac{\partial \vec{x}_G(0)}{\partial \vec{A}_n(0)}$; and from the vehicle model determine $\frac{\partial \vec{x}_v(0)}{\partial \vec{A}_n(0)}$. If the governing equations are

linear, the partial derivatives can be found directly. Otherwise, they can be estimated by the running of each model and the noting of changes in $\vec{x}_G(0)$ and $\vec{x}_v(0)$ as the \vec{A}_n are varied.

6. Write the equation in (4.) for each of k points in time scattered throughout $(0, t_0)$. One knows the values of $\vec{x}_v(0)$, $\vec{x}_G(0)$, $\vec{\alpha}$, $\frac{\partial \vec{x}_G(0)}{\partial \vec{A}_n(0)}$ and $\frac{\partial \vec{x}_v(0)}{\partial \vec{A}_n(0)}$. Thus, one has k algebraic equations and k unknowns $\Delta \vec{A}_n$.

7. Solve for $\Delta \vec{A}_n$ and write $\vec{A}_{n(1)} = \vec{A}_{n(0)} + \Delta \vec{A}_n$.

8. Repeat the above process, and use $\vec{A}_{n(1)}$ as the initial guess in the Fourier series.

9. Repeat the entire process until $\vec{x}_v(i) - \vec{x}_G(i)$ is as close as possible or as close as desired to $\vec{\alpha}$.

There are two basic problems that may be encountered when one applies this method. First, for a very irregular interaction force over a long time interval, a large k value may be needed. Also, for some problems, a large number of iterations may be necessary.

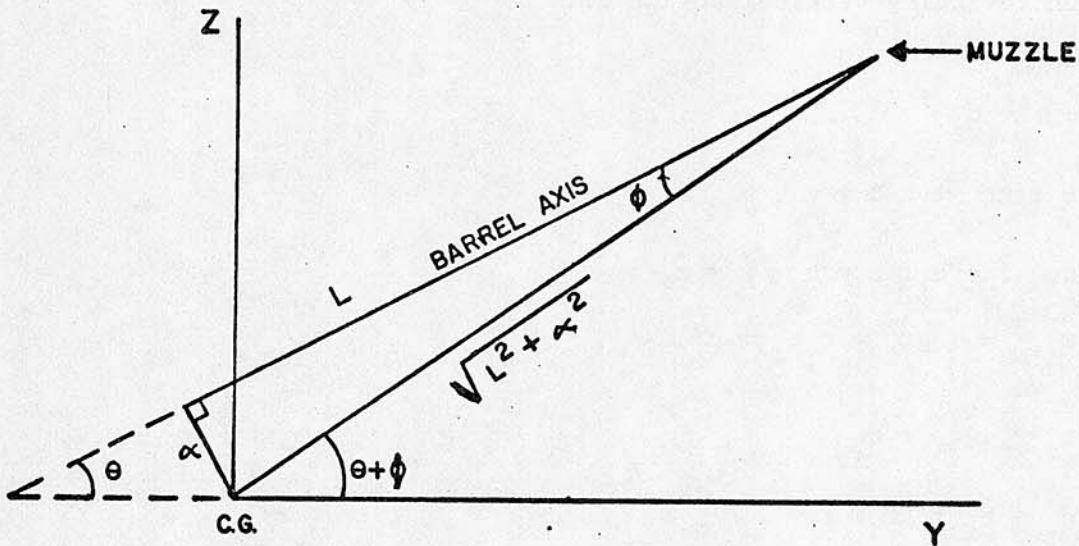
The method was applied to simple spring mass systems with nonlinear spring constants. It was also applied to a complex weapon model (XM207)

in conjunction with a simple mount model. In all cases, convergence to the known interaction force was very rapid. Four iterations gave very good results. The method will not work equally well for all problems.

The approach described represents a "first cut" at the problem. Much work remains to satisfactorily solve it.

APPENDIX H
SIMPLIFIED EXTERIOR BALLISTICS ANALYSIS

A simplified exterior ballistics analysis was made for the determination of accuracy trends as various weapon parameters were varied. A sophisticated analysis is considered unnecessary for this purpose. The projectile is given only two degrees of freedom, and the drag force is assumed to be zero. Gravity is the only force that acts on the projectile. The axis of the barrel does not pass through the pivot point, which is assumed to be the center of gravity.



NOTE: α = Minimum distance from pivot point to bore axis

Figure 45 Geometry for Exterior Ballistics Analysis

Coordinates of Muzzle:

$$Z = \sqrt{L^2 + \alpha^2} \sin (\theta + \phi)$$

$$Y = \sqrt{L^2 + \alpha^2} \cos (\theta + \phi)$$

Components of Muzzle Velocity:

$$V_y = V \cos \theta \quad V = \text{muzzle velocity}$$

$$V_z = V \sin \theta$$

Total velocity components of projectile at muzzle:

$$\dot{y} = \dot{Y} + V \cos \theta = -\dot{\theta} \sqrt{L^2 + \alpha^2} \sin(\theta + \phi) + V \cos \theta$$

$$\dot{z} = \dot{Z} + V \sin \theta = \dot{\theta} \sqrt{L^2 + \alpha^2} \cos(\theta + \phi) + V \sin \theta$$

The equation governing vertical motion is:

$$\ddot{z} = -g$$

$$\dot{z} = -gt + c_1$$

$$z = -\frac{1}{2}gt^2 + c_1 t + c_2$$

$$\text{at } t = t_i: z = z_i \quad \dot{z} = \dot{z}_i$$

$$\therefore z_i = -\frac{1}{2}gt_i^2 + c_1 t_i + c_2$$

$$\dot{z}_i = -gt_i + c_1$$

$$c_1 = \dot{z}_i + gt_i$$

$$c_2 = z_i + \frac{1}{2}gt_i^2 - t_i(\dot{z}_i + gt_i)$$

$$\therefore z = -\frac{1}{2}gt^2 + [\dot{z}_i + gt_i]t + [z_i + \frac{1}{2}gt_i^2 - t_i(\dot{z}_i + gt_i)] \quad (\text{H-1})$$

The equation governing horizontal motion is:

$$\ddot{y} = 0$$

$$y = k_1 t + k_2$$

$$\text{at } t = t_i: y = y_i \quad \dot{y} = \dot{y}_i$$

$$\therefore y_i = k_1 t_i + k_2$$

$$\dot{y}_i = k_1$$

$$k_2 = y_i - \dot{y}_i t_i \quad \text{H-2}$$

$$y = \dot{y}_i t + [y_i - \dot{y}_i t_i] \quad (H-2)$$

Now determine t when y = R = range to target

$$R = \dot{y}_i t_R + [y_i - \dot{y}_i t_i]$$

$$t_R = \frac{1}{\dot{y}_i} [R - y_i + \dot{y}_i t_i] \quad (H-3)$$

Next determine z when t = t_R by substitution of (H-3) into (H-1)

$$z_R = -\frac{1}{2} g \frac{1}{\dot{y}_i} [R - y_i + \dot{y}_i t_i]^2 + [\dot{z}_i + y t_i] \frac{1}{\dot{y}_i} [R - y_i + \dot{y}_i t_i]$$

$$+ [z_i + \frac{1}{2} g t_i - t_i (\dot{z}_i + g t_i)] \quad (H-4)$$

Assume initial time t_i = 0.

Then

$$z_R = -\frac{1}{2} g \frac{1}{\dot{y}_i^2} [R - y_i]^2 + \dot{z}_i \frac{1}{\dot{y}_i} [R - y_i] + z_i \quad (H-5)$$

where

$$z_i = \sqrt{L^2 + \alpha^2} \sin (\theta + \phi)$$

$$\dot{z}_i = \dot{\theta} \sqrt{L^2 + \alpha^2} \cos (\theta + \phi) + V \sin \theta$$

$$y_i = \sqrt{L^2 + \alpha^2} \cos (\theta + \phi)$$

$$\dot{y}_i = -\dot{\theta} \sqrt{L^2 + \alpha^2} \sin (\theta + \phi) + V \cos \theta$$

$$\phi = \sin^{-1} \frac{\alpha}{L}$$

z_R is the vertical coordinate of the impact point at range R.

Nominal Values are:

$$\alpha = .065 \text{ ft}$$

$$L = 1.645 \text{ ft}$$

$$R = 450 \text{ ft}$$

$$V = 3180 \text{ ft/sec}$$

$$\theta = 0$$

$\dot{\theta}$ is obtained from the analysis of rifle rotation.

APPENDIX I
GAS DYNAMICS

Accurate representations of gas forces are of great importance to mathematical models of automatic gas-operated weapons. At the present time, experimental pressure-time curves provide probably the most accurate simulation of these forces. However, to retain this accuracy, one must make new measurements each time changes are made in weapon parameters. Also, because of round-to-round variations, the use of any single curve is not realistic. Theoretical descriptions would allow one to account for the influence on gas force caused by changes in weapon parameters, but much work remains before really accurate predictions can be made.

In the M16A1 model, experimental pressure curves are used. However, there are many uncertainties associated with these measurements. Pressure gauges made by different manufacturers tend to disagree with one another. Also, it is very difficult to get measurements at exact points where they are needed. In addition, the presence of the gauge tends to perturb the normal fluid flow. Nevertheless, experimental measurements are probably more reliable than theoretical predictions.

There exist many computer programs that treat interior ballistics. Very few consider propellants with deterrent coatings. An unpublished one that does was constructed by T. Trafton at BRL. All of these analyses introduce gross approximations, but for many purposes, the results are satisfactory. One of the most successful analyses of flow in the M16A1 gas tube is described in Reference⁹. This analysis is fairly detailed and complex, and it provides an excellent tool for individual study of the gas tube. However, in the modeling of the dynamics of an entire weapon, a delicate balance must be struck between complexity and the limitations of computer storage and running time. This problem is compounded if the Monte Carlo technique, which requires

⁹Spurk, J.H., "The Gas Flow in Gas-Operated Weapons," Ballistic Research Laboratories Report No. 1475, Aberdeen Proving Ground, Md. (Feb 1970)

many computer runs, is used. What is needed in this instance is a relatively simple model that can accept wide modifications in flow geometry and thermal properties and provide approximate answers. With such a gas flow analysis included in the weapon model, one could, for example, look at trends in unlocking force as changes are made in the location of the gas port, the thermal properties of the tube, or changes in the tube geometry. Such an analysis has been developed in theory in Reference⁷. However, the present computer program exhibits an instability for high rates of change of pressure typical of a weapon, and this problem is currently under investigation.

⁷ Ehle, P.E. and Rahe, A.E., "Development of a Finite Element Approach for Approximate Analysis of Unsteady Compressible Fluid Flow," WECOM Technical Report SWERR-TR-72-36, AD 746234, Rock Island, Ill (Jun 72)

APPENDIX J
CONVERSION OF PHYSICAL UNITS

TABLE 12 CONVERSION OF PHYSICAL UNITS

Area: $1 \text{ m}^2 = 10.764 \text{ ft}^2 = 1,550 \text{ in}^2$
 Force: $1 \text{ newton} = 1 \text{ kg-m/sec}^2 = .22482 \text{ lb}_{\text{force}}$
 Length: $1 \text{ m} = 3.281 \text{ ft} = 39.37 \text{ in}$
 Mass: $1 \text{ kg} = 2.205 \text{ lb}_{\text{mass}}$
 Pressure: $1 \text{ newton/m}^2 = 1 \text{ kg/(m-sec}^2) = .14511 \times 10^{-3} \text{ psi}$
 Velocity: $1 \text{ m/sec} = 3.281 \text{ ft/sec} = 39.37 \text{ in/sec}$
 Cyclic rate:

<u>Time for One Shot (milliseconds)</u>	<u>Shots Per Minute</u>
60	1,000
61	984
62	968
63	952
64	938
65	923
66	909
67	896
68	882
69	870
70	857
71	845
72	833
73	822
74	811
75	800
76	789
77	779
78	769
79	759
80	750
81	741
82	732
83	723
84	714
85	706

APPENDIX K

COMPUTER PROGRAM FOR M16A1 RIFLE AND SAMPLE OUTPUT

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0001 IMPLICIT REAL*8(A-H,O-7)
0002 EXTERNAL FCT,CUIP
0003 REAL*4, I,ITMIN,ITMAX,ITX,ITY
0004 COMMON/MAS/NMASS
0005 COMMON/LAIN/ITMIN,ITMAX,ITX,ITY
0006 COMMON/RUNKA/PRMT(4),X(21),XD(21),XDD(21)
0007 COMMON/MODES/IMODE,IFG(30),IPT, IHAM, IUNLO, ISEP, IHBC, INIT, IRND
1,NPUON,ISR,IFRE4,IRX,IPICK,I4DG
COMMON/ASTHT/PBRXC ,PBRXY(50),TFIRF,CCL,XLAMCC,XMUCC,ACC,ROREA,
1 XKHYP,XCHYP,CF4,XMUA,XHR,XLO,PBRL,PCAL,CARTE(50),CARTX,PCAR
COMMON/ASIT/XPHC,XRRHC,XCHRC,DERIV,XMHUL,XMARC ,PUC,AFEHOL,
1 TDEL,XIRAG,TPORT,XCBCMG,XMURCG,EJCAR,ELEV ,XKLRC,XCLRC
COMMON/NUM/NBRCH,IFLAG,ICAV,IFLG,IFIRE
COMMON/ASTHFE/PCAVT ,PCAVY(50),FBRDY(50),PRORR,FRORX
COMMON/ER/XKBUFF,XCBUFF,XHBC,XMBV(3),XIM,RH,ALPHA,TSPRIN,
1 RH0,ALH0,RH1,ALH1,HK,HC,
1 DCOCK ,XKMCUN,XCMOUN,XUHBC,XDHA,EXTC,EXTF,FDIS,FSIC
1,R18,R2R,XMUH,FCONX
COMMON/S/X9,XC9,XK5,XC5,DETA,ALPI,THETO,XKHG,XCHG,XMHAM,XMG,
1 XHRUFF,CON1,XPUC,XPUC1,RO,CON2,XMUFF,XCUFF
COMMON/BUFF/XBFORW,XCFORW,XBREAR,XCREAR,SLACK,XUBUF,XMWT,WTI
COMMON/GRAP/TGR1,TGR2
FCONX=-1.D-10
DO 1 I=1,21
X(I)=0.000
XD(I)=0.000
1 XD(I)=0.000
C I4DG USE ONE IF THE RUFFER MASSES ARE TREATED SEPERATELY AND 0
C IF USING A LUMPED MASS MODEL
DO 2000 I=1,30
2000 IFG(I)=0
C IRND
C IMODE IRND DENOTES WHICH ROUND IS BEING FIRED
C IF IMODE IS ONE THEN PRINT FORCES
C IF IMODE IS ZERO THEN DON'T PRINT FORCES
C ISEP IS ZERO WHEN WEAPON FIRES
C ISEP IS ONE WHEN CAVITY FORCE GOES ON
C IUNLO IF 1 THEN PARREL AND THE ROLT CAN SEPERATE
C IF ZERO THEN THEY REMAIN LOCKED TOGETHOR
C IHAM ONCE LOCKING OF THE WEAPON HAS OCCURRED, THE
C HAMMER BEGINS TO ROTATE FORWARD. IT STRIKES THE FIRING PIN,AND
C IT IS ASSUMED ALL ITS ENERGY IS ABSORBED IN FIRING THE WEAPON,
C AND WHAT EVER GETS TRANSFERRED TO THE MAIN GUN DUE TO THE SPRING FORCE.
C WHEN IT JUST BEGINS TO REVERSE DIRECTION, THEN ITS ROTATIONAL VELOCITY
C AND THE DERIVATIVE OF THE ROTATIONAL VELOCITY ARE SET TO ZERO.
C (IHAM TAKES ON THE VALUE ZERO AT THIS POINT.)
C NOTE! THERE IS NO ROTATIONAL MOTION UNTIL THE BOLT CARRIER BEGINS TO
C REVERSE. (IHAM IS RESET TO ONE WHEN THE HAMMER IS COCKED AGAIN.)
C IFLAG TAKES THE VALUE ONE WHEN XMG - BLTC IS LESS THAN FDIS
C AND TAKES THE VALUE ZERO WHEN RL - BLTC IS GREATER THAN XLD
C IFLG 1 BOLT HAS NO CARTRIDGE
C 2 BOLT HAS JUST THE CASE
C 3 BOLT HAS CASE AND PROJECTILE

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0024 TFIRE=0.00000100
C IRBC TAKES ON THE VALUE ONE WHEN THE SPRING FORCE BETWEEN THE BOLT
C CARRIER AND THE BOLT IS ON, AND WHEN WEAPON IS UNLOCKED
C INIT WHEN WEAPON JUST BEGINS TO COCK IT TAKES ON THE VALUE ONE,
C AND IS SET BACK TO ZERO WHEN WEAPON FIRES.
C IFIRE WHEN THE HAMMER ANGLE (THEIA) BECOMES GREATER THAN ZERO, THEN
C THE WEAPON FIRES. (IFIRE IS SET TO ONE AT THIS POINT)
C IFIRE IS SET BACK TO ZERO WHEN BOLT BREAKS AWAY FROM THE GUN
C NROUN NUMBER OF ROUNDS DESIRED
C IFREQ FREQUENCY THAT THE FORCES ARE PRINTED OUT.
C IF IREAD IS ONE THEN READ IN X AND XD
C IPICK TAKES ON THE VALUE ONE WHEN THE BOLT CARRIER GETS
C BACK FAR ENOUGH TO PICK UP ANOTHER ROUND, AND GOES BACK TO ZERO WHEN THE
C GUN FIRES
C NSMPR SAMPLE NUMBER FOR DETERMINING SAMPLE RATE IN GRAPHING
0025 NAMELIST/MODEY,IRND,IMODE,ISEP,IUNLO,THAM,IFLAG,IFLG,IBBC,INIT,
1 IFIRE,IFREQ,NROUN,IREAD,IPICK,NSMPR,I*DDG,NMASS
PRINT 1776
1776 FORMAT(30X,'CONTROL VARIABLES',/)
WRITE(6,MODEY)
READ 100, PBRCH
READ 2,NBRCH,(PBRCH(I),I=1,NBRCH)
PBRCH(NBRCH + 1) =PBRCH(NBRCH)
READ 100, PCAVT
READ 2,NCAY,(PCAVY(I),I=1,NCAY)
PCAVY(NCAY + 1)=PCAVY(NCAY)
2 FORMAT(110,/, (8F10.5))
PRINT 102, PBRCH, ( PBRCH(I),I=1,NBRCH)
102 FORMAT(30X,'BREECH PRESSURE (LB/FT**2)',/
1 30X, ' TIME STEP=',F16.8,/, (6F15.2))
PRINT 103, PCAVT, (PCAVY(I),I=1,NCAY)
103 FORMAT(20X,' CAVITY PRESSURE IN BOLT CARRIER (LB/FT**2)',/
1 30X, ' TIME STEP=',F16.8,/, (6F15.2))
READ 100,CARTX
READ 2,NCAR,(CARTF(I),I=1,NCAR)
PRINT 165, CARTX,(CARTF(I),I=1,NCAR)
CARTF(NCAR + 1)=CARTF(NCAR)
165 FORMAT(30X,' CARTRIDGE CASE FORCE (LBS) ',/ 30X,
1, ' TIME STEP=',F16.8,/, (6F15.2))
'XI=NBRCH - 1
PBRL=XI*PBRCH
XI=NCAY - 1
PCAL=XI*PCAVT
XI=NCAR - 1
PCAH=XI*CARTX
READ 100,FRORX
READ 2,NBOR,(FBORY(I),I=1,NBOR)
XI=NBOR - 1
PBOR=XI*FRORX
PRINT 7001,FBORX,(FBORY(I),I=1,NBOR)
7001 FORMAT(30X,' BORE FRICTION (LBS)',/ 30X,
1, ' TIME STEP=',F16.8,/, (6F15.2))

```

```

0058 NAMELIST/INPT1/
1 XMB, G
0059 NAMELIST/INPT2/XRBC,XKRBC,XCRBC,DERIV,XKLBC,XMRUL,XCLBC,
      BOREA,XKHYP,XCHYP,XLD,CF4,XMU4
0060 READ(5,INPT1)
      ,TDEL,XIB ,XMARC
0061 READ(5,INPT2)
      C CONVERTS FROM POUNDS TO SLUGS
      XMRUL=XMBUL/G
      XMAHC=XVARC/G
      XMB=AMRZ/G
0062 NAMELIST/INPT3/TPORT,XKBUFF,XCRBUF,XCRCMG,XIM,PH,ALPHA,
      1 XMOUN,XCMOU,TSPRIN,XMUBCG,XMBC,RIB,R2B,CCL,XMUB
0063 NAMELIST/INPT4/XK9,XC9,XK5,XC5,BETA,ALP1,THE10,XKHG,XCHG,XHAM,
      1 XMG,XMRUFF ,EJCAR,ELEV,XPUC,XPUC1,RO
0064 NAMELIST/INPT5/XBFORW,XCFORW,XRREAR,XCMEAR,SLACK,XUBUF,XMWT
      1,IGR1,TGR2,XUHBC,ADHA,FXTC,EXTF,FDIS,XMUFF,XCUFF,FTSTIC
0065 2,RHH0,RHH1,ALH0,ALH1,HK,HC
      C COMMENTS ABOUT INPUTS
      C TGR1 ESTIMATED TIME FOR THE FIRST SHOT
      C TGR2 ESTIMATED TIME FOR SUBSEQUENT SHOTS
0068 READ(5,INPT3)
0069 READ(5,INPT4)
0070 READ(5,INPT5)
0071 CALL READIN
      C CONVERTS FROM POUNDS TO SLUGS
      XHAM=XHAM/G
      XMG=XMG/G
0072 XMRUFF=XMRUFF/G
0073 XMRBC=XMRBC/G
0074 XMAHC=XMAHC/G
0075 XMAWT=XMAWT/G
0076 PRINT 9
0077 9 FORMAT(' NAME ',29X,'DESCRIPTION',39X,'VALUE',2X,'UNITS',/)
0078 PRINT 10
0079 10 FORMAT(
0080 5' BOREA',4X,'HORE AREA',51X,F20.8,2X,'FT**2',//,
      6 ' XKHYP',4X,'SPRING THAT PREVENTS MOTION BETWEEN THE ROLT AND GUN
      7',8X,F20.8,2X,'LBS/FT',/)
0081 PRINT 11,XCHYP,XLD,CF4,XMU4,XMR,G
0082 11 FORMAT(' XCHYP',4X,'DAMPING FOR THE ABOVE SPRING',32X,F20.8,
      2 2X,'LBS-SEC',//, ' XLD',6X ,
      2'DISTANCE ROLT CARRIER TRAVELS BEFORE GUN IS UNLOCKED',8X,
      * F20.8,2X,'FT',// ,
      3 ' CF4',6X,'DAMPING COEFFICIENT BETWEEN BOLT AND BOLT CARRIER',11X
      4 ,F20.8,2X,'LBS-SEC/FT',//, ' XMU4',5X,
      5'COEFFICIENT OF FRICTION BETWEEN BOLT AND ROLT CARRIER',7X,F20.8,
      6 2X,'NONE',//, ' XMB',6X,'MASS OF THE ROLT',44X,F20.8,2X,'SLUGS',//
      7 , ' G',8X,'GRAVITATIONAL CONSTANT',38X,F20.8,2X,'FT/SEC**2',//)
0083 PRINT 12 ,XBBC,XKRBC,XCRBC,DERIV,XKLBC,XMBUL
0084 12 FORMAT(' XBBC',5X,'DISTANCE BOLT TRAVELS IN THE BOLT CARRIER',19X,
      1 F20.8,2X,'FT', //, ' XKRBC',4X,
      2'SPRING BETWEEN BOLT AND BOLT CARRIER',24X,F20.8,2X,'LBS/FT',//,
      3' XBBC',4X,'DAMPING COEFFICIENT FOR ABOVE SPRING',24X,F20.8,2X,
      4'LBS-SEC/FT',//, ' DERIV',4X,'DERIVATIVE OF THETA WITH RESPECT TO X

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0085 48-XBC1,1HX,F20.8,2X,IRAD/FT,/,/
0086 51 XKLRC,4X,SPRING BETWEEN BOLT AND ROLT CARRIER DURING LOCKING,
69X,F20.8,2X,ILBS/FT,/,/ XMRUL,4X,MASS OF THE BULLET AND THE PR
70PELLANT,23X,F20.8,2X,SLUGS,/,/
PRINT 13,XCLBC ,APEROL ,TDEL,XIR,XMARC
13 FORMAT( ,XCLBC, ,4X, ,DAMPING CORRESPONDING TO THE SPRING XKLBC ,
118X,F20.8,2X,ILBS-SEC/FT,/,/
31 ,AMEROL,3X,AREA OF CAVITY THE GAS FORCE ACTS ON,24X,F20.8,2X,
41FT*2,/,/
41 TDEL,5X,TIME DELAY BETWEEN PORT AND CAVITY,26X,F20.8,2X,
51 SEC,/,/
51 XIB,6X,MOMENT OF INERTIA OF THE ROLT,31X,
5F20.8,2X,SLUGS-FT*2,
6,/,/ ,XMARC,4X,MASS OF THE CARTRIDGE CASE ,33X,F20.8,2X,SLUGS
6,/,/
PRINT 14,TPORT,XKHUFF,XCRUFF,XCBCMG,XIM,IRM
14 FORMAT( ,TPORT,4X,TIME IT TAKES PROJCTILE TO REACH PORT,22X,
1 F20.8,2X,SEC,/,/ ,XKHUFF,3X,
2 ,INTERACTION SPRING BETWEEN BOLT CARRIER AND RUFFER',
310X,F20.8,2X,ILBS/FT,
3 ,/,/ ,XCRUFF,3X, ,DAMPING COEFFICIENT FOR ABOVE SPRING,24X,
4 F20.8,2X,ILBS-SFC/FT,/,/ ,XCBCMG,3X,
4 ,DAMPING BETWEEN BOLT CARRIER AND MAIN GUN,19X,F20.8,2X,
4ILBS-SEC/FT,
5. ,/,/ ,XIM,5X,MOMENT OF INERTIA OF THE HAMMER,29X,F20.8,2X,SL
6UGS-FT*2 ,/,/ ,RM,7X,RADIAL DISTANCE OF THE HAMMER TO POINT OF
7 CONTACT,11X,F20.8,2X,FT,/,/
PRINT 15,ALPHA
15 FORMAT( ,ALPHA,4X,REFERENCE ANGLE ON THE HAMMER,31X,F20.8,2X,
2,RAD,/,/
31 XKMOUN,3X, ,MOUNT SPRING CONSTANT,39X,F20.8,2X,ILBS/FT,/,/
41 XCMOUN,3X, ,DAMPING FOR ABOVE SPRING,36X,F20.8,2X,ILRS-SEC/FT,
6/,/ ,TSPRIN,3X, ,CONSTANT TORQUE DUE TO THE HAMMER SPRING,20X,
6F20.8,2X,FT-LBS,/,/
PRINT 16,XMUBCG,XMHC,XK9,XK5,XG5,BETA
16 FORMAT( ,XMUBCG,3X, ,COEFFICIENT OF FRICTION BETWEEN THE BOLT CARR
1RIER AND GUN,3X,F20.8,2X,NONE,/,/
21 XMBC,5X, ,MASS OF THE BOLT CARRIER,36X,F20.8,2X,SLUGS,/,/
31 XK9,6X, ,BACK PLATE SPRING CONSTANT,34X,F20.8,2X,ILRS/FT,/,/
4,/,/ ,XK5,6X, ,DRIVE SPRING CONSTANT,39X,F20.8,2X,ILBS/FT,/,/
51 XG5, 6X, ,DAMPING FOR ABOVE SPRING,36X,F20.8,2X,ILBS-SEC/FT,/,/
6 , BETA,5X, ,PRELOAD DISTANCE FOR THE DRIVE SPRING,23X,F20.8,2X,
7,FT,/,/
PRINT 17,ALP1,THETO,XKHG,XCHG,XHAM,XMG
17 FORMAT( ,ALP1,5X, ,DISTANCE BUFFER TRAVELS BEFORE BACK PLATE IS HI
1T,12X ,F20.8,2X,FT,/,/
21 THETO,4X, ,INITIAL ANGLE OF THE HAMMER,33X,F20.8,2X,RAD,/,/
31 XKHG,5X, ,SPRING BETWEEN HAMMER AND GUN,31X,F20.8,2X,ILBS/FT,/,
4/,/ ,XCHG, 5X, ,DAMPING FOR THE ABOVE SPRING,
532X,F20.8,2X,ILBS-SEC/FT,
5,/,/ ,XHAM,4X, ,MASS OF THE HAMMER,42X,F20.8,2X,SLUGS,/,/
61 XMG,6X, ,MASS OF THE MAIN GUN,40X,F20.8,2X,SLUGS,/,/

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0095
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PRINT 18,XHBUFF,EJCAR,FLEV,XPUC,XPUC1,RO
FORMAT(1,XHBUFF,1JX,MASS OF THE BUFFER,42X,F20.8,2X,SLUGS,
1//,1,EJCAR,14X,1DISTANCE BOLT TRAVELS BEFORE CARTRIDGE EJE
2CTION,12X,F20.8,2X,FT,/,1,ELEV,15X,
3ANGLE OF ELEVATION OF THE GUN,31X,F20.8,2X,RAD,/,/
3 XPUC,5X,1DISTANCE OF ROLT WHEN ROUND IS PICKED UP,20X
4 F20.8,2X,FT,/,/, XPUC1,4X,1DISTANCE OF ROLT WHEN PIC
5K UP FORCE IS OFF,18X,F20.8,2X,FT,/,/, RO,7X,
6DISTANCE FROM THE PIVOT POINT TO THE C.G. OF THE HAMMER, 5X
6,F20.8,2X,FT,/,/)

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0097
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PRINT 19,XRFORM,XCFORM,XRREAR,XCREAR
FORMAT(1,XRFORM,13X,EFFECTIVE INTERNAL BUFFER SPRING CONSTANT (FO
1WARD),9X,F20.8,2X,LRS/FT,/,/
21 XCFORM,3X,1DAMPING COEF. FOR THE ABOVE SPRING,26X,F20.8,2X,
3LRS-SEC/FT,/,/, XRREAR,3X,EFFECTIVE INTERNAL BUFFER SPRING CON
4STANT (REARWARD),8X,F20.8,2X,LRS/FT,/,/
51 XCREAR,3X,1DAMPING COEF. FOR THE ABOVE SPRING,26X,F20.8,2X,
5LRS-SEC/FT,/,/)

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PRINT 20,SLACK,XURUF,XMWT,XUHHC,XDHA
FORMAT(1,SLACK,14X,DISTANCE WTS. ARE ALLOWED TO MOVE,27X,F20.8,
12X,FT,/,/
21 XUBUF,14X,1COEF. OF FRICTION BETWEEN WTS. AND BUFFER,19X,
3 F20.8,2X,NONE,/,/
41 XMWT,5X,1MASS OF THE WTS IN THE BUFFER,31X,F20.8,2X,SLUGS,/,/
51 XUHHC,14X,1COEF. OF FRICTION BETWEEN HAMMER AND R.C.,19X,
6F20.8,2X,NONE,/,/, XDHA,5X,
7DISTANCE R.C. MOVES BEFORE FRICT. BETWEEN THE HAM. AND B.C.,
81X,F20.8,2X,FT,/,/)

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0101
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PRINT 31,EXTC,EXTF,FDIS,XMUJFF,XCUFF,FCSTIC
FORMAT(1,EXTC,5X,EXTRACTOR DISTANCE,141X,F20.8,2X,FT,/,/
1 EXTFF,5X,EXTRACTOR FORCE,45X,F20.8,2X,LBS,/,/
2 7X,F20.8,2X,FT,/,/
31 XMUJFF,14X,1COEF. OF FRIC. BETWEEN GUN AND BUFFER,23X,
4F20.8,2X,NONE,/,/
41 XCUFF,14X,1DAMPING BETWEEN BUFFER AND GUN,30X,F20.8,2X,NONE,/,/
51 FCSTIC,14X,1STICION FORCE,46X,F20.8,2X,LRS,/,/

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0103
0104

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PRINT 35,RHH0,RHH1,ALH0,ALH1,HK,MC
FORMAT(1,RHH0,5X,1DISTANCE FROM HAMMER PIVOT TO B.C.,26X,
1F20.8,2X,FT,/,/, RHH1,5X,
2DISTANCE FROM PIVOT TO TOP OF HAMMER,23X,F20.8,2X,FT,/,/
3 ALH0,5X,1ANGLF CORRESPONDING TO RHH0,33X,F20.8,
32X,RAD,/,/, ALH1,5X,1ANGLE CORRESPONDING TO RHH1,32X,F20.8,
12X,RAD,/,/, HK,7X,1HAMMER IMPACT SPRING,40X,F20.8,2X,
1LRS/RAD,/,/, HC,7X,1ABOVE DAMPING CONSTANT,38X,F20.8,
12X,LRS-SEC/RAD,/,/)

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0105
0106

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PRINT 39,R1B,R2B,CCL,XMUB
FORMAT(1,R1B,6X,1INNER RADIUS CARTRIDGE CASE,1
132X,F20.8,2X,FT,/,/, R2B,6X,1OUTER RADIUS CARTRIDGE CASE,1
232X,F20.8,2X,FT,/,/, CCL,6X,
3CARTRIDGE CASE LENGTH,38X,F20.8,2X,FT,/,/, XMUB,5X,
4COEFFICIENT OF FRICTION AT THE BASE,25X,F20.8,2X,NONE,/,/
XMBV(1)=XMB

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0107

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0108 XHRV(2)=XMB + XHARC
0109 XHRV(3)=XMR + XMBUL + XHARC
0110 X(5)=THETO
0111 CON1= DSIN(ELEV)
0112 CON2=DCOS(ELEV)
0113 IF (IREAD.EQ.1) GO TO 1700
C
0114 COMPUTE EQUILIBRIUM POINTS
0115 X(2)=-CON1*(XHMT+ XHBC + XMBUFF)*G+BETA*XXS)/XKLCB
0116 X(4)=X(2) +(-CON1*(XHMT + XMBUL)*G+BETA*XXS)/XKBUFF
0117 IF (ELEV.GT.0.000)X(6)=X(4) - SLACK
0118 IF (ELEV.LE.0.000)X(6)=X(4)
0119 DO 1915 I=1,4
1915 X(6 + I)=X(6)
0120 READ 100,PRMT
0121 IF (IREAD.NE.1) GO TO 1700
C
0122 NOTE: X(6) CAN TAKE ON VALUES BETWEEN ZERO AND SLACK
0123 READ 100, X
0124 READ 100,XD
100 FORMAT(8F10.5)
C ***FOLLOWING DETERMINES SAMPLING RATE FOR PLOTTING PURPOSES
170 XN=NROUN
    WTI=0.000
1929 NP= .075/PRMT(3)
    ISR=NP/NSMPR
    IF (ISR.EQ.0) ISR=1
    ITMAX= TGN1 + PRMT(1)
    ITMIN=PRMT(1)
    IXX=IRND
0132 CALL READIT
0133 IF (I4DG.NE.1) GO TO 2900
0134 CALL READUM
0135 CALL INITIAL
0136 C INIEGRATION ROUTINE BEGINS HERE
2900 KNUM=7 + NMASS
    IF (I4DG.EQ.1)KNUM=11 + NMASS
0138 PRINT 1938,NROUN
1938 FORMAT('1',15X,'COMPUTED RESULTS FOR A ',I2,' SHOT BURST',///)
    N11=7
0141 IF (I4DG.EQ.1) N11=11
0142 L11=KNUM - 1
0143 DO 1942 I=N11,L11
0144 1942 X(I)=X(4)
0145 CALL RUNGE(KNUM,FCT,OUTP)
0146 CALL EXIT
0147 END
0148

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0001 SURROUTINE FCI(T,X,XD,XDD,IITEST)
0002 IMPLICIT REAL*(A-H,O-7)
0003 DIMENSION X(21),XD(21),XDD(21)
0004 COMMON/MODES/IMODE,IFG(30),IPT,IPHAM,IUNLO,ISEP,IRBC,INIT,IRND
1,NROUN,ISR,IFEQ,IRX,IPICK,I40G
COMMON/ASTHT/PHRCX ,PHRCY(50),TFIRE,CCL,XLAMCC,XMUCC,ACC,ROREA,
1 AKHYP,ACHYP,CF4,XMU4,XMR,XLD,PHRL,PCAL,CAPT(50),CARTX,PCAR
COMMON/NUM/NARCH,IFLAG,NCAV,IFLG,IFIRE
COMMON/ASTHEE/PCAVT ,PCAVY(50),FHORY(50),PHORE,FRORX
COMMON/ASTT/XHRC,XHRC,XHRC,DERIV,XMRUL,XMARC ,PUC,AREROL,
1 TEL,XIR,G,IPORT,XCRCMG,XMURCG,EJCAR,FLEV ,XKLBC,XCLBC
COMMON/ER/XBUFF,XBUFF,XMRC,XMBV(3),XIM,PH,ALPHA,TSPRIN,
1 RHH0,ALH0,RHH1,ALH1,HK,HC,
1 DCOCK ,XKMCUN,XCMOUN,XUHRC,XDHA,EXTC,EXTF,FNIS,FSTIC
1,R1B,R2B,XMUB,FCOIX
COMMON/S/XK9,XC9,XK5,XC5,BETA,ALP1,THEU,XKHG,XCHG,XHAM,XMG,
1 XIRUFF,CON1,XPUC,XPUC1,RO,CON2,XMUFF,XCUFF
COMMON/BUFF/XBFORM,XCFORW,XHREAR,XCREAR,SLACK,XUBUF,XMWT,WTI
COMMON/CNTRL/DELT,SUM
COMMON/PRT/FB(12),FBC(12),FMG(15),FRUF(6)
COMMON/DIF/DIFF1,DIFF2,DIFF3,DIFF4,DIFF5,DIFF6
COMMON/MSSS/XMS(5),XKS(6),XCS(6)
DATA TPEF/-10.007,TCOMP/0.000/,IKPTRK/0/ ,IBOR/0/,IHD/0/,IDIO/0/
COMMON/MAS/MMASS
C
C COMMENTS
C X(1),XD(1) CORRESPOND TO THE BOLT
C X(2),XD(2) CORRESPOND TO THE BOLT CARRIER
C X(3),XD(3) CORRESPOND TO THE MAIN GUN
C X(4),XD(4) CORRESPOND TO THE RUFFER
C X(5),XD(5) CORRESPOND TO THE HAMMER
C X(6),XD(6) CORRESPOND TO THE WTS. IN THE BUFFER
C X(7),XD(7),X(8),X(9),X(10) CORRESPOND TO RUFFER WEIGHTS
C X(11)..... CORRESPOND TO DRIVE SPRING MASSES
C X(12)..... CORRESPOND TO DRIVE SPRING MASSES
C LAST DEFINED PARAMETER IS THE GUN ROTATION
DIFF1=X(1) - X(3)
DIFF2=X(1) - X(2)
DIFF3=X(1) - XD(2)
DIFF4=X(3) - X(2)
DIFF5=X(4) - X(2)
DIFF6=X(4) - X(4)
DIFF7=X(6) - X(4)
DIFF8=X(6) - XD(4)
DIFF9=X(3) - X(4)
DIFF10=X(3) - XD(4)
DIFF11=X(6) - X(4)
DIFF12=X(6) - XD(4)
IF (IPICK.EQ.0.AND.DIFF3.GT.XPUC) IPICK=1
IF (DIFF2.GE.XBHC.AND.IRBC.EQ.0) IRBC=1
IF (IRBC.EQ.1.AND.DIFF1.GE.0.000.AND.IFIRE.EQ.0.AND.DIFF2.LT.XBHC)
1 IRBC=0
IF (IUNLO.EQ.1.AND.DIFF1.GT.0.000.AND.DIFD1.LT.0.000.AND.IFLG.EQ.3
1.AND.IITEST.EQ.1) GO TO 79
0018
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0034 IF(IFLAG.EQ.0.AND.DIFF3.LT.FDIS.AND.IRRC.EQ.0) IFLAG=1
0035 GO TO 78R
0036 79 IUNLO=0
0037 XD(1)=XD(3)
0038 78R IF(-DIFF1.GT.EJCAR.AND.IFLG.EQ.2) IFLG=1
0039 IF(IFLG.EQ.1.AND. IFG(21).NE.1) GO TO 7500
0040 GO TO 7501
0041 7500 PRINT 7502,T
0042 7502 FORMAT(' CARTRIDGE CASF IS EJECTED AT T=',F15.8)
0043 IFR(21)=1
0044 7501 IF(X(5).GE.0.000.AND.IFIRE.EQ.0.AND.IFLG.EQ.3) GO TO 75
0045 6655 IF(DIFF3.GE.XLD.AND.IFLAG.EQ.1) IFLAG=0
0046 IF(IRRC.EQ.1.AND.IFIRE.EQ.1.AND.F8C(8).LT.1.D-20) IFIRE=0
0047 GO TO 700
0048 75 IF(IDID.EQ.0) TNOV=T
0049 IF(INIT.NE.0) INIT=0
0050 IF(IDID.NE.1) IDID=1
0051 IF(T.LT.TNOV + TBFO) GO TO 6655
0052 IDIU=0
0053 IFIRE=1
0054 TFIRE=T
0055 IFG(21)=0
0056 IFLG=2
0057 INIT=0
0058 ISEP=0
0059 IRND=IRND + 1
0060 IPICK=0
0061 IF(IRND.EQ.1) THETA=THETO
0062 IF(IRND.EQ.1)THETO=THETO - .122
0063 PRINT 150,IRND
0064 150 FORMAT(' *****',ROUND NUMBER ',11', REGINS ',
1,*****')
0065 PRINT 100,TFIRE
0066 100 FORMAT(' WEAPON FIRES AT T=',F15.8)
C
0067 COMPUTE FR(1)
0068 700 IFLI=T - TFIRE
0069 IF(IHAM.EQ.1.AND.XD(5).LT.0.000)IHAM=0
0070 IF(IHCC.EQ.1.AND.IHAM.FQ.0.AND.IUNLO.EQ.1) IHAM=1
0071 IF(TFL.GT.P9RL.OR.TFL.LT.0.000.OR.IFLG.EQ.3) GO TO 10
0072 FR(1)=-XINTP(TFL,PBRXY,PBRXY)*BOREA
0073 IF(IFG(1).EQ.1) GO TO 11
0074 IFG(1)=1
0075 PRINT 550,T
0076 550 FORMAT(' BREECH FORCE IS ON AT T=',F15.8)
0077 GO TO 11
0078 10 FR(1)=0.000
0079 IF(IFG(1).EQ.0) GO TO 11
0080 IFG(1)=0
0081 PRINT 551,T
0082 551 FORMAT(' BREECH FORCE IS OFF AT T=',F15.8)
C
0083 COMPUTE FB(2)
0084 11 IF(-DIFF1.LT.XPUC.AND.DIFF3.GT.XPROJ.AND.IFLG.NE.3.AND.
1XD(2).LT.0.000) GO TO 8915

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0083 GO TO 8916
0084 TEMP=IKPTRK
0085 FB(2)=(XME*G*CONZ+ XKHAG*IDELSP*TEMP - XFL + YINT))* 2. *
      10SIGN(1.00, DIFD1)*XUMAG1
0086 IF(IFG(30).EQ.1) GO TO 13
0087 IFR(30)=1
0088 PRINT 8917,T,FB(2)
0089 8917 FORMAT(' AMMO AND BOLT FRICTION IS ON AT T=',F15.8,'FRIC=',F15.8)
0090 GO TO 13
0091 8916 FB(2)=0.000
0092 IF(IFG(30).EQ.0) GO TO 13
0093 IFR(30)=0
0094 PRINT 8918,T
0095 8918 FORMAT(' BOLT AND AMMO FRICTION IS OFF AT T=',F15.8)
      C
      COMPUTE FR(3)
0096 13 IF(1*IFIRE*EQ.1.OR.IUNLO.EQ.0 .OR.DIFF1.LT.-EXTC.OR.IFLG.EQ.2)
      1 GO TO 14
0097 FR(J)=-EXTF
0098 IF(DIFF1.GT.0.000) GO TO 9916
0099 GO TO 9917
0100 9916 FR(3)=-XKHYP*OJFF1 - XCHYP*OJFD1 +FB(3)
0101 9917 IF(IFG(3).EQ.1) GO TO 15
0102 IFR(3)=1
0103 PRINT 554,T,FR(3)
0104 554 FORMAT(' SPRING EXTRACTOR FORCE IS ON AT T=',F15.8,' CONSTANT FORC
      LE', F15.8)
      GO TO 15
0105 14 FR(3)=0.000
0106 15 IF(IFG(3).EQ.0) GO TO 15
0107 IFR(3)=0
0108 PRINT 556,T
0109 556 FORMAT(' SPRING BETWEEN BOLT AND BARREL IS OFF AT T=',F15.8)
0110 COMPUTE FR(4)
      C
0111 15 FB(4)= - CF4*OJFD2 + XMU4*XHBV(IFLG)*G*DSIGN(1.000,-DIFD2)*CONZ
      COMPUTE FR(5)
0112 IF(188C.EQ.0.OR.(DIFF2.LT.X88C.AND.DIFF2.GT.X88C - .52083D-2)
      1.OR.IFG(13).EQ.1) GO TO 17
0113 IF(DIFF2.LE.X88C-.52083D-2) GO TO 9161
0114 FR(5)=-XKHRC*OJFF2 - XHRC - XC88C*OJFD2
0115 IF(FR(5).GT.0.000) FR(5)=0.000
      GO TO 9162
0116 9161 FR(5)=-XKHRC*OJFF2 - X88C + .52083D-2 - XC88C*OJFD2
0117 IF (FR(5).LT.0.000) FR(5)=0.000
0118 9162 IF(IFG(4).EQ.1) GO TO 80
0119 IFR(4)=1
0120 PRINT 557,T
0121 557 FORMAT(' SPRING FORCE BETWEEN THE BOLT AND BOLT CARRIER IS ON AT T
      1=',F15.8)
      GO TO 80
0123 17 FR(5)=0.000
0124 IF(IFG(4).EQ.0) GO TO 80
0125 IFR(4)=0
0126 PRINT 558,T
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0128 558 FORMAT(' SPRING FORCE BETWEEN BOLT AND BOLT CARRIER IS OFF AT T=',
      1 F15.8)
      C COMPUTE FB(6) AND FB(7) (ZERO UNLESS SPECIFIED OTHERWISE)
0129 FB(6)=0.000
0130 FB(7)=0.000
      C COMPUTE FR(8)
0131 IF(DIFF2-GE.0-0.000) GO TO 20
0132 FB(8)=- XLRG*(DIFF2) - XCLBC*DIFFD2
0133 IF(FH(R).LT.0.000)FB(8)=0.000
0134 IF(IFG(5).EQ.1) GO TO 201
0135 IFG(5)=1
0136 PRINT 559,T
0137 559 FORMAT(' (FORWARD) BOLT PIN STRIKES END OF CAM PATH AT T=',F15.8)
0138 GO TO 201
0139 20 FB(8)=0.000
0140 IF(IFG(5).EQ.0) GO TO 201
0141 PRINT 560,T
0142 560 FORMAT(' (FORWARD) FORCE AT END OF CAM PATH IS OFF AT T=',F15.8)
0143 IFG(5)=0
      C COMPUTE FB(9),
0144 IF(IPICK.EQ.0) GO TO 21
0145 IF(-DIFF1.LT.XPUC.AND.-DIFF1.GT.XPUC1.AND.
      1 XHBC*XD(2) + XMBV(IFLG)*XD(1).GT.0.000.AND.(ITEST.EQ.1.OR.
      2 IFG(6).EQ.1).AND.T - TOFF.GT.TCOMP)GO TO 2111
      GO TO 21
0146 2111 TEMP=IKPTRK
0147 FB(9)=DTAC*(XME*G*CON2 + XKMAG*(DELSP*TEMP - XFL + YIHT))*
0148 1(XUMAG2 + XUMAG3)*DSIGN(1.D0,DIFFD1)/DELSP
0149 IF(IFG(6).EQ.1) GO TO 22
0150 PRINT 561,T
0151 561 FORMAT(' FORCE TO PICK UP NEW ROUND IS ON AT T=',F15.8)
0152 IFG(6)=1
0153 IFLG=3
0154 PRINT 7553
0155 7553 FORMAT('/',, INELASTIC COLLISION OF THE ROLT AND ROUND')
0156 PRINT 7550,XMG,XD(1),XD(3)
0157 7550 FORMAT('/',, MASS OF THE GUN AND VELOCITIES OF THE ROLT AND ROUND ',
      1 /, ' PRIOR TO ROUND BEING PICKED UP',/, ' GUN MASS=',F15.8,
      2 ' BOLT VELOCITY=',F15.8, ' ROUND VELOCITY (VELOCITY OF THE GUN)='/,
      3 F15.8)
0158 XMG=XMG-(XMARCV*XMBUL)
0159 XD(1)=(XD(3)*(XMARCV*XMBUL) + XD(1)*XMRV(IFLG))/
      1(XMARCV + XMBUL + XMBV(IFLG))
0160 PRINT 7589,XMG,XD(1),XD(1)
0161 7589 FORMAT('/',, MASS OF THE GUN AND VELOCITIES OF THE ROLT AND ROUND ',
      1 /, ' AFTER ROUND IS PICKED UP',/, ' GUN MASS=',F15.8,
      2 ' BOLT VELOCITY=',F15.8, ' ROUND VELOCITY =',F15.8)
      PRINT 7599,FB(9)
0162 7599 FORMAT(' FRICTIONAL FORCE TO PICK UP NEW ROUND IS',F15.8)
0163 GO TO 22
0164 21 FB(9)=0.000
0165 IF(IFG(6).EQ.0) GO TO 22
0166 PRINT 562,T

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0168 562 FORMAT(' FORCE TO PICK UP NEW ROUND IS OFF AT T=,F15.8)
0169 IFG(16)=0
0170 TOFF=T
0171 IKPTRK=IKPTRK + 1
0172 TEMP=IKPTRK
0173 FMI=XKMG*(TEMP*DELSP + YIHT - XFL)
0174 FMA=XKWAG*((TEMP - 1.)*DELSP + YIHT - XFL)
0175 TCOMP=CSORT(AVE/(EPSOL*XKMAG))*DARCOS(FMI/FMA)
0176 PRINT 7600,TCOMP,FMI,FMA
0177 7600 FORMAT(/,' RESPONSE TIME OF THE MAGAZINE IS, F15.8,' FMI=,
      1,F15.8,' FMA=,F15.8)
      C
      C COMPUTE FR(10)
0178 22 TGAS= T - TFINE - TPORT - TDEL
0179 IF(TGAS.LT.0.000.OR.TGAS.GT.PCAL .OR.IFLG.EQ.3) GO TO 23
0180 FR(10)=XINTP(TGAS,PCAVT,PCAVY)*AREBOL
0181 IF(IFG(8).EQ.1) GO TO 24
0182 PRINT 563,T
0183 563 FORMAT(' CAVITY FORCE IS ON AT T=,F15.8)
0184 IFG(8)=1
0185 ISEP=1
0186 GO TO 24
0187 23 FR(10)=0.000
0188 IF(IFG(8).EQ.0) GO TO 24
0189 PRINT 564,T
0190 564 FORMAT(' CAVITY FORCE IS OFF AT T=,F15.8)
0191 IFG(8)=0
0192 24 IF(IUNLO.EQ.0.OR.IBRC.EQ.0) GO TO 5510
      C
      C TRACL=1 - IBRC
0193 IF(THRC1.GT.PCAR) GO TO 5510
0194 FR(11)= XINTP(IBRC1,CARTX,CARTF)
0195 IF(-DIFFL.GT.CCL.OR.-XNBC*DIFD3.GT.0.000) GO TO 5510
0196 FR(11)=-CARTF(1)*(DIFL + CCL)/CCL
0197 IFG(20)=1
0198 PRINT 5540,T
0199 5540 FORMAT(' EXPERIMENTAL CARTRIDGE CASE EJECTION FORCE IS ON AT T=
      1, F15.3)
      C
0200 GO TO 5520
0201 5510 FR(11)=0.000
0202 IF(IFG(20).EQ.0) GO TO 5520
0203 IFG(20)=0
0204 PRINT 5530,T
0205 5530 FORMAT(' EXPERIMENTAL CARTRIDGE CASE EJECTION FORCE IS OFF AT T=
      1,F15.8)
      C
      C COMPUTE FORCES ON THE BOLT CARRIER
0206 5520 FB(12)=-CONI*G*XMBV(IFLG)
0207 FBC(1)= - FB(4)
0208 FBC(2)= - FB(5)
0209 FBC(3)=-FB(6)
0210 FBC(4)=-FB(8)
0211 FBC(5)=-FB(7)
      C
      C COMPUTE FBC(6)
0211 FBC(6)=XBUFF*DIFD4 + XCBUFF*DIFD4

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0212 IF(FBC(6).LT.0.000)FBC(6)=0.000
0213 GO TO 29
0214 25 FBC(6)=0.000
C      COMPUTE FRC(7)
0215 29 FRC(7)=-XCRMG*(-DIFD3) + XMBCG*(XMBV(IFLG) + XMBC)*G*DSIGN(1.D0,
10IFD3)*CON2
C      COMPUTE FRC(8)      (ENTERS INTO THE CONSTRAINT EQUATIONS)
0216 27 FRC(8)=0.000
C      COMPUTE FRC(9)
0217 28 FRC(9)=-FB(10)
C      FRICTIONAL FORCE ACTING ON THE B.C. DUE TO THE AMMO IS COMPUTED HERE
0218 IF(T - TOFF.GT.TCOMP.AND.DIFF3.LT.XPROJ.AND.IFG(6).EQ.0)
160 TO 9600
GO TO 9601
0219 GO TO 9601
0220 TEMP=IKPIRK
0221 FRC(10)=(XMEG*CON2+XK*AG*(DELS*TEMP - XFL + YIHT))*2.0
10SIGN(1.00,-DIFD3)*XUMAG1
0222 IF(IFG(28).EQ.1) GO TO 9650
0223 PRINT 9607,T,FRC(10)
0224 IFG(28)=1
0225 9607 FORMAT(' FRICTIONAL FORCE BETWEEN CARTRIDGES AND B.C. IS ON AT T=
1',F15.8,' FRC(10)=',F15.8)
GO TO 9650
0226 9601 FRC(10)=0.000
0227 IF(IFG(28).EQ.0) GO TO 9650
0228 IFG(28)=0
0229 PRINT 9609,T
0230 9609 FORMAT(' FRICTIONAL FORCE BETWEEN CARTRIDGES AND B.C. IS OFF AT T=
1',F15.8)
0231 9650 IF(DIFF3.LT.XDHA.OH.IFG(12).NE.2) GO TO 9705
0232 FBC(11)=-DSIGN(1.D0,DIFD3)*TSPRIN*XUHC*2./(RHH1*CONF1*OSIN(X(5)))
0233 IF(IFG(29).EQ.1) GO TO 9706
0234 IFG(29)=1
0235 PRINT 9707,T,FBC(11)
0236 9707 FORMAT(' FRICTION FORCF COMES ON BETWEEN HAMMER AND B.C. AT T=',
1F15.8,' FRC(11)=',F15.8)
GO TO 9706
0237 9705 FRC(11)=0.000
0238 IF(IFG(29).EQ.0) GO TO 9706
0239 PRINT 9709,T
0240 9709 FORMAT(' FRICTION FORCE BETWEEN HAMMER AND B.C. IS OFF AT T=',
1F15.8)
0241 IFG(29)=0
0242 9706 FRC(12)=-CON1*XMBC*G
FMG(1)=-XKMOUN*X(3) - XCMOUN*XD(3)
FMG(2)=- FB(2)
FMG(3)=- FB(3)
FMG(4)=- FBC(7)
C      COMPUTE FMG(5)
FMG(5)=XK5*(X(L11) - X(3) )      + XC5*(XO(L11) - XO(3))
1 - PREL
IF(T.GE.TFIRE + PBORE.OR.IRND.EQ.0) GO TO 7050
TNO=T - TFIRE
0243
0244
0245
0246
0247
0248
0249
0250
0251

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0252 FMG(6)=XINTP(TNO,FBORX,FBORY)
0253 IF(IHOR.EQ.1) GO TO 7051
0254 IROH=1
0255 PRINT 7053,T
0256 FORMAT(' MORE FRICTION IS ON AT T=',F15.8)
0257 GO TO 7051
0258 FMG(6)=0.000
0259 IF(IHOR.EQ.0) GO TO 7051
0260 IROH=0
0261 PRINT 7052,T
0262 FORMAT(' MORE FRICTION IS OFF AT T=',F15.8)
C
0263 COMPUTE FMG(7)
0264 IF(X(15).LT.0.000.OR.IHAM.EQ.0) GO TO 30
0265 FMG(7)=XKHG*RH*DSIN(X(15)) * XCHG*RH*XD(5)*DCOS(X(5))
0266 IF(IFG(10).EQ.1) GO TO 31
0267 IFG(10)=1
0268 PRINT 567,T
0269 FORMAT(' HAMMER STRIKE OF THE MAIN GUN IS ON AT T=',F15.8)
GO TO 31
0270 30 FMG(7)=0.000
0271 IF(IFG(10).EQ.0) GO TO 31
0272 IFG(10)=0
0273 PRINT 568,T
0274 FORMAT(' HAMMER STRIKE OF THE MAIN GUN IS OFF AT T=',F15.8)
C
0275 COMPUTE FMG(8)
0276 COMPUTE FMG(9)
0277 IF(DIFFS.LT.ALPI) GO TO 32
0278 FMG(9)=XK9*(-DIFFS+ALPI) - XC9*DIFD5
0279 IF(FMG(9).GT.0.000) FMG(9)=0.000
0280 IF(IFG(11).EQ.1) GO TO 33
0281 IFG(11)=1
0282 PRINT 569,T
0283 FORMAT(' BUFFER STRIKES HACK PLATE AT T=',F15.8)
0284 PRINT 6666,XD(1),XD(2),XD(3)
6666 FORMAT(/,' VELOCITIES PRIOR TO IMPACT',/,
1' RLTD=',F15.8,' RLTCO=',F15.8,' XMGO=',F15.8,/)
GO TO 33
0285 32 FMG( 9)=0.000
0286 IF(IFG(11).EQ.0) GO TO 33
0287 PRINT 570,T
0288 FORMAT(' BUFFER COMES OFF OF BACK PLATE AT T=',F15.8)
0289 PRINT 6667,XD(1),XD(2),XD(3)
0290 6667 FORMAT(/,' VELOCITIES AFTER IMPACT',/,
1' RLTD=',F15.8,' BLTCD=',F15.8,' XMGO=',F15.8,/)
IFG(11)=0
0291 C ** SUM THE FORCES ON THE BOLT, BOLT CARRIER, AND GUN.
0292 33 FMG(10)=-FB(11)
0293 FMG(11)=-CON1*XMG*G
0294 FMG(12)=-FBC(10)
0295 FMG(13)=-FBC(11)
0296 FMG(14)=-XMBUFF * XMWT)*G*CON2*DSIGN(1.00,DIFD5)*XMUFF - XCUFF
0297 1*DIFD5

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0298 FMG(15)=-FR(9)
0299 FRUF(1)=-FBC(6)
0300 FRUF(2)=-FMG(9)
0301 FRUF(3)=-XK5*(X(4) - X(N11)) - XC5*(XD(4) - XD(N11)) * PREL
0302 FRUF(4)=-CON1*(XBUFF - .0014)*G
0303 FRUF(6)=-FMG(14)
0304 IF(I40G.EQ.1) GO TO 7042
0305 FRUFF=XMT*XURUF*DSIGN(1.00,DIFD6)*G*CON2
0306 IF(DIFF6.GT.WTI) GO TO 40
0307 IF(DIFF6.LT.-SLACK*WTI) GO TO 41
0308 IF(IFG(16).EQ.0) GO TO 42
0309 PRINT 2222,T
0310 FORMAT(' FORWARD OR REARWARD INTERNAL SPRING BUFFER FORCE IS OFF A
      1 T=',F15.8)
0311 IFG(16)=0
0312 GO TO 42
0313 40 FRUFF=FRUFF - XBFORM*(DIFF6 - WTI) - XCFORM*DIFD6
0314 IF(IFG(16).EQ.1) GO TO 42
0315 IFG(16)=1
0316 PRINT 3333,T
0317 FORMAT(' FORWARD INTERNAL SPRING FORCE IS ON AT T=',F15.8)
0318 GO TO 42
0319 41 FRUFF=FRUFF - XBREAR*(DIFF6 + SLACK - WTI) - XCREAR*DIFD6
0320 IF(IFG(16).EQ.1) GO TO 42
0321 IFG(16)=1
0322 PRINT 4444, T
0323 FORMAT(' REAR INTERNAL BUFFER SPRING IS ON AT T=',F15.8)
0324 42 XDN(6)=(-G*XMT*CON1*FRUFF)/XMT
0325 GO TO 427
0326 7042 FRUFF=0.000
0327 DO 8 K1=1,5
0328 XDN(5 + K1)= - XUBUF*DSIGN(1.00,XD(5 + K1) - XD(4))*G*CON2
0329 FBUFF=XMS(K1)*XDN(5 + K1) + FRUFF
0330 8 XDN(5 + K1)=XDN(5 + K1) - CON1*G
0331 DO 8056 K1=1,4
0332 IF(X(6 + K1) - X(5 + K1).GT.0.000) GO TO 8057
0333 GO TO 8059
0334 8057 XDN(6 + K1)=XDN(6 + K1) - (XKS(K1+1)*(X(6 + K1) - X(5 + K1)) +
      1 XCS(K1 + 1))*(XD(6 + K1) - XD(5 + K1))/XMS(K1+1)
0335 XDN(5 + K1)=XDN(5 + K1) + (XKS(K1+1)*(X(6 + K1) - X(5 + K1)) +
      1 XCS(K1 + 1))*(XD(6 + K1) - XD(5 + K1))/XMS(K1)
0336 IF(IFG(22 + K1).EQ.1) GO TO 8056
0337 IFG(22 + K1)=1
0338 PRINT 8060,T,(IFG(I),I=22,27)
0339 FORMAT(' NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T=',F15.8,
      16I3)
0340 GO TO 8056
0341 8059 IF(IFG(22 + K1).EQ.0) GO TO 8056
0342 IFG(22 + K1)=0
0343 PRINT 8060,T,(IFG(I),I=22,27)
0344 8056 CONTINUE
0345 IF(DIFF6.GT.WTI) GO TO 8070
0346 IF(IFG(22).EQ.0) GO TO 8090

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0347 IFG(22)=0
0348 PRINT 8060,T,(IFG(I),I=22,27)
0349 GO TO 8090
0350 FRUFF=FRUFF - XKS(1)*(DIFF6 - WTI) - XCS(1)*DIFD6
0351 XDN(6)=XDD(6) - (XKS(1)*(DIFF6 - WTI) + XCS(1)*DIFD6)/XMS(1)
0352 IF(IFG(22).EQ.1) GO TO 8090
0353 IFG(22)=1
0354 PRINT 8060,T,(IFG(I),I=22,27)
0355 IF(X(10) - X(4)*LT.WTI - SLACK) GO TO 8091
0356 IF(IFG(27).EQ.0) GO TO 4227
0357 IFG(27)=0
0358 PRINT 8060,T,(IFG(I),I=22,27)
0359 GO TO 4227
0360
0361 FRUFF=FRUFF - (X(10) - X(4) - WTI + SLACK)*XKS(6) -
1 (X(10) - X(4))*XCS(6)
1 XDN(10)=XD(10) - ((X(10) - X(4) - WTI + SLACK)*XKS(6) +
1 (X(10) - X(4))*XCS(6))/XMS(5)
IF(IFG(27).EQ.1) GO TO 4227
IFG(27)=1
PRINT 8060,T,(IFG(I),I=22,27)
4227 SUM1=0.000
SUM2=0.000
SUM3=0.000
SUM4=0.000
SUM5=0.000
FRUF(5)=-FRUFF
XDN(L11)=( - FMG(5) - XKS*(X(L11) - X(L11 - 1))
1 - XCS*(X(L11) - X(L11 - 1))-PREL)/XMDSP
XDN(V11)=( - FBUF(3) - XKS*(X(N11) - X(N11 + 1))
1 - XCS*(X(N11) - X(N11 + 1))*PREL)/XMDSP
DO 1952 I=111,M,L11M
1952 XDN(I)=(XKS*(X(I + 1) - 2.*X(I) + X(I - 1))
1 + XCS*(X(I + 1) - 2.*X(I) + X(I - 1)))/XMDSP
DO 70 I=1,12
SUM1=SUM1 + FR(I)
SUM2=SUM2 + FBC(I)
SUM3=SUM3 + FMG(I)
DO 71 I=1,6
SUM4=SUM4 + FBUF(I)
SUM3=SUM3 + FMG(I) + FMG(I+5)
XDN(4)=SUM4/XMBUFF
IF(1HAM.EQ.1.AND.IFLAG.EQ.1) GO TO 800
GO TO 801
C *** FOLLOWING COMPUTES THE ACCELERATION OF THE HAMMER WHEN
C THE GUN IS ABOUT TO FIRE
800 XDN(5)=(TSPRIN - RH*FMG(7)*DCOS(X(5) + ALPHA))/XIM
IF(IFG(12).EQ.1) GO TO 804
PRINT 571,T
11 T=F15.0)
IFG(12)=1
GO TO 804
801 IF(1ISEP.EQ.0 .OR. IFIRE.EQ.0.OR. ITEST.EQ.0.OR. DIFD3.LT.0.000.
1OR. DIFF3.LT. 0.000) GO TO 803

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0391 IF(INIT.EQ.1) GO TO 804
0392 INIT=1
C **** AFTER THE WEAPON HAS FIRED, THE HAMMER IS GIVEN AN INITIAL VELOCITY
C DETERMINED BY THE CONSTRAINT THE BOLT CARRIER HAS PUT ON THE HAMMER
C (I.E. ROTATIONAL VELOCITY)
0393 SEM=(RHH0*DSIN(ALH0) - DIFF3)/(RHH0*DC0)
0394 XD(5)=(1./1. + SEM*SFM)*OIFD3/(RHH0*DC0)
0395 X(5)=-ALH0 + DATAN(SEM)
0396 IFG(12)=5
0397 PRINT 572,T
0398 FORMAT(' HAMMER BEGINS TO COCK AT T=',F15.8)
0399 GO TO 804
C **** THE ROTATIONAL VELOCITY AND ACCELERATION OF THE HAMMER ARE ZERO EXCEPT
C WHEN THE WEAPON IS ABOUT TO BE FIRED, AND WHEN IT IS COCKED
0400 IF(INIT.EQ.1.AND.IFIRE.EQ.1) GO TO 830
0401 SEM=(RHH0*DSIN(ALH0) - DIFF3)/(RHH0*DC0)
0402 IF((TEST.FO.1.AND.(X(5).LT.THETO.OR.(XD(5).GT.0.000.AND.
1 -ALH0 + DATAN(SEM)).LT.THEIK.AND.X(5).LT.
1 THETK)).OR.IFG(12).EQ.2) GO TO 8509
0403 IF(INIT.EQ.0) GO TO 8509
0404 IF(X(5).LT.-ALH0 + DATAN(SEM)) GO TO 5577
0405 FBC(8)=DCOS(X(5))*(X(5)+ ALH0 - DATAN(SEM))*HK + DCOS(X(5))*HC
1*( DIFD3/(1. + SEM*SFM)*RHH0*DC0) + XD(5)
RHHU=OSORT((RHH0*DS0 - DIFF3)*(RHH0*DS0 - DIFF3) + RHH0*RHH0*DC0
1*DCU)
0407 ALTEM=DARSIN(RHH0*DS0/RHHU)
0408 IF(IMD.EQ.1) GO TO 5555
0409 IMD=1
0410 PRINT 5578,T,FBC(8)
0411 FORMAT(' HAMMER FORCE IS ON AT T=',F15.8,' FBC(8)=',F15.8)
0412 GO TO 5555
0413 IMD=0
0414 XDD(5)=(TSPRIN- RHHU*FRC(8)*DCOS(X(5) +ALTEM))/XIM
0415 IF(IFG(12).EQ.1) GO TO 804
0416 PRINT 691,T
0417 FORMAT(' HAMMER CONSTRAINT FORCE IS OFF AT T=',F15.8 )
0418 IFG(12)=1
0419 GO TO 804
0420 XDD(5)=0.000
0421 XD(5)=0.000
0422 IF(IUNLO.EQ.1.AND.X(5).GT.THETO)X(5)=THETO
0423 IF(IFG(12).EQ.2) GO TO 804
0424 PRINT 573,T
0425 FORMAT(' HAMMER ROTATIONAL VELOCITY AND ACCELERATION ARE ZERO STAR
TING AT T=',F15.8)
IFG(12)=2
0426 IF(INIT.EQ.1.AND.IFIRE.EQ.1) GO TO 830
0427 SOHV=DCOS(X(5))*RO*XHHAM
0428 IF(DABS(XDD(5) + XD(5)).LT.1.D-10) SOMV=0.000
0429 XDD(3)=- (RO*(XDD(5)*DCOS(X(5)) - XD(5)*XD(5)*DSIN(X(5)))*
1XHHAM + FBC(8)*FMG(7)) + SUM3/(XHHAM + XMG - SOHV*SOMV/XIM)
0430 IF((DIFF2.GE..520830-2.AND.DIFF2.LE.XLD.AND.IBRC.EQ.0).AND.
1(IFG(13).EQ.1.OR.ITEST.EQ.1)) GO TO 879

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0432 IF (IFG(13).EQ.1.AND.ITFST.EQ.0) GO TO R79
C ***** IF THE GUN ISN'T BEING COCKED OR LOCKED UP THEN THE ACCELERATIONS OF
C THE BOLT AND BOLT CARRIER ARE COMPUTED HERE
0433 IF (IUNLO.EQ.0)XDD(3)=(XDD(3))*(XHAM + XMG -SOMV*SOMV/XIM) + SUM1//
1(XHAM + XMG + XMRV(IFLG) - SOMV*SOMV/XIM)
XDD(1)=SUM1/XMRV(IFLG)
XDD(2)=SUM2/XMBC
0434 IF (IUNLO.EQ.0)XDD(1)=XDD(3)
0435 IF (IFG(13).EQ.0) GO TO R75
0436 IF (IFG(13).EQ.0) GO TO R75
0437 PRINT 575,T
0438
0439 575 FORMAT(' WEAPON LOCKING FORCE IS OFF AT T=',F15.8)
0440
0441 IFR(13)=0
0442 XMOH1=(XMG + XMRV(IFLG)*XHAM)*XD(1) + XMBC*XD(2)
XKEN=(XMG+XMRV(IFLG)*XHAM)*XD(1)*XD(1)/2.+XMBC*XD(2)*XD(2)/2.+
1 XH*DFRIV*DERIV*(XD(2)-XD(1))*XD(2)-XD(1))/2.
SA=XMG+XMRV(IFLG) +XHAM
0443 SH=XMBC
0444 SC=(XMG+XMRV(IFLG)+XHAM)/2.
0445 SD=XMBC/2.
0446 SE=0.0DU
0447 PRINT 2200,XD(1),XD(2)
0448 XD(2)=POSRT(SA,SH,SC,SD,SE,XMOH1,XKEN,1)
0449 XD(1)=(XMOH1 - SH*XD(2))/SA
0450 XD(3)=XD(1)
0451 PRINT 2300,XD(1),XD(2)
0452 GO TO R75
0453
0454 C ***** IF BEING LOCKED UP BUT NOT BEING COCKED THEN THE FOLLOWING EQUATIONS
0455 C COMPUTE THE ACCELERATIONS OF THE BOLT AND BOLT CARRIER
0456 879 C1 = - XH*DFRIV*DERIV
IF (IUNLO.EQ.0) GO TO 1701
TEMP=(XMRV(IFLG) - C1)*(XMBC - C1) - C1*C1
XDD(1)=(SUM1*(XMBC - C1) - SUM2*C1) /TEMP
XDD(2)=(-SUM1*C1 + (XMRV(IFLG) - C1)*SUM2)/TEMP
GO TO 1700
0457
0458 1701 TEMP=(XMRV(IFLG) + XMG + XHAM - C1)*(XMBC - C1) - C1*C1
TEMP=TEMP - SOMV*SOMV*(XMBC - C1)/XIM
SUM1=SUM1 + XDD(3)*(XHAM + XMG - SOMV*SOMV/XIM)
XDD(1)=(SUM1*(XMBC - C1) - SUM2*C1) /TEMP
XDD(2)=(-SUM1*C1 + (XMRV(IFLG) + XMG + XHAM - C1 -SOMV*SOMV/XIM)
1*SUM2)/TEMP
XDD(3)=XDD(1)
0459
0460 C ***** CONSTRAINT FORCE ACTING ON THE BOLT DURING LOCKING
0461 1700 FB(7)= C1*(XDD(1) - XDD(2))
0462 FBC(5)=-FR(7)
0463 IF (IFG(13).EQ.1) GO TO R75
0464 PRINT 574,T
0465 574 FORMAT(' WEAPON LOCKING FORCE IS ON AT T=',F15.8)
IFG(13)=1
0466 XMOH1=(XMG + XMRV(IFLG)*XHAM)*XD(1) + XMBC*XD(2)
0467 IF (IUNLO.EQ.0)XMOH1=XMRV(IFLG)*XD(1) + XMBC*XD(2)
0468 XKEN=(XMG+XMRV(IFLG)*XHAM)*XD(1)*XD(1)/2.+XMBC*XD(2)*XD(2)/2.
0469 IF (IUNLO.EQ.0)XKEN=XMRV(IFLG)*XD(1)*XD(1)/2. + XMBC*XD(2)*XD(2)/2.
0470 SA=XMG+XMRV(IFLG) +XHAM
0471
0472
0473
0474
0475
0476

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0477 IF(IUNLO.EQ.1)SA=XMBV(TFLG)
0478 SB=XMRC
0479 SC=(AMG*XMRV(IFLG)+XMHAM)/2.
0480 IF(IUNLO.EQ.1)SC=XMBV(TFLG)/2.
0481 SD=XMRC/2.
0482 SE=XIR*DERIV*DERIV/2.
0483 PRINT 2200,XD(1),XD(2)
0484
2200 FORMAT(/, VELOCITIES OF THE BOLT AND ROLT CARRIER PRIOR TO THE DI
1SCONTINUITY OF THE CAM,/, BLTD=,F15.6, BLTCD=,F15.6)
XD(1)=(X*OM1 - SH*XD(2))/SA
IF(IUNLO.EQ.0) XD(3)=XD(1)
PRINT 2300,XD(1),XD(2)
2300 FORMAT(/, VELOCITIES OF THE BOLT AND ROLT CARRIER AFTER THE DISCO
INFINUITY OF THE CAM,/, BLTD=,F15.6, BLTCD=,F15.6)
GO TO A75
0490
C *** FROM THIS POINT TO $$$ , THE EQUATIONS ARE USED TO COMPUTE THE
C THE ACCELERATIONS OF THE ROLT AND THE ROLT CARRIER IF ONLY THE WEAPON IS
C BEING UNLOCKED
0491 030 RHHU=USORT((RHHU)*DS0 - DIFF3)*(RHHU)*DS0 - DIFF3) + RHHO*RMHO
1*DC0*DC0)
0492 ALTEM=DARSIN(RHHO*DS0/RHHU)
0493 TEMP1=RHHU*DCOS(X(5) *ALTEM)
0494 SEM=(RH+0*DS0 - DIFF3)/(RHHO*DC0)
0495 THFS=-1./((1. + SEM*SEM)*(RHHO*DC0))
0496 SEM1=(1. + SEM*SEM)*RHHO*DC0
0497 FDDP=-2.*SEM/(SEM1*SEM1)
0498 TEMP2=XTIME*THES/TEMP1
0499 SOMV=DCOS(X(5))*R0*XMHAM
0500 IF(UABS(XD(15) + XD(5)).LT.1.0-10) SOMV=0.000
0501 TT1=XMRC - TEMP2
0502 TT2=TEMP2 +SOMV/TEMP1
0503 TT3=SUM2 + TSPHIN/TEMP1 - FDDP*DIFF3*DIFF3*XIM/TEMP1
0504 TEMP3=R0*DCOS(X(5))
0505 V1=-XMHAM*TEMP3*THES + TEMP2
0506 V2=XMG + XMHAM*TEMP3*THES + XMHAM - TEMP2- SOMV*SOMV/XIM
0507 1 - SOMV/TEMP1
V3=SUM3 + XMHAM*XD(5)*XD(5)*DSIN(X(5))*R0 - TSPHIN/TEMP1 - FMG(7)
1 + XIM*DIFF3*DIFF3*FDDP/TEMP1 - XMHAM*R0*DCOS(X(5))*FDDP*DIFF3*
2DIFF3
0508 IF((DIFF2*GE.*52083D-2.AND.DIFF2*LE.XLD.AND.IBRC.EQ.0).AND.
1(IFG(14).EQ.1.OR.ITEST.EQ.1)) GO TO 831
0509 IF(IFG(14).EQ.1.AND.ITEST.EQ.0) GO TO 831
0510 IF(IUNLO.EQ.0) GO TO 1702
GO TO 1703
0512 DIVI=TT1*V2 - V1*TT2
0513 XGE =(TT1*V3 - V1*TT3)/DIVI
0514 V2=V2 + XMBV(IFLG)
0515 V3=V3 + SUM1
0516 ADD(1)=SUM1/XMBV(IFLG)
0517 XBE=XDD(1)
0518 DIVI=TT1*V2 - V1*TT2
0519 XDD(2)=(TT3 * V2 - TT2*V3)/DIVI

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0520 XDD(3)=(TT1*V3 - V1*TT3)/DIVI
0521 IF(IUNLO.EQ.0)XDD(1)=XDD(3)
0522 IF(IFG(14).EQ.0) GO TO 877
0523 IFG(14)=0
0524 PRINT 596,T
0525
0526 596 FORMAT(' WEAPON IS UNLOCKED AT T=',F15.8)
0527 XMM1=(XMG + XMBV(IFLG)+XMHAM)*XD(1) + XMBC*XD(2)
XKEN=(XMG+XMBV(IFLG)+XMHAM)*XD(1)/2.+XMBC*XD(2)*XD(2)/2.+
1 XIB*DERIV*DERIV*(XD(2)-XD(1))*XD(2)-XD(1))/2.
0528 SA=XMG+XMBV(IFLG) +XMHAM
SB=XMBC
SC=(XMG+XMBV(IFLG)+XMHAM)/2.
SD=XMBC/2.
SE=0.000
0530 PRINT 2200,XD(1),XD(2)
0531 XD(2)=PO:RT(SA,SB,SC,SD,SE,XMM1,XKEN,0)
0532 XD(1)=(XMM1 - SR*XDD(2))/SA
0533 XD(3)=XD(1)
0534 PRINT 2300,XD(1),XD(2)
0535
0536
0537
0538
0539
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C
GO TO 877
C
C THESE EQUATIONS ARE USED TO COMPUTE THE ACCELERATIONS OF THE BOLT
C AND BOLT CARRIER IF THE WEAPON IS UNLOCKING (TO 8666)
831 C1=-XIR*DERIV*DERIV
1705 CONX=1. - DSIGN(1.00,FCONX)*CONH
TT1=XMRC - C1 - TEMP2 -V1*(CONX - 1.)
TT2=C1 + TT2 -(CONX -1.)*(V2 + XMARC)
TT3=TT3 - V3*(CONX - 1.)
V1=V1*CONX + C1
V2=(V2 + XMARC)*CONX - XMARC + XMBV(IFLG) - C1
V3=V3*CONX + SUM1
DIVI=TT1*V2 - V1*TT2
XDD(2)=(TT3 * V2 - TT2*V3)/DIVI
XDD(3)=(TT1*V3 - V1*TT3)/DIVI
XDD(1)=XDD(3)
8666
C
C CONSTRAINT FORCE ACTING ON THE BOLT DURING UNLOCKING
1704 FCONX=(-C1*(XDD(1) - XDD(2)) + (XMBV(IFLG) - XMARC)*XDD(1)
1 - SUM1)/CONX
FB(6)=FCONX*(CONX - 1.) + C1*(XDD(1) - XDD(2))
FBC(3)=-FB(6)
IF(IFG(14).EQ.0) GO TO 877
IFG(14)=1
0595 PRINT 595,T
0596
0597 XMM1=(XMG + XMBV(IFLG)+XMHAM)*XD(1) + XMBC*XD(2)
0598 XKEN=(XMG+XMBV(IFLG)+XMHAM)*XD(1)/2.+XMBC*XD(2)*XD(2)/2.+
0599 SA=XMG+XMBV(IFLG) +XMHAM
0600 SB=XMBC
0601 SC=(XMG+XMBV(IFLG)+XMHAM)/2.
0602 SD=XMBC/2.
0603 SE=XIB*DERIV*DERIV/2.
0604 PRINT 2200,XD(1),XD(2)
0605

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0566 XD(2)=POSRT(SA,SR,SC,SD,SE,XNOM),XKEN*0
0567 XU(1)=(XMOM1 - SR*XD(2))/SA
0568 XU(3)=XD(1)
0569 PRINT 2300,XD(1),XD(2)
C ***** CONSTRAINT ACTING BETWEEN THE THE BOLT CARRIER AND GUN
C DURING COCKING OF THE HAMMER
0570 FBCL(R)=(-XIM*THES*(XDD(3) - XDD(2)) + TSPRIN + XIM*2.*DIFD3*DIFD3
1*SEM/(SEM1*SEM1) - 50*V*XDD(3))/(RHMU*DCOS(X(5) + ALTEM))
C ***** COMPUTED ACCELERATION OF THE HAMMER . THIS PROVIDES A CHECK ON THE
C CORRECT SOLVING OF THE SYSTEM OF ALGEBRAIC EQUATIONS NECESSARY TO GET THE
C ACCELERATIONS
0571 XD(5)=(TSPRIN - RHMU*FRC(R)*DCOS(X(5) +ALTEM))/XIM
C CONSTRAINT FORCE ACTING AT THE HAMMER PIVOT POINT
0572 XD(5)=XD(5) - 50*V*XDD(3)/XIM
0573 FMS(8)=(XHAM*RO*(XDD(5)*DCOS(X(5)) - XU(5)*XD(5)*DSIN(X(5))
1 + XDD(3)/RO) + FHC(8) + FMC(7))
0574 XD(L1)=(P20*FMC(1) - R10*(XDD(3)* AMG
1 - FMC(1)) - XKG* X(L1) - XCG*XDD(L1))/XIG
0575 IF(IUNLO.EQ.0.AND.IFLAG.NE.0.OR.IFLG.EQ.3) GO TO 1775
0576 1.GT.FSTIC)IUNLO=1
1775 IF(IUNLO.EQ.1) GO TO 951
0577 951 IF(IFG(15).EQ.0) RETURN
0578 IFG(15)=0
0579 PRINT 598,T
0580 598 FORMAT(' BOLT AND MAIN GUN ARE TWO MASSES AT T=,F15.8)
0581 TBRCT=
0582 RETURN
0583 950 IF(IFG(15).EQ.1) RETURN
0584 IFG(15)=1
0585 PRINT 597,T
0586 597 FORMAT(' BOLT AND MAIN GUN ARE CONSIDERED AS ONE MASS AT T=,
0587 1 F15.8)
0588 RETURN
0589 ENTRY READIT
0590 NAMELIST/MAG/NC,NI
0591 NAMELIST/MAG/ YE,YF,XK,MAG,XFL,XUMAG1,XUMAG2,XUMAG3
0592 1 ,XPROJ,SPR,EPSOL,DIAC
C XPROJ MUST BE LESS THAN XPUC1
NAMELIST/MANP/TBFO,XMOSP
0593 XMDSP=XMDSP/G
0594 PRINT 9560,TBFO,XMOSP
0595 9560 FORMAT(' TRFO,5X,DELAY BETWEEN HAMMER STRIKE AND IGNITION',
0596 120X,F20.8,2X,'SEC',// , XMDSP,4X,'MASS OF THE DRIVE SPRING',
0597 136X,F20.8,2X,'SLUGS',//)
0598 READ (5,MAG)
0599 READ (5,MAG1)
0600 PREL=XK5*BETA
0601 TEMP=NMASS + 1
0602 XK5=XK5*TEMP
0603 XMDSP=XMOSP/(TEMP - 1.)

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0604 PRINT '500*NC, YE, YF, NI, XKMAG
0605 9500 FORMAT(/, /, 10X, ' INPUT PARAMETERS FOR FFD MECHANISM', /, /,
11 NC=, I2, ' (NUMBER OF ROUNDS WHEN MAGAZINE IS FULL)', /, /,
21 YE=, F10.5, 'FT', ' (HT. OF MAG. SPRING WHEN FULL)', /, /,
31 YF=, F10.5, 'FT', ' (HT. OF MAG. SPRING WHEN EMPTY)', /, /,
41 NI=, I2, ' (INITIAL NUMBER OF ROUNDS IN THE MAGAZINE)', /, /,
51 XKMAG=, F10.5, 'LBS/FT', ' (MAGAZINE SPRING CONSTANT)', /, /,
PRINT 9501, XFL, XUMAG1, XUMAG2, XUMAG3, XPROJ, SPR, FPSOL, DIAC
9501 FORMAT(' XFL=, F10.5, 'FT', ' (FREE LENGTH OF MAGAZINE SPRING)', /, /,
11 XUMAG1=, F10.5, ' (COEFF. OF FRICTION BETWEEN R.C. AND ROUNDS)', /, /,
21 XUMAG2=, F10.5, ' (COEFF. OF FRICTION BETWEEN MAG. AND ROUNDS)', /, /,
31 XUMAG3=, F10.5, ' (COEFF. OF FRICTION BETWEEN ROUNDS)', /, /,
4 'F10.5, 'FT', ' (DISTANCE BOLT CARRIER MOVES WHEN FRICTION B
51 SPR=, F10.5, 'LBS', ' (A THIRD THE SPRING MASS AND THE FOLLOWER MA
55S)', /, /,
61 FPSOL=, F10.5, ' (EFFICIENCY OF MAGAZINE SPRING SYSTEM)', /, /,
71 DIAC=, F10.5, 'FT', ' (DIAM. OF CASE)', /, /,
LI=7 + NMASS
IF(I4DG.EQ.1) LI=11 + NMASS
LI1=LI - 1
NI1=7
IF(I4DG.EQ.1) NI1=11
LI1=LI1 - 1
NI1=NI1 + 1
TEMP=NC
DFLSP=(YE - YF)/TEMP
TEMPI=NI
XMF=TEMPI*(XMBUL + XMARC) + SPR/G
XMG=TEMPI*(XMRUL + XMARC) + XMG
YIHT=YE - DELSP*TEMPI
SINMI=DELSP/DIAC
COSMI=DSQRT(1.00 - SINMI*SINMI)
PRINT 9056, DELSP, YIHT, XMF
9056 FORMAT(/, /, 10X, ' COMPUTED PARAMETERS ASSOCIATED WITH THE MAG.', /, /,
11 DELSP=, F10.5, 'FT', ' (SPACE PER ROUND)', /, /,
21 YIHT=, F10.5, 'FT', ' (INITIAL HT. OF THE MAG. SPRING)', /, /,
11 XME=, F10.5, 'SLUGS', ' (EFFECTIVE MASS ASSOC. WITH THE MAGAZINE)',
1)
CONF1=DCOS(ALH1)
DS0=DSIN(ALH0)
DC0=DCOS(ALH0)
CONT=DCOS(ALPHA)
CONB=2.*XMUR*(R2B*R2B + R1B*R2B + R1B*R1B)*DERIV/
1((R1B + R2R)*3.)
RETURN
ENTRY READUM
READ 1718, XMS, XKS, XCS
1718 FORMAT(5F10.0, /, /, 6F10.0, /, /, 6F10.0)
PRINT 1719, XMS, XKS, XCS
1719 FORMAT(/, /, ' MASSES, SPRING CONSTANTS, AND DAMPING COEFS.',
1/, ' UNITS ARE SLUGS, LBS/FT, AND LBS-SEC/FT, RESPECTIVELY',
1/, ' THE FIRST MASS CORRESPONDS TO THE MASS NEAREST THE BOLT CARRIE

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0636 2R 1,/, XMS=1.5F15.8,/, XKS=1.6F10.2,/, XCS=1.6F10.2)
0637 NAMELIST/ACCUR/R10,R20,XIG,XKG,XCG
0638 READ(5,ACCUR)
0639 PRINT 1720,R10,R20,XIG,XKG,XCG
1720 FORMAT(/,15X,PARAMETERS FOR ACCURACY,/,/,
11 R10=1,F15.8,FT (BARREL OFFSET FROM THE C.G.),/,/,
12 R20=1,F15.8,FT (MOUNT FORCE OFFSET FROM THE C.G.),/,/,
13 XIG=1,F15.8,SLUG - FT**2 (MOMENT OF INERTIA OF THE GUN),/,/,
14 XKG=1,F15.8,FT-LBS/RAD (TORSIONAL SPRING CONSTANT),/,/,
15 XCG=1,F15.8,FT-LBS-SEC/RAD (DAMPING FOR TORSIONAL SPRING),/,
RETURN
END

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0001 SURROUTINE OUTP(T,N)
0002 IMPLICIT REAL*8(A-H,O-7)
0003 REAL*8 FBMA(12)/12*0.000/, FRCMA(12)/12*0.000/, FRCMA(15)/15*0.000/
0004 REAL*4 ITMIN,ITMAX,ITY
0005 DATA IOUIT,K/O,1/,IDUMR/O/,IAIM/O/,IDPR/O/,FMCI/O.000/,
0006 IFCA/O.000/
0007 COMMON/MAS/NMASS
0008 COMMON/LAIN/ITMIN,ITMAX,ITY,ITY
0009 COMMON/DIF/DIFF1,DIFF2,DIFF3,DIFF4,DIFF5,DIFF6
0010 COMMON/RUNKA/PRMT(4),X(21),XD(21),XDD(21)
0011 COMMON/MOSES/IMODE,IFG(30),IPT,IHAM,IUNLO,ISEP,IRBC,INIT,IRND
0012 COMMON/ASTHT/PJNCH ,PANCY(50),IFIRE,CCL,XLAMCC,XMUCC,ACC,ROREA,
0013 XKHYP,XCHYP,CF4,XMU4,XHR,XLD,PRRL,PCAL,CARTF(50),CARTX,PCAR
0014 COMMON/ASIT/XHHC,XHRC,XHRC,DERIV,XMRUL,XMARC ,PUC,AREBOL,
0015 TDEL,XIR,G,TPORT,XCHCG,XMURCG,EUCAR,FLEV ,XKLCB,XCLBC
0016 COMMON/NUH/NRCH,IFLAG,NCAY,IFLGF,IFIRF
0017 COMMON/ASTHEE/PCAVT ,PCAVY(50),FRORY(50),PBOHE,FRORX
0018 COMMON/ER/XKBUFF,XCBUFF,XMRC,XMBV(3),XIM,RH,ALPHA,ISPRIN,
0019 1 RHOD,ALHC,RHMI,ALMI,HK+HC,
0020 1 DCOCK ,XKMOUN,XCMOUN,XUHBC,XDMA,EXTC,EXTF,FDIS,FTIC
0021 1,R1B,P2R,XMUR,FCONX
0022 COMMON/S/YK9,XC9,XK5,XC5,BETA,ALP1,THETO,XKHG,XCHG,XMHAM,XMG,
0023 XMBUFF,CON1,XPUC,XPUC1,RO,CON2,XMUFF,XCUFF
0024 COMMON/RUFF/XHFORM,XCFORM,XHREAR,XCREAR,SLACK,XUBUF,XHWT,WTI
0025 COMMON/PRT/FH(12),FHC(12),FNG(15),FRUF(6)
0026 COMMON/GRAP/TGR1,TGR2
0027 COMMON/MSS/XMS(5),XKS(6),XCS(6)
0028 REAL*4 IMT(600),IRLT(600),IRLTD(600),IRLTC(600),IRLTC(600),IRLTC(600),
0029 1IBUF(600),IRHUF(600),IRFW(600),IRFWD(600),IGUN(600),IGUND(600),
0030 2IBFMA(600),IROT(600),IROTD(600),IDRR(600),IDRG(600)
0031 REAL*4 IRLTD(600),ICDR(600),IGDD(600),IBDD(600),IHDD(600),
0032 1IWD(600),TAXI(600),FRK(600),FB7(600),FBC8(600)
0033 COMMON/PLCP/MTX
0034 INTEGER*2 MTX(51,111)
0035 INTEGER*2 CHR,CHRI,CHS(5)/1,2,3,4,5//
0036 INTEGER XCORD(4)/TIME/, (SE, 'C) , , /, YCORD(4) NAME(4)
0037 REAL*4 BUFGI,RI,FMA,GUNMI,GUNMA,GUNDMI,GUNDMA,BFMMI,BFWMA,
0038 1ROTHI,ROTHA,ROIDMI,ROIDMA,FHTMI,FHTMA,RLTMI,BLTMA,
0039 2BLTMI,RLTMA,HLDDMI,RLCDMA
0040 3BUDDMI,HDCMI,HDDMA,WDDMI,WDDMA,FENMI,FENMA
0041 IF (IMAX.EQ.0) GO TO 5000
0042 DO 5001 I=1,12
0043 FBMA(I)=DMAX1(FBMA(I),DABS(FR(I)))
0044 5001 FBCMA(I)=DMAX1(FBCMA(I),DABS(FBC(I)))
0045 DO 5002 I=1,15
0046 5002 FMGMA(I)=DMAX1(FMGMA(I),DABS(FMG(I)))
0047 DO 5003 I=1,6
0048 5003 FBUFMA(I)=DMAX1(FBUFMA(I),DABS(FBUF(I)))
0049 FMCI=DMIN1(FCONX,FMCI)
0050 FMCA=DMAX1(FCONX,FMCA)
0051 5000 IF (IRND.EQ.1.AND.T.GE.TFIRE + TMUZ.AND.IAIM.EQ.0)

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0039      1 GO TO 4995
0040
0041 4996 IF (IRND.GT.1.AND.IRND.IRND.GT.IDPR) GO TO 4899
0042      GO TO 4897
0043 4899 CALL ACCUM
0044      IDPR=IRND
0045      PRINT 1000,I,X,XD,XDD
0046      PRINT 1001,FR,FBC,FMG,FRUF,XMBV(IFLG),XMG
0047      IF (IPT.NE.-1) GO TO 9270
0048      PRINT 1000,I,X,XD,XDD
0049      PRINT 1001,FR,FBC,FMG,FRUF,XMBV(IFLG),XMG
0050      REAL% IFH(12),EFB(12),IFBC(12),EFBC(12),IFMG(15),
0051      IEFMG(15),IFHUF(6),EFBUF(6)
0052      COMMON/ACHY/IFB,EFB,IFRC,EFBC,IFMG,EFMG,IFRUF,EFBUF
0053      DO 7880 I=1,12
0054      IFR(I)=FR(I)*XD(1)/2.
0055      EFR(I)=FRC(I)/2.
0056      DO 7881 I=1,15
0057      IFMG(I)=FMG(I)/2.
0058      EFMG(I)=FMG(I)*XD(3)/2.
0059      DO 7882 I=1,6
0060      IFRUF(I)=FRUF(I)/2.
0061      EFRUF(I)=FRUF(I)*XD(4)/2.
0062      DO 7883 I=1,12
0063      IFR(I)=IFR(I) + FB(I)
0064      EFR(I)=EFR(I) + FR(I)*XD(1)
0065      IFRC(I)=IFRC(I) + FBC(I)
0066      EFRC(I) = EFRC(I) + FRC(I)*XD(2)
0067      DO 7884 I=1,15
0068      IFMG(I)=IFMG(I) + FMG(I)
0069      EFMG(I)=EFMG(I) + FMG(I)*XD(3)
0070      DO 7885 I=1,6
0071      IFRUF(I)=IFRUF(I) + FRUF(I)
0072      EFRUF(I)=EFRUF(I) + FRUF(I)*XD(4)
0073      IFDIFF3.GT.ALPI.AND.INUMB.EQ.0) GO TO 7886
0074      GO TO 9200
0075      IDUMB=1
0076      CALL ACCUM
0077      IPT=IPT + 1
0078      IF (IMDRE.EQ.1) GO TO 1600
0079      IF (MOD(IPT,ISR).NE.0) GO TO 1600
0080      IDPB(K)=FRUF(3)
0081      IDRG(K)=FMG(5)
0082      IBUF(K)=X(4) - X(3)
0083      IGUN(K)=X(3)
0084      IBOUND(K)=XD(3)
0085      IF (I4DG.EQ.1) GO TO 2000
0086      IBFWX(K)=X(6) - X(3)
0087      IBFMD(K)=XD(6)
0088      GO TO 2001
0089      2000 IBFWX(K)=(X(6)*XMS(1) + X(7)*XMS(2) + X(8)*XMS(3) +
      1X(9)*XMS(4) + X(10)*XMS(5))/XNWT - X(3)

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0140 2' FMG=,7D12.5,8013.5,/,
0141 3' FBUF=,6D12.5,/, XMR=,E12.5,/, XMG=,E12.5,/,
0142 C //////////////// PRINT FORCES
0143 99 IF(T - TFIRE.GT.,11000) GO TO 100
0144 IF(1PICK.EQ.0.AND.T - TFIRE.GT.,0600) GO TO 201
RETURN
100 PRINT 101,T
101 FORMAT(' TIME IS EQUAL TO ',F15.8,/, '//////////WEAPON HAS FAILED T
GO TO 200
201 PRINT 203
203 FORMAT(' //////////// WEAPON HAS FAILED TO RECOIL FAR ENOUGH TO PICK.
1UP ANOTHER ROUND ////////////)
200 IQUIT=1
PRINT 1000,T,X,XD,XDD
PRINT 1001,FR,FBC,FMG,FBUF,XMRV(IFLG),XMG
GO TO 1500
50 PRINT 1000,T,X,XD,XDD
PRINT 1001,FR,FBC,FMG,FBUF,XMRV(IFLG),XMG
51 FORMAT(' PROGRAM TERMINATED SINCE ROUND REQUEST IS MET')
1500 PRINT 150
150 FORMAT(/,/, TERM,20X,DEFINITION,15X,UNITS,/,/,
1' XMG,6X,GUN DISPLACEMENT, 24X,FT,/,/
2' XMGD,5X,VELOCITY OF THE GUN,21X,FT/SEC,/,/
3' RLTD,6X,BOLT DISPLACEMENT,23X,FT,/,/
4' RLTD,5X,BOLT VELOCITY,27X,FT/SEC,/,/
5' RLTC,5X,BOLT CARRIER DISPLACEMENT,15X,FT,/,/
6' RLTC,4X,BOLT CARRIER VELOCITY,19X,FT/SEC,/,/
PRINT 151
151 FORMAT(' RUFF,5X,RUFFER DISPLACEMENT,21X,FT,/,/
1' RUFFD, 4X,RUFFER VELOCITY,25X,FT/SEC,/,/
2' RUFFW,5X,INTERNAL RUFFER WT. DISPLACEMENT,8X,FT,/,/
3' RUFFWD,4X,INTERNAL RUFFER WT. VELOCITY,12X,FT/SEC,/,/
4' HAMR,5X,ROTATION OF THE HAMMER,18X,TRAD,/,/
5' HAMRD,4X,ROTATIONAL VELOCITY OF THE HAMMER,7X,RAD/SEC,/,/
PRINT 152
152 FORMAT(' NOTE:80LT, BOLT CARRIER, AND RUFFER DISPLACEMENTS ARE WIT
1H RESPECT TO THE GUN AND NOT THE INERTIAL REFERENCE FRAME')
IF(IMAX.EQ.0) GO TO 5010
PRINT 5011,FBMA,FBCMA,FMGMA,FBUFMA
5011 FORMAT(/,/, ABSOLUTE MAXIMUMS OF INDIVIDUAL FORCES,
1',, MAX FB=,6E12.5,/,6E12.5,/, MAX FBC=,6E12.5,/,6E12.5,/,
1' MAX FMG=,7E12.5,/,8E12.5,/, MAX FBUF=,6E12.5,/,
PRINT 9076,FMCI,FMCA
9076 FORMAT(' FMCI=,E12.5, FMCA=,E12.5)
5010 K=K - 1
IUM8=0
4995 IAIM=1
TEMP=DSIN(X(L1)) * PHIE)
TEMP1=DCOS(X(L1)) * PHIE)
ZI=SGRI*TEMP
ZDI=XD(L1)*SGRI*TEMP1 + DSIN(X(L1))*VPR
YI=SGRI*TEMP1
YDI=XD(L1)*SGRI*TEMP + DCOS(X(L1))*VPR

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0175 TEMP=RANGE - YI
0176 ZCAP=-.5*G*TEMP*TEMP/
      1(YDI*YDI) + ZDI*TEMP/YDI + ZI
0177 PRINT 2988,ZCAP,X(L1),XD(L1)
0178 FORMAT(' ZCAP=',F15.8,FT (DEFLECTION OF PROJECTILE),/,
      1 X(L1)=',F15.8,RAD (GUN ROTATION),/,
      2 X(D(L1))=',F15.8,RAD/SEC (ROTATIONAL VELOCITY OF THE GUN)'),
0179 IF(IRND.EQ.1.AND.IQUIT.NE.1) GO TO 4996
0180 IF(IMORE.EQ.1) CALL EXIT
0181 IF(IWSV.NE.1) GO TO 2043
0182 PRINT 2045,(TAXI(I),FR6(I),FR7(I),FR8(I),I=1,K)
0183 FORMAT(' CONSTRAINT FORCES FR(6) FR(7) FR(8)',/(4E10.3,2X,4E10.3
      1,/)
0184 IF(IPRI.NE.1) GO TO 1920
0185 PRINT 1917,(TAXI(I),IRLTC(I),I=1,K)
0186 FOPMAT(//,40X,'BOLT CARRIER DISPLACEMENT',/(10F12.5//))
0187 PRINT 1918,(TAXI(I),IBLTCO(I),I=1,K)
0188 FOPMAT(//,40X,'BOLT CARRIER VELOCITY',/(10F12.5//))
0189 PRINT 1920 IF(I4DG.NE.1) GO TO 2073
0190 CALL PLOT2(ITMIN,ITMAX,RR*MI,RR*MA,XCORD,16HDISP. (FT)
      1 16HRUFN--RUFF
      1CHR,CHR1)
0191 CALL PLOT0(TAXI
      1ITMIN,ITMAX,0.0,0.0,XCORD,YCORD,NAME,CHR,
      1MT,0.0,0.0,16H
      10,CHR,K)
0192 CALL PLOT1(IMT,1H,FMTMI,FMTMA)
0193 CALL PLOT2(ITMIN,ITMAX,FMTMI,FMTMA,XCORD,16HFORCE (LBS)
      1 16HMOUNT FORCE
      10.0,0.0,1,16H
      1CHR1)
0194 CALL PLOT0(TAXI
      1ITMIN,ITMAX,0.0,0.0,XCORD,YCORD,NAME,CHR,
      1MT,0.0,0.0,16H
      10,CHR,K)
      C CALL PLOT1(IBLT,1H,BLTI,BLTMA)
      C CALL PLOT1(IBLTC,1H,BLTCI,BLTMA)
      C CALL PLOT2(ITMIN,ITMAX,BLTI,BLTMA,XCORD,16HDISP. (FT)
      1 16HBLT-- BLTC--
      10.0,0.0,16H
      1CHR1)
0195 CALL PLOT1(IDRB,1H,FENMI,FENMA)
0196 CALL PLOT1(IDRG,1H,FENMI,FENMA)
0197 CALL PLOT2(ITMIN,ITMAX,FENMI,FENMA,XCORD,16HFORCE (LBS)
      1 16HFREF-- FMG--
      10.0,0.0,16H
      2,1,CHR,CHR1)
0198 CALL PLOT0(TAXI
      1ITMIN,ITMAX,0.0,0.0,XCORD,YCORD,NAME,CHR,
      1MT,0.0,0.0,16H
      10,CHR,K)
0199 CALL PLOT1(IBLTD,1H,BLTDMI,BLTDMA)
0200 CALL PLOT1(IBLTI,1H,BLTI,BLTMA)
0201 CALL PLOT2(ITMIN,ITMAX,BLTDMI,BLTDMA,XCORD,16HVELOC. (FT/SEC)
      1 16HBLTD-- BLTI--
      10.0,0.0,16H
      1 2,CHR,CHR1)
0202 CALL PLOT0(TAXI
      1ITMIN,ITMAX,0.0,0.0,XCORD,YCORD,NAME,CHR,
      1MT,0.0,0.0,16H
      10,CHR,K)
0203 CALL PLOT1(IBLTCO,1H,BLTCOMI,BLTCOMA)
0204 CALL PLOT1(IBLTCI,1H,BLTCI,BLTMA)
0205 CALL PLOT2(ITMIN,ITMAX,BLTCOMI,BLTCOMA,XCORD,16HVELOC. (FT/SEC)
      1 16HBLTCO-- BLTCI--
      10.0,0.0,16H
      1 2,CHR,CHR1)

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0206 CALL PLOT0(TAXI ,ITMIN,ITMAX,0.0,0.0,XCORD,YCORD,NAME,CHR,
      1,IT,0.0,0.0,0.16H
0207 CALL PLOT1(18UF0,1H*,8UFDM1,RUFDMA
      CALL PLOT1(18UF,1H*,8UFMI,8UFMA)
0209 CALL PLOT2(ITMIN,ITMAX,8UFDM1,RUFDMA,XCORD,16HVELOC. (FT/SEC)
      1 16HRUFFC-- BUFF--
      2,*,*,*)
      ,8UFMI,8UFMA,16HDISP. (FT) (*),
0210 CALL PLOT0(TAXI ,ITMIN,ITMAX,0.0,0.0,XCORD,YCORD,NAME,CHR,
      1,IT,0.0,0.0,0.16H
0211 CALL PLOT1(1BF,0.1H*,8FWDMI,RFWDMA)
0212 CALL PLOT1(1BF,*,1H*,8FMI,8FMA)
0213 CALL PLOT2(ITMIN,ITMAX,8FWDMI,RFWDMA,XCORD,16HVELOC. (FT/SEC)
      1 16HRUFWC-- RUFW--
      2,*,*,*)
      ,8FMI,8FMA,16HDISP. (FT) (*),
0214 CALL PLOT0(TAXI ,ITMIN,ITMAX,0.0,0.0,XCORD,YCORD,NAME,CHR,
      1,IT,0.0,0.0,0.16H
0215 CALL PLOT1(IGUN,*,*,GUNMI,GUNMA)
0216 CALL PLOT1(IGUND,*,*,GUNDMI,GUNDMA)
0217 CALL PLOT2(ITMIN,ITMAX,GUNDMI,GUNDMA,XCORD,16HVELOC. (FT/SEC)
      116HXMGD-- XMG--
      2 2,*,*,*)
      ,GUNMI,GUNMA,16HDISP. (FT) (*),
0218 CALL PLOT0(TAXI ,ITMIN,ITMAX,0.0,0.0,XCORD,YCORD,NAME,CHR,
      1,IT,0.0,0.0,0.16H
0219 CALL PLOT1(IKOT,*,*,ROTKI,ROTKA)
0220 CALL PLOT1(IKOTD,*,*,ROTKDI,ROTKDA)
0221 CALL PLOT2(ITMIN,ITMAX,ROTKDI,ROTKDA,XCORD,16HHAMRD (RAD/SEC)
      1 16HHAMRD-- HAMR--
      2 2,*,*,*)
      ,ROTKI,ROTKA,16HHAMR (RAD) (*),
0222 IF (14DG.EQ.1) GO TO 2700
0223 CALL PLOT0(TAXI ,ITMIN,ITMAX,0.0,0.0,XCORD,YCORD,NAME,CHR,
      1,IT,0.0,0.0,0.16H
0224 CALL PLOT1(1BF,1H*,88WMI,88WMA)
0225 CALL PLOT2(ITMIN,ITMAX,88WMI,88WMA,XCORD,16HDISP. (FT)
      1 16HRUFW--RUFF
      1,CHP,CHR1)
      ,0.0,0.0,0.16H
      ,1,
2700 IF (1OPT.NE.1) GO TO 1845
0226 CALL PLOT0(TAXI ,ITMIN,ITMAX,0.0,0.0,XCORD,YCORD,NAME,CHR,
      1,IT,0.0,0.0,0.16H
0227 CALL MINMAX(18LTD0,8LDDMI,8LDDMA,K)
0228 CALL PLOT1(18LTD,*,*,8LDDMI,8LDDMA)
0229 CALL PLOT2(ITMIN,ITMAX,8LDDMI,8LDDMA,XCORD,16HACC. (FT/SEC**2)
      116HBLTD0--
      1,CHP,CHR1)
      ,0.0,0.0,0.16H
0231 CALL PLOT0(TAXI ,ITMIN,ITMAX,0.0,0.0,XCORD,YCORD,NAME,CHR,
      1,IT,0.0,0.0,0.16H
0232 CALL MINMAX(1CDD ,CADDMI,CADDMA,K)
0233 CALL PLOT1(1CDD ,*,*,CADDMI,CADDMA)
0234 CALL PLOT2(ITMIN,ITMAX,CADDMI,CADDMA,XCORD,16HACC. (FT/SEC**2)
      116HBLTCD0--
      1,CHP,CHR1)
      ,0.0,0.0,0.16H
0235 CALL PLOT0(TAXI ,ITMIN,ITMAX,0.0,0.0,XCORD,YCORD,NAME,CHR,
      1,IT,0.0,0.0,0.16H
0236 CALL MINMAX(1GDD ,GDDMI,GDDMA,K)
0237 CALL PLOT1(1GDD ,*,*,GDDMI,GDDMA)
0238 CALL PLOT2(ITMIN,ITMAX,GDDMI,GDDMA,XCORD,16HACC. (FT/SEC**2)
      116HXMGDD--
      ,0.0,0.0,0.16H
      ,1,CHP,CHR1)

```

```

0239 CALL PLOT0(TAXI ,ITMIN,ITMAX,0.0,0.0,XCORD,YCORD,NAME,CHR,
      LIMT,0.0,0.0,16H ,0,CHR,K)
0240 CALL MINMAX(IRDD ,RUDDMI,RUDDMA,K)
0241 CALL PLOT1(IRDD,*,*,HDDMI,BUDDMA)
0242 CALL PLOT2(ITMIN,ITMAX,RUDDMI,RUDDMA,XCORD,16HACC. (FT/SEC**2) ,
      116HBUFFDD--e ,0.0,0.0,16H ,1,CHR,CHRI)
0243 CALL PLOT0(TAXI ,ITMIN,ITMAX,0.0,0.0,XCORD,YCORD,NAME,CHR,
      LIMT,0.0,0.0,16H ,0,CHR,K)
0244 CALL MINMAX(IRDD ,HDDMI ,HDDMA ,K)
0245 CALL PLOT1(IRDD,*,*,HDDMI,HDDMA)
0246 CALL PLOT2(ITMIN,ITMAX,HDDMI ,HDDMA ,XCORD,16HACC. (FT/SEC**2) ,
      116HHAUPDD--e ,0.0,0.0,16H ,1,CHR,CHRI)
0247 CALL PLOT0(TAXI ,ITMIN,ITMAX,0.0,0.0,XCORD,YCORD,NAME,CHR,
      LIMT,0.0,0.0,16H ,0,CHR,K)
0248 CALL MINMAX(IRDD ,WDDMI ,WDDMA ,K)
0249 CALL PLOT1(IRDD,*,*,WDDMI,WDDMA)
0250 CALL PLOT2(ITMIN,ITMAX,WDDMI ,WDDMA ,XCORD,16HACC. (FT/SEC**2) ,
      116HBUFWDD--e ,0.0,0.0,16H ,1,CHR,CHRI)
1865 PRINT 18
18 FORMAT('11')
0252 IF (IRND.LE.NROUND) GO TO 700
0253 PRINT 51
0254
0255 CALL EXIT
0256
0257 K=1
0258 ITMIN=1
0259 ITMAX=ITMIN + TGR2
0260 IF (IQUIT.EQ.1) CALL EXIT
0261 IF (I4DG.NE.1) GO TO 2074
0262 CALL PLOT0(TAXI ,ITMIN,ITMAX,0.0,0.0,XCORD,YCORD,NAME,CHR,
      LIMT,0.0,0.0,16H ,0,CHR,K)
0263
0264 ENTRY READIN
C BUFGI,RUFMA EST. MAX. AND MIN. VALUES FOR BUFFER DISPLACEMENT
C GUNMI,GUNMA EST. MAX. AND MIN. VALUES FOR GUN DISPLACEMENT
C GUNDMI,GUNDMA EST. MAX. AND MIN. VALUES FOR GUN VELOCITIES
C BFWMI,RFWMA EST. MAX AND MIN. VALUES FOR BUFFER WEIGHT DISPL.
C (WITH RESPECT TO THE GUN)
C ROTMI,ROTHA MAX. AND MIN. VALUES OF ROTATION
C ROTDMI,ROTDMA , MAX. AND MIN. VALUES OF ROTATIONAL VELOCITY
C FMTMI,FMTMA MAX. AND MIN. VALUES OF THE MOUNT FORCE
C RLTM1,RLTMA MAX. AND MIN. VALUES OF THE ROLT DISPL.
C RLCDMI,RLCDMA MAX. AND MIN. VALUES OF THE HOLT CARRIER VELOCITY
C BUFDMI,HUFDMA MAX. AND MIN. VALUES OF THE BUFFER VELOCITY
C BRWMI,BRWMA MAX. AND MIN. VALUES OF THE RUFFER WT. WITH RESPECT TO
C THE RUFFER
C BFWDMI,RFWOMA MAX. AND MIN. VALUES OF THE RUFFER WT. VELOCITY
C FENMI,FENMA EST. MAX AND MIN VALUES FOR DRIVE SPRING CONSTANT
0265 NAMELIST/INPT6/BUFGI,RFWMA,GUNMI,GUNMA,GUNDMI,GUNDMA,BFWMI,BFWMA,
      1ROTM1,ROTHA,ROTDMI,ROTDMA,FMTMI,FMTMA,RLTM1,RLTMA,
      2BLTDMI,BLTOMA,RLCDMI,RLCDMA,
      2BRWDMI,BFWOMA,FENMI,FENMA
      .BUFDMI,BUFDMA,BBWMI,BBWMA,
      READ(5,INPT6)
0266
0267 LI=7 +NMAS

```



```

0001 SURROUTINE RUNGE(N,FCT,OUTP)
0002 IMPLICIT REAL*8(A-H,O-7)
C SOLVES A SYSTEM OF SECOND ORDER DIFF EQS.
C PRMT(1) INITIAL TIME
C PRMT(2) FINAL TIME
C PRMT(3) TIME INCREMENT
C PRMT(4) STOPS INTEGRATING IF THIS TERM IS NEGATIVE
C ITEST ITEST IS ONE THEN A NEW TIME INCREMENT IS STARTING
C AND IF ITEST IS ZERO THEN IN THE MIDDLE OF A TIME INCREMENT
C X,DX CONTAINS INITIAL VALUES BEFORE COMING INTO THE ROUTINE
C DXD SECOND DERIVATIVE TERM
C FCT SUBROUTINE TO EVALUATE THE SECOND DERIVATIVES
C OUTP SUBROUTINE TO PRINT
COMMON/RUNKA/PRMT(4),X(21),XD(21),XDD(21)
DIMENSION AK1(21),AK2(21),AK3(21),AK4(21)
DIMENSION TX(21),TXD(21)
C TIME INCREMENT CAN BE CHANGED BY JUST CHANGING DELT IN ROUTINE
C FCT AND COMPUTING THE NEW VALUE FOR SUM
COMMON/CNTRL/DELT,SUM
FINAL=PRMT(2)
DELT=PRMT(3)
T=PRMT(1)
PRMT(4)=1.0D0
SUM=DELT*.500
1000 CALL FCT(T,X,DX,XDD,1)
CALL OUTP(T,N)
IF(T.GE.FINAL.OR.PRMT(4).LT.0.0D0) RETURN
DO 1 I=1,N
AK1(I)=DELT*XDD(I)
TX(I)=X(I)+.500*DELT*XD(I)+DELT*AK1(I)*.125D0
1 TXD(I)=XD(I)+.500*AK1(I)
T=T+SUM
CALL FCT(T,TX,DX,XDD,0)
DO 2 I=1,N
AK2(I)=DELT*XDD(I)
2 TXD(I)=XD(I)+.500*AK2(I)
CALL FCT(T,TX,DX,XDD,0)
DO 3 I=1,N
AK3(I)=DELT*XDD(I)
TX(I)=X(I)+DELT*XD(I)+DELT*AK3(I)*.500
3 XD(I)=XD(I)+AK3(I)
T=T+SUM
CALL FCT(T,TX,DX,XDD,0)
DO 4 I=1,N
AK4(I)=DELT*XDD(I)
X(I)=X(I)+DELT*XD(I)+.16666666666666667D0*(AK1(I)
1+AK2(I)+AK3(I))
4 XD(I)=XD(I)+(AK1(I)+AK2(I)+2.0D0*(AK3(I)+AK4(I))
1*.16666666666666667D0
GO TO 1000
END

```

```
0001 SUBROUTINE DETER(X,Y,XMIN,XMAX,YMIN,YMAX,I,J)
0002 I=(X - XMIN)*110./(XMAX - XMIN) + 1.5
0003 IF(I.LT.1)I=1
0004 IF(I.GT.111)I=111
0005 J=(Y - YMIN)*50./(YMAX - YMIN) + 1.5
0006 IF(J.LT.1)J=1
0007 IF(J.GT.51)J=51
0008 RETURN
0009 END
```

```
0001 FUNCTION XINTP(A,X,Y)
0002 IMPLICIT REAL*8(A-H,O-Z)
0003 DIMENSION Y(50)
      C ASSUMES EQUALLY SPACED POINTS
      C X-COORDINATE STARTS AT ZERO WITH STEP SIZE OF X
0004 I=A/X
0005 XI=I
0006 I=I + 1
0007 XL = XI*X
0008 XINTP=Y(I) + (Y(I + 1) - Y(I))*(A - XL )/X
0009 RETURN
0010 END
```

```
0001 FUNCTION POSRT(SA*SR*SC*SD*SE*XMOMI*XKEN,IITE)
0002 IMPLICIT REAL*8(A-H,O-Z)
0003 CA=SC*SH*SR/(SA*SA)*SE*(SB/SA + 1.)*(SR/SA*1.)*SD
0004 CB=-2.*XMOMI*(SC*SH/(SA*SA) + SE*(SR/SA*1.)/SA)
0005 CC=-XKEN + XMOMI*XMOMI*(SC*SE)/(SA*SA)
0006 IF(IITE.EQ.0)
0007 IPOSRT=(-CB-DSQRT(CB*CB - 4.*CA*CC))/(2.*CA)
0008 IF(IITE.EQ.1)
0009 IPOSRT=(-CB+DSQRT(CB*CB - 4.*CA*CC))/(2.*CA)
0010 RETURN
0011 END
```

```
0001 SUBROUTINE DETER(X,Y,XMIN,XMAX,YMIN,YMAX,I,J)
0002 I=(X - XMIN)*110./(XMAX - XMIN) + 1.5
0003 IF(I.LT.1)I=1
0004 IF(I.GT.111)I=111
0005 J=(Y - YMIN)*50./(YMAX - YMIN) + 1.5
0006 IF(J.LT.1)J=1
0007 IF(J.GT.51)J=51
0008 RETURN
0009 END
```

```

0001 .SURROUTINE ACCUM
0002 IMPLICIT REAL*8(A-H,O-Z)
0003 REAL*4 IFR(12),EFR(12),FRC(12),EFC(12),IFMG(15),
0004 IEFMG(15),IFBUF(6),EFRUF(6)
0005 COMMON/ACKY/IFH,EFR,FRC,IFMG,EFC,IFMG,EFMG,IFBUF,EFRUF
0006 COMMON/RUNKA/PRMT(4),X(11),XD(11),XDO(11)
0007 COMMON/PHT/FR(12),FRC(12),FMG(15),FBUF(6)
0008 DIMENSION SUM(15)
0009 DO 1 I=1,12
0010 1 SUM(I)=(IFR(I) - FR(I)/2.)*PRMT(3)
0011 PRINT 2,(SUM(I),I=1,12)
0012 2 FORMAT(' ROLI IMPULSES',6F15.8,/,6F15.8)
0013 DO 3 I=1,12
0014 3 SUM(I)=(EFR(I) - FR(I)*XD(1)/2.)*PRMT(3)
0015 PRINT 4,(SUM(I),I=1,12)
0016 4 FORMAT(' ROLI ENERGY',6F15.8,/,6F15.8)
0017 DO 5 I=1,12
0018 5 SUM(I)=(IFRC(I) - FRC(I)/2.)*PRMT(3)
0019 PRINT 6,(SUM(I),I=1,12)
0020 6 FORMAT(' ROLI CARRIER IMPULSES',6F15.8,/,6F15.8)
0021 DO 7 I=1,12
0022 7 SUM(I)=(FRC(I) - XD(2)*FRC(I)/2.)*PRMT(3)
0023 PRINT 8,(SUM(I),I=1,12)
0024 8 FORMAT(' ROLI CARRIER ENERGY',6F15.8,/,6F15.8)
0025 DO 9 I=1,15
0026 9 SUM(I)=(IFMG(I) - FMG(I)/2.)*PRMT(3)
0027 PRINT 10,(SUM(I),I=1,15)
0028 10 FORMAT(' GUN IMPULSE',7F14.8,/,8F15.8)
0029 DO 11 I=1,15
0030 11 SUM(I)=(EFMG(I) - FMG(I)*XD(3)/2.)*PRMT(3)
0031 PRINT 12,(SUM(I),I=1,15)
0032 12 FORMAT(' GUN ENERGY',7F14.8,/,8F15.8)
0033 DO 13 I=1,6
0034 13 SUM(I)=(IFRUF(I) - FBUF(I)/2.)*PRMT(3)
0035 PRINT 14,(SUM(I),I=1,6)
0036 14 FORMAT(' BUFFER IMPULSF',6F15.8)
0037 DO 15 I=1,6
0038 15 SUM(I)=(EFRUF(I) - XD(4)*FBUF(I)/2.)*PRMT(3)
0039 PRINT 16,(SUM(I),I=1,6)
0040 16 FORMAT(' BUFFER ENERGY',6F15.8)
0041 RETURN
0042 END

```

```

0001 SUBROUTINE PLOT0(K ,XMIN,XMAX,YMIN,YMAX,XCORD,YCORD,
0002 1 NAME,CHR,ISTORE,ZMIN,ZMAX,ZCORD,ITEST,CHR1,NPTS)
0003 COMMON/PL0P/MTX
0004 INTEGER*2 MTX(51,11),CHR,B/' ',CHR1
0005 INTEGER XCORD(4),YCORD(4),NAME(4),ZCORD(4)
0006 REAL*4 ISTORE(600),X(600)
0007 DO 1 I=1,51
0008 DO 1 J=1,11
0009 1 MTX(I,J)=B
0010 RETURN
0011 C //////////////////////////////////////////////////
0012 C ENTRY PLOT1(ISTORE,CHR,YMIN,YMAX)
0013 C //////////////////////////////////////////////////
0014 DO 1772 K=1,NPTS
0015 I=(X(K) - XMIN)*110./(XMAX - XMIN) + 1.5
0016 IF(I.LT.1)I=1
0017 IF(I.GT.11)I=11
0018 J=(ISTORE(K) - YMIN)*50./(YMAX - YMIN) + 1.5
0019 IF(J.LT.1)J=1
0020 IF(J.GT.51)J=51
0021 1772 MTX(J,I)=CHR
0022 RETURN
0023 C //////////////////////////////////////////////////
0024 C ENTRY PLOT2(XMIN,XMAX,YMIN,YMAX,XCORD,YCORD,NAME,ZMIN,ZMAX,ZCORD,
0025 1 ITEST,CHR,CHR1)
0026 C //////////////////////////////////////////////////
0027 DIMENSION XVAR(12)
0028 TEMP=ALOG10(YMAX-YMIN)
0029 IF((TEMP.LT.0.0)TEMP=TEMP-1.
0030 IFY=TEMP
0031 TEMP=ALOG10(XMAX-XMIN)
0032 IF((TEMP.LT.0.0)TEMP=TEMP-1.
0033 IFZ=TEMP
0034 DELZ=(ZMAX-ZMIN)/10.**((IFZ*1)
0035 Z1=ZMAX/10.**IFZ
0036 35 IF(ITEST.NE.2) GO TO 36
0037 PRINT 5,CHR,CHR1,(NAME(I),I=1,4)
0038 5 FORMAT('11/' ,CHR('A1'),, CHR('A1'),,40X,4A4)
0039 GO TO 38
0040 36 PRINT 47,(NAME(I),I=1,4)
0041 47 FORMAT('11/' ,55X,4A4)
0042 38 DO 13 L=1,11
0043 NV=56-5*L
0044 N1=N0-26
0045 IF(ITEST.NE.2) GO TO 27
0046 PRINT 28,Y1,Z1
0047 28 FOR:AT(' ,F6.2' ,,,F6.2) ,,,F6.2)
0048 Z1=Z1-DELZ

```

```

0049      GO TO 26
0050      27 PRINT 6,Y1
0051      6 FORMAT(' ',9X,F6.2)
0052      26 Y1=Y1-DEL7
0053      PRINT 7
0054      7 FORMAT(' ',16X,11(' |-----'),11)
0055      PRINT 8,(MTX(N0,J),J=1,1111)
0056      8 FORMAT(' ',16X,11A1)
0057      IF(L.EQ.11) GO TO 13
0058      DO 12 K=1,4
0059      PRINT 9
0060      9 FORMAT(' ',16X,11(' |',11)
0061      IF(11TEST.EQ.2) GO TO 30
0062      IF(N0-K.EQ.24) PRINT 10,(YCORD(I),I=1,4)
0063      10 FORMAT(' ',4A4)
0064      IF(N0-K.EQ.23) PRINT 11,IFY
0065      11 FORMAT(' ',SCALE 10**I3)
0066      GO TO 12
0067      30 IF(N0-K.EQ.29)PRINT 10,YCORD
0068      IF(N0-K.EQ.28)PRINT 11,IFY
0069      IF(N0-K.EQ.24)PRINT 10,ZCORD
0070      IF(N0-K.EQ.23)PRINT 11,IFZ
0071      12 PRINT 8,(MTX(N0-K,J),J=1,111)
0072      13 CONTINUE
0073      PRINT 14
0074      14 FORMAT(' ',16X,11(' .....'),11)
0075      XVAR(1)=XMIN/10.**IFX
0076      DELY=(XMAX - XMIN)/(11.*10.**IFX)
0077      DO 19 I=2,12
0078      19 XVAR(I)=XVAR(I-1)*DELY
0079      PRINT 15,XVAR
0080      15 FORMAT(' ',13X,F6.2,11(4X,F6.2))
0081      PRINT 16,(XCORD(I),I=1,4)
0082      16 FORMAT('0',55X,4A4)
0083      PRINT 17,IFX
0084      17 FORMAT(' ',55X,' SCALED BY 10**I3)
0085      RETURN
0086      ENTRY MINMAX(ISTORE,ZMIN,ZMAX,NPTS)
0087      ZMIN=ISTORE(1)
0088      ZMAX=ISTORE(1)
0089      DO 50 I=2,NPTS
0090      ZMIN=AMIN1(ZMIN,ISTORE(I))
0091      ZMAX=AMAX1(ZMAX,ISTORE(I))
0092      RETURN
0093      END

```


XIR	MOMENT OF INERTIA OF THE BOLT	0.00000105	SLUGS-FT**2
XMARC	MASS OF THE CARTRIDGE CASE	0.00041078	SLUGS
TPORT	TIME IT TAKES PROJECTILE TO REACH PORT	0.00110000	SEC
XKRUFF	INTERACTION SPRING BETWEEN BOLT CARRIER AND BUFFER	100000.00000000	LBS/FT
XCRUFF	DAMPING COEFFICIENT FOR ABOVE SPRING	14.06000000	LRS-SEC/FT
XCBCMG	DAMPING BETWEEN BOLT CARRIER AND MAIN GUN	0.01000000	LRS-SEC/FT
XIM	MOMENT OF INERTIA OF THE HAMMER	0.00002000	SLUGS-FT**2
RH	RADIAL DISTANCE OF THE HAMMER TO POINT OF CONTACT	0.11083000	FT
ALPHA	REFERENCE ANGLE ON THE HAMMER	0.30530000	RAD
XKMOUN	MOUNT SPRING CONSTANT	300.00000000	LBS/FT
XCMOUN	DAMPING FOR ABOVE SPRING	9.53100000	LRS-SEC/FT
TSPRIN	CONSTANT TORQUE DUE TO THE HAMMER SPRING	0.50000000	FT-LBS
XMURCG	COEFFICIENT OF FRICTION BETWEEN THE BOLT CARRIER AND GUN	0.20000000	NONE
XMBC	MASS OF THE BOLT CARRIER	0.01820807	SLUGS
XK9	BACK PLATE SPRING CONSTANT	60000.00000000	LRS/FT
X09	DAMPING FOR THE ABOVE SPRING	10.78000000	LBS-SEC/FT
XK5	DRIVE SPRING CONSTANT	19.20000000	LRS/FT
XC5	DAMPING FOR ABOVE SPRING	0.00010000	LRS-SEC/FT
BETA	PRELOAD DISTANCE FOR THE DRIVE SPRING	0.33700000	FT
ALP1	DISTANCE BUFFER TRAVELS BEFORE BACK PLATE IS HIT	0.31700000	FT
THETO	INITIAL ANGLE OF THE HAMMER	-1.22173000	RAD
XKHG	SPRING BETWEEN HAMMER AND GUN	50000.00000000	LRS/FT
XCHG	DAMPING FOR THE ABOVE SPRING	0.27800000	LBS-SEC/FT
XMHAM	MASS OF THE HAMMER	0.00212112	SLUGS
XMG	MASS OF THE MAIN GUN	0.17670807	SLUGS
XMBUFF	MASS OF THE BUFFER	0.00306460	SLUGS
EJCAR	DISTANCE BOLT TRAVELS BEFORE CARTRIDGE EJECTION	0.17400000	FT
ELEV	ANGLE OF ELEVATION OF THE GUN	-0.00110000	RAD
XPUC	DISTANCE OF BOLT WHEN ROUND IS PICKED UP	0.23441670	FT
XPUC1	DISTANCE OF BOLT WHEN PICK UP FORCE IS OFF	0.15316670	FT

RO	DISTANCE FROM THE PIVOT POINT TO THE C.G. OF THE HAMMER	0.06250000	FT
XBFORM	EFFECTIVE INTERNAL BUFFER SPRING CONSTANT (FORWARD)	22567.00000000	LBS/FT
XCFORM	DAMPING COEF. FOR THE ABOVE SPRING	6.99000000	LBS-SEC/FT
XBREAR	EFFECTIVE INTERNAL BUFFER SPRING CONSTANT (REARWARD)	26405.00000000	LBS/FT
XCREAR	DAMPING COEF. FOR THE ABOVE SPRING	7.56000000	LBS-SEC/FT
SLACK	DISTANCE WTS. ARE ALLOWED TO MOVE	0.01083000	FT
XURUF	COEF. OF FRICTION BETWEEN WTS. AND BUFFER	0.20000000	NONE
XMT	MASS OF THE WTS IN THE BUFFER	0.00695497	SLUGS
XUHRC	COEF. OF FRICTION BETWEEN HAMMER AND B.C.	0.15000000	NONE
XDHA	DISTANCE B.C. MOVES BEFORE FRICT. BETWEEN THE HAM. AND B.C.	0.11500000	FT
EXTC	EXTRACTOR DISTANCE	0.00780000	FT
EXTF	EXTRACTOR FORCE	7.50000000	LBS
FDIS	DISTANCE BOLT CARRIER IS FROM BATTERY, HAMMER ROTATES	0.01966700	FT
XMUFF	COEF. OF FRIC. BETWEEN GUN AND BUFFER	0.20000000	NONE
XCUFF	DAMPING BETWEEN BUFFER AND GUN	0.01000000	NONE
FSTIC	STICTION FORCE	0.0	LBS
RHH0	DISTANCE FROM HAMMER PIVOT TO B.C.	0.00783000	FT
RHH1	DISTANCE FROM PIVOT TO TOP OF HAMMER	0.13200000	FT
ALH0	ANGLE CORRESPONDING TO RHH0	0.38400000	RAD
ALH1	ANGLE CORRESPONDING TO RHH1	0.26200000	RAD
HK	HAMMER IMPACT SPRING	4500.00000000	LBS/RAD
HC	ABOVE DAMPING CONSTANT	0.08600000	LBS-SEC/RAD
R1R	INNER RADIUS CARTRIDGE CASE	0.00781250	FT
R2R	OUTER RADIUS CARTRIDGE CASE	0.01562500	FT
CCL	CARTRIDGE CASE LENGTH	0.14580000	FT
XMUR	COEFFICIENT OF FRICTION AT THE BASE	0.20000000	NONE
TBFO	DELAY BETWEEN HAMMER STRIKE AND IGNITION	0.0	SEC
XMDSP	MASS OF THE DRIVE SPRING	0.00414907	SLUGS

INPUT PARAMETERS FOR FEED MECHANISM

NC=19 (NUMBER OF ROUNDS WHEN MAGAZINE IS FULL)

YE= 0.37500FT (HT. OF MAG. SPRING WHEN EMPTY)
 YF= 0.08850FT (HT. OF MAG. SPRING WHEN FULL)
 NI=19 (INITIAL NUMBER OF ROUNDS IN THE MAGAZINE)
 XKMAG= 10.00000LBS/FT (MAGAZINE SPRING CONSTANT)
 XFL= 0.64700FT (FREE LENGTH OF MAGAZINE SPRING)
 XUMAG1= 0.24200 (COEF. OF FRICTION BETWEEN B.C. AND ROUNDS)
 XUMAG2= 0.30200 (COEF. OF FRICTION BETWEEN MAG. AND ROUNDS)
 XUMAG3= 0.30200 (COEF. OF FRICTION BETWEEN ROUNDS)
 XPROJ= 0.19496FT (DISTANCE BOLT CARRIER MOVES WHEN FRICTION BETWEEN B.C. AND ROUNDS IS OFF)
 SPR= 0.03000LBS (A THIRD THE SPRING MASS AND THE FOLLOWER MASS)
 EPSOL= 1.00000 (EFFICIENCY OF MAGAZINE SPRING SYSTEM)
 DIAC= 0.03125FT (DIAM. OF CASE)

COMPUTED PARAMETERS ASSOCIATED WITH THE MAG.

DELSP= 0.01508FT (SPACE PER ROUND)
 YIHT= 0.08450FT (INITIAL HT. OF THE MAG. SPRING)
 XME= 0.01589SLUGS (EFFECTIVE MASS ASSOC. WITH THE MAGAZINE)

MASSSES, SPRING CONSTANTS, AND DAMPING COEFS.

UNITS ARE SLUGS, LBS/FT, AND LBS-SEC/FT, RESPECTIVELY
 THE FIRST MASS CORRESPONDS TO THE MASS NEAREST THE BOLT CARRIER
 XMS= 0.00126136 0.00126136 0.00126136 0.00126136
 XKS= 60000.00 60000.00 60000.00 60000.00 60000.00 60000.00
 XCS= 12.00 12.00 12.00 12.00 12.00 12.00

0.00191190

PARAMETERS FOR ACCURACY

R10= 0.06500000FT (BARREL OFFSET FROM THE C.G.)
 R20= -0.02830000FT (MOUNT FORCE OFFSET FROM THE C.G.)
 XIG= 0.13520000SLUG - FT*2 (MOMENT OF INERTIA OF THE GUN)
 XKG= 50.00000000FT-LBS/RAD (TORSIONAL SPRING CONSTANT)
 XCG= 2.82800000FT-LBS-SEC/RAD (DAMPING FOR TORSIONAL SPRING)
 XLR= 1.64500000FT (PIVOT POINT TO BARREL END)
 XALPH= 0.06500000FT (BARREL OFFSET)
 RANGE= 450.00000000FT (TARGET RANGE)
 VPR= 3180.00000000FT/SEC (MUZZLE VELOCITY)
 TMUZ= 0.00130000SEC (TIME TO REACH MUZZLE)

COMPUTED RESULTS FOR A 1 SHOT BURST

(FORWARD) BOLT PIN STRIKES END OF CAM PATH AT T= 0.0 FBC(10)= -2.67172608
 FRICTIONAL FORCE BETWEEN CARTRIDGES AND B.C. IS ON AT T= 0.0
 HAMMER IS CONTROLLED BY HAMMER SPRING AND MAIN GUN STOP AT T= 0.0
 BOLT AND MAIN GUN ARE CONSIDERED AS ONE MASS AT T= 0.0
 0.124470-03 0.129420-03 0.129420-03 0.129420-03 0.129420-03 0.129420-03 0.129420-03 0.129420-03 0.129420-03 0.129420-03
 X= 0.0 0.6470-04 0.0 0.129420-03 0.129420-03 0.129420-03 0.129420-03 0.129420-03 0.129420-03 0.129420-03
 XD= 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0
 XDN= 0.69480 0.14030 0.0 0.69480 0.1 0.35590 0.2 0.24980 0.5-0.64050 0.1-0.64050 0.1-0.64050 0.1-0.64050 0.1-0.64050 0.1-0.64050 0.1-0.64050 0.1-0.64050 0.1-0.64050 0.1-0.64050 0.1-0.64050 0.1-0.64050
 FB= 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0
 FBC=-0.310910-01 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0
 FMG= 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0
 FRUFE=0.64700 0.1 0.0 0.64700 0.1 0.589600-04 0.448050-01 0.645260-01 0.0
 XMR= 0.482770-02 XMG= 0.191670 0.0

NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.00015000 1 1 0 0 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.00017500 1 1 0 0 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.00020000 0 1 0 0 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.00020000 0 1 1 0 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.00020000 0 1 1 0 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.00022500 1 1 1 0 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.00025000 1 1 1 0 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.00025000 0 1 1 0 1 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.00025000 0 1 1 1 1 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.00025000 0 1 1 1 1 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.00035000 0 1 1 1 1 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.00035000 0 1 1 1 1 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.00052500 1 1 1 1 1 0 0
 ***** ROUND NUMBER 1 BEGINS *****
 WEAPON FIRES AT T= 0.00990000
 BREECH FORCE IS ON AT T= 0.00990000
 ROPE FRICTION IS ON AT T= 0.00990000
 HAMMER STRIKE OF THE MAIN GUN IS ON AT T= 0.00990000
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01012500 0 1 1 1 1 0
 HAMMER STRIKE OF THE MAIN GUN IS OFF AT T= 0.01017500
 HAMMER ROTATIONAL VELOCITY AND ACCELERATION ARE ZERO STARTING AT T= 0.01017500
 (FORWARD) FORCE AT END OF CAM PATH IS OFF AT T= 0.01025000
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01040000 0 0 1 1 1 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01042500 0 0 0 1 1 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01045000 0 0 0 0 1 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01045000 0 0 0 0 0 0
 (FORWARD) BOLT PIN STRIKES END OF CAM PATH AT T= 0.01050000
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01090000 1 0 0 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01092500 1 1 0 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01095000 1 1 1 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01100000 1 1 1 1 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01105000 1 1 1 1 1 0
 CAVITY FORCE IS ON AT T= 0.0112500
 ZCAP= -0.04125794FT (DEFLECTION OF PROJECTILE)
 X(LL)= 0.00026430RAD (GUN ROTATION)
 XD(LL)= 0.40899012RAD/SEC (ROTATIONAL VELOCITY OF THE GUN)
 BREECH FORCE IS OFF AT T= 0.01132500

BORE FRICTION IS OFF AT T= 0.01132500
(FORWARD) FORCE AT END OF CAM PATH IS OFF AT T= 0.01175000
HAMMER BEGINS TO COCK AT T= 0.01175000
WEAPON BEGINS TO UNLOCK AT T= 0.01225000

VELOCITIES OF THE BOLT AND ROLT CARRIER PRIOR TO THE DISCONTINUITY OF THE CAM
BLTD= -2.367407 BLTCD= -18.026211

VELOCITIES OF THE BOLT AND ROLT CARRIER AFTER THE DISCONTINUITY OF THE CAM
BLTD= -2.392839 BLTCD= -17.749314
WEAPON IS UNLOCKED AT T= 0.01305000

VELOCITIES OF THE BOLT AND BOLT CARRIER PRIOR TO THE DISCONTINUITY OF THE CAM
BLTD= -1.126611 BLTCD= -22.884965

VELOCITIES OF THE BOLT AND ROLT CARRIER AFTER THE DISCONTINUITY OF THE CAM
BLTD= -1.090576 BLTCD= -23.277296

SPRING FORCE BETWEEN THE ROLT AND ROLT CARRIER IS ON AT T= 0.01327500
BOLT AND MAIN GUN ARE TWO MASSES AT T= 0.01327500
EXPERIMENTAL CARTRIDGE CASE EJECTION FORCE IS ON AT T= 0.01327500
HAMMER CONSTRAINT FORCE IS OFF AT T= 0.01327500
NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01380000 0 1 1 1 1 0
NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01382500 0 0 1 1 1 0
NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01382500 0 0 1 1 1 0
NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01385000 0 0 0 1 1 0
NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01385000 0 0 0 1 1 0
NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01385000 0 0 0 1 1 0
SPRING FORCE BETWEEN ROLT AND ROLT CARRIER IS OFF AT T= 0.01422500
SPRING FORCE BETWEEN THE BOLT AND ROLT CARRIER IS ON AT T= 0.01447500
NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01467500 0 0 0 0 1 1
NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01467500 0 0 0 0 1 1
NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01467500 0 0 0 0 1 1
NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01515000 0 0 0 1 1 1
CAVITY FORCE IS OFF AT T= 0.01570000
NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01592500 0 0 0 0 1 1
SPRING FORCE BETWEEN ROLT AND ROLT CARRIER IS OFF AT T= 0.01605000
NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01635000 0 0 0 1 1 1
NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01635000 0 0 0 1 1 1
NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01637500 0 0 1 1 1 1
NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.01847500 0 1 1 1 1 1
HAMMER ROTATIONAL VELOCITY AND ACCELERATION ARE ZERO STARTING AT T= 0.01855000
FRICTION FORCE COMES ON BETWEEN HAMMER AND P.C. AT T= 0.01857500 FRC(11)= 1.20600619
EXPERIMENTAL CARTRIDGE CASE EJECTION FORCE IS OFF AT T= 0.02175000
AMMO AND ROLT FRICTION IS ON AT T= 0.02175000
FRICTIONAL FORCE BETWEEN CARTRIDGES AND B.C. IS OFF AT T= 0.02340000
CAPTRIDGE CASE IS EJECTED AT T= 0.02342500
SPRING FORCE BETWEEN THE BOLT AND ROLT CARRIER IS ON AT T= 0.02382500
SPRING FORCE BETWEEN BOLT AND ROLT CARRIER IS OFF AT T= 0.02435000
SPRING FORCE BETWEEN BOLT AND ROLT CARRIER IS OFF AT T= 0.02435000
BOLT AND AMMO FRICTION IS OFF AT T= 0.02750000
SPRING FORCE BETWEEN THE ROLT AND ROLT CARRIER IS ON AT T= 0.02797500
SPRING FORCE BETWEEN ROLT AND ROLT CARRIER IS OFF AT T= 0.02860000
BUFFER STRIKES BACK PLATE AT T= 0.03042500

VELOCITIES PRIOR TO IMPACT
BLTD= -16.52286533 BLTCD= -15.73988025 XMGD= -0.90620382

ROLT IMPULSES -1.23365625 0.01095406 0.0 0.00026084 -0.18698827 -0.02008823
0.0 0.18188831 0.0 0.80843809 0.00395857 0.00000497
BOLT ENERGY 2.58875864 -0.17248485 0.0 -0.00507367 2.63359922 0.03631125
0.0 -0.42042500 0.0 -3.61488438 -0.07243737 -0.00004847
BOLT CARRIER IMPULSES -0.00026084 0.18698827 0.02008823 -0.18188831 0.0
0.00455814 0.05452869 -0.80843809 0.00447669 0.01540618 0.00002025
BOLT CARRIER ENERGY -0.00191320 -3.66714375 -0.40927129 0.28863145 0.0
-0.10436242 -0.96319688 13.15499375 -0.55793066 -0.25632532 -0.00022124

0.0 -0.42455156 0.0 -3.60525625 -0.07255833 -0.00004904 0.43797752
 BOLT CARRIER IMPULSES -0.00029817 0.19920697 0.02006953 -0.19589825 0.0
 0.00460198 0.05433560 -0.080843809 0.00461028 0.01570649 0.00002044
 BOLT CARRIER ENERGY -0.00145280 -3.69377188 -0.40796426 0.29269648 0.0
 -0.103969A9 -0.95755605 13.12821250 -0.55841211 -0.25689453 -0.000221A9
 GUN IMPULSE 0.351063A5 -0.0110H765 0.0 -0.00460198 -0.2865875 0.21A249A0
 -0.10524895 -0.00344598 -0.00398173 0.00021519 -0.00461028 -0.01570649 -0.0034864A 0.0
 GUN ENERGY -0.51334131 0.01080588 0.0 -0.00017917 0.03154416 0.0156346A 0.00441456 0.0
 0.091A3917 0.00620876 0.00403501 -0.00001797 0.000001A7 0.051490H1 0.0039864A
 RUFFER IMPULSE -0.43797752 0.0034459A 0.34055798 0.00000202A -0.6A7H0433 -0.04263077
 RUFFER ENERGY 5.54423899 -0.08666709 -4.46477598 -0.03172500
 SPRING FORCE BETWEEN THE HOLT AND HOLT CARRIER IS ON AT T= 0.03622500 0 0 0 0 1
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.03377500 0 0 0 0 1 1
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.03392500 0 0 0 0 1 1
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.03392500 0 0 0 0 1 1
 SPRING FORCE BETWEEN BOLT AND BOLT CARRIER IS OFF AT T= 0.03395000
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.03480500 0 0 1 1 1 1
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.03415000 0 0 1 1 1 1
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.03507500 0 0 1 1 1 1
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.03507500 0 0 1 1 1 1
 BUFFER COMES OFF OF BACK PLATE AT T= 0.03512500

VELOCITIES AFTER IMPACT
 RLTD= 6.71591603 BLTCD# 5.81941633 XMGD# -3.35274859
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.03512500 0 0 0 0 1 1
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.03515000 0 0 0 0 1 1
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.03627500 0 0 0 0 1 1
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.03747500 0 0 0 0 1 1
 SPRING FORCE BETWEEN THE BOLT AND BOLT CARRIER IS ON AT T= 0.03920000
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.03942500 0 0 1 1 1 1
 SPRING FORCE BETWEEN BOLT AND BOLT CARRIER IS OFF AT T= 0.03982500
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.04032500 0 1 1 1 1 1
 FORCE TO PICK UP NEW ROUND IS ON AT T= 0.04045000

INELASTIC COLLISION OF THE BOLT AND ROUND
 MASS OF THE GUN AND VELOCITIES OF THE BOLT AND ROUND
 PRIOR TO ROUND BEING PICKED UP
 GUN MASS= 0.19166786 BOLT VELOCITY= 5.4006587 ROUND VELOCITY (VELOCITY OF THE GUN)= -2.64627849

MASS OF THE GUN AND VELOCITIES OF THE BOLT AND ROUND
 AFTER ROUND IS PICKED UP
 GUN MASS= 0.19088050 BOLT VELOCITY= 4.27179363 ROUND VELOCITY = 4.27179363
 FRICTIONAL FORCE TO PICK UP NEW ROUND IS -6.90975269
 SPRING FORCE BETWEEN THE BOLT AND BOLT CARRIER IS ON AT T= 0.04232500
 SPRING FORCE BETWEEN THE BOLT AND BOLT CARRIER IS OFF AT T= 0.04310000
 SPRING FORCE BETWEEN THE BOLT AND BOLT CARRIER IS ON AT T= 0.04425000
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.04967500 0 1 1 1 1 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.04992500 0 1 1 1 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.04995000 0 1 1 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.04997500 0 1 0 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05000000 0 0 0 0 0 0
 SPRING FORCE BETWEEN HOLT AND BOLT CARRIER IS OFF AT T= 0.05065000
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05070000 1 0 0 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05075000 1 1 0 0 0 0
 SPRING FORCE BETWEEN THE HOLT AND HOLT CARRIER IS ON AT T= 0.05085000
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05085000 1 1 1 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05100000 1 1 1 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05115000 1 1 1 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05247500 1 0 1 1 1 0

NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05247500 1 0 0 1 1 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05247500 1 0 0 0 1 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05247500 1 0 0 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05247500 0 0 0 0 0 0
 FORCE TO PICK UP NEW ROUND IS OFF AT T= 0.05435000 0 0 0 0 0 1

RESPONSE TIME OF THE MAGAZINE IS 0.00893390 FMI= 5.86894737 FMA= 6.03180000
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05435000 0 0 0 0 1 1
 SPRING FORCE BETWEEN BOLT AND BOLT CARRIER IS OFF AT T= 0.05470000
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05487500 0 0 0 1 1 1
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05585000 0 0 1 1 1 1
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05745000 0 1 1 1 1 1
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05880000 0 1 1 1 1 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05885000 0 1 1 0 1 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05885000 0 1 1 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05887500 0 0 1 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.05887500 0 0 1 0 0 0
 FRICTION FORCE BETWEEN HAMMER AND H.C. IS OFF AT T= 0.06150000
 SPRING FORCE BETWEEN THE BOLT AND BOLT CARRIER IS ON AT T= 0.06217500

FRICTIONAL FORCE BETWEEN BOLT AND BOLT CARRIER IS OFF AT T= 0.06300000
 SPRING FORCE BETWEEN THE BOLT AND BOLT CARRIER IS ON AT T= 0.06330000 FBC(10)= -2.59290541
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.06405000
 SPRING FORCE BETWEEN BOLT AND BOLT CARRIER IS OFF AT T= 0.07007500 0 0 0 0 1
 SPRING EXTRACTOR FORCE IS ON AT T= 0.07062500 CONSTANT FORCE -7.50000000
 SPRING FORCE BETWEEN THE BOLT AND BOLT CARRIER IS ON AT T= 0.07082500
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.07097500 0 0 0 0 0
 SPRING FORCE BETWEEN BOLT AND BOLT CARRIER IS OFF AT T= 0.07147500
 WEAPON LOCKING FORCE IS ON AT T= 0.07150000

VELOCITIES OF THE BOLT AND BOLT CARRIER PRIOR TO THE DISCONTINUITY OF THE CAM
 BLTD= R.322588 BLTCD= 9.911152

VELOCITIES OF THE BOLT AND BOLT CARRIER AFTER THE DISCONTINUITY OF THE CAM
 BLTD= R.419253 BLTCD= 9.885522
 HAMMER IS CONTROLLED BY HAMMER SPRING AND MAIN GUN STOP AT T= 0.07157500
 SPRING BETWEEN BOLT AND BARREL IS OFF AT T= 0.07175000
 BOLT AND MAIN GUN ARE CONSIDERED AS ONE MASS AT T= 0.07175000
 WEAPON LOCKING FORCE IS OFF AT T= 0.07340000

VELOCITIES OF THE BOLT AND BOLT CARRIER PRIOR TO THE DISCONTINUITY OF THE CAM
 BLTD= 0.856913 BLTCD= 9.685520

VELOCITIES OF THE BOLT AND BOLT CARRIER AFTER THE DISCONTINUITY OF THE CAM
 BLTD= 0.842262 BLTCD= 9.844710
 (FORWARD) BOLT PIN STRIKES END OF CAM PATH AT T= 0.07395000
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.07437500 1 0 0 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.07460000 1 1 0 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.07490000 1 1 1 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.07535000 1 1 1 1 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.07545000 1 1 1 1 0 0
 (FORWARD) FORCE AT END OF CAM PATH IS OFF AT T= 0.07590000
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.07660000 1 1 1 1 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.07665000 1 1 1 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.07667500 1 1 0 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.07670000 1 0 0 0 0 0
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.07675000 0 0 0 0 0 0
 (FORWARD) BOLT PIN STRIKES END OF CAM PATH AT T= 0.07822500
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.07875000 0 0 0 0 0 1
 NEW STATUS OF COLLISIONS OF BUFFER WTS. AT T= 0.07917500 0 0 0 0 1 1

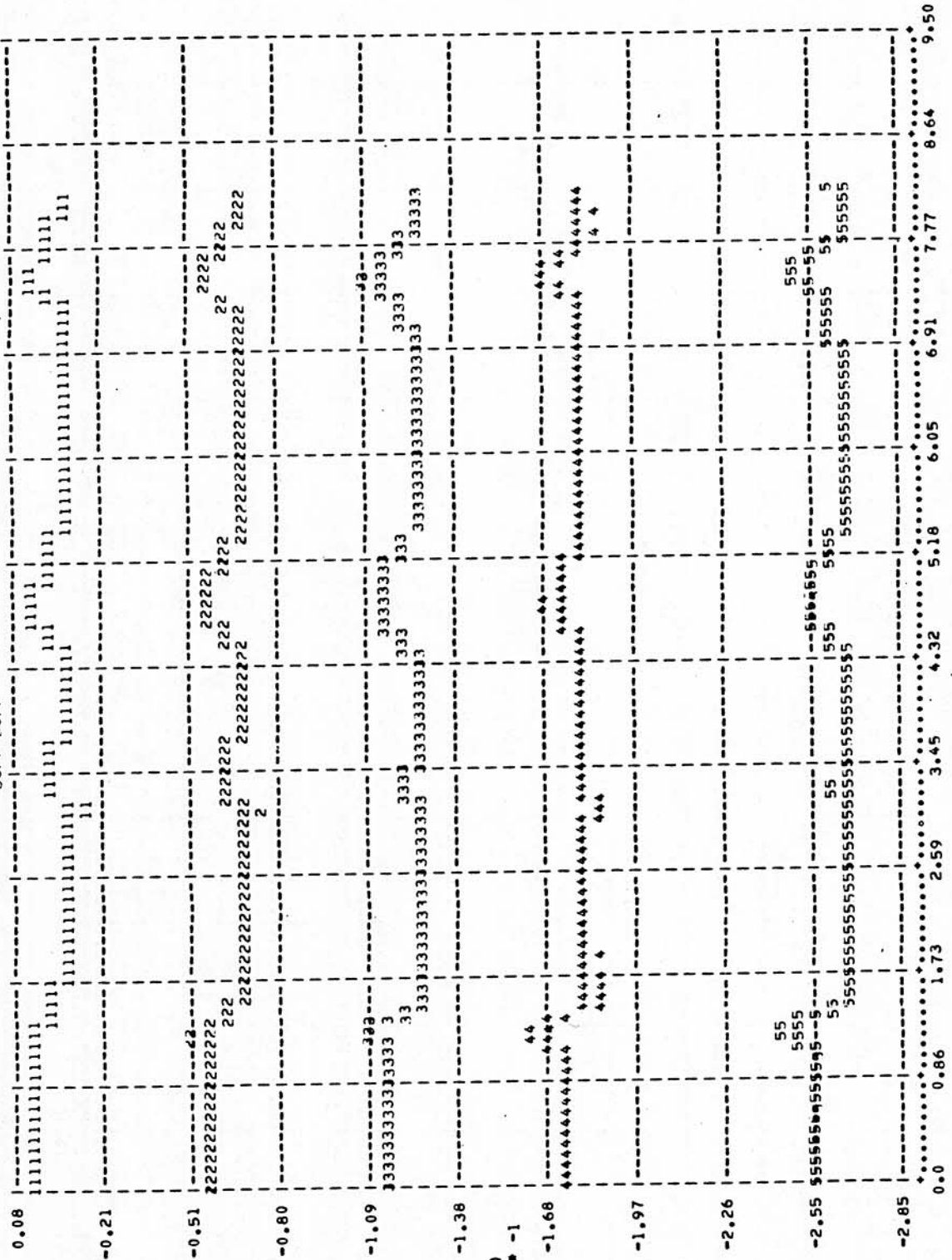
FMC= 0.37631D 02 0.0 0.0 0.14923D 00-0-0.37957D 01 0.0 0.0
 0.20658D 01 0.0 0.0 0.67610D-02 0.25929D 01 0.0
 FRUF=-0.39275D 02 0.0 0.0 0.15236D 02 0.58960D-04 0.44805D-01-0.67672D-01 0.0
 XMR= 0.44511D-02 XMG= 0.19088D 00

TERM	DEFINITION	UNITS
XMG	GUN DISPLACEMENT	FT
XMGD	VELOCITY OF THE GUN	FT/SEC
BLT	BOLT DISPLACEMENT	FT
BLTD	BOLT VELOCITY	FT/SEC
BLTC	BOLT CARRIER DISPLACEMENT	FT
BLTCD	BOLT CARRIER VELOCITY	FT/SEC
BUFF	BUFFER DISPLACEMENT	FT
BUFFD	BUFFER VELOCITY	FT/SEC
BUFW	INTERNAL BUFFER WT. DISPLACEMENT	FT
BUFWD	INTERNAL BUFFER WT. VELOCITY	FT/SEC
HAMR	ROTATION OF THE HAMMER	RAD
HAMRD	ROTATIONAL VELOCITY OF THE HAMMER	RAD/SEC

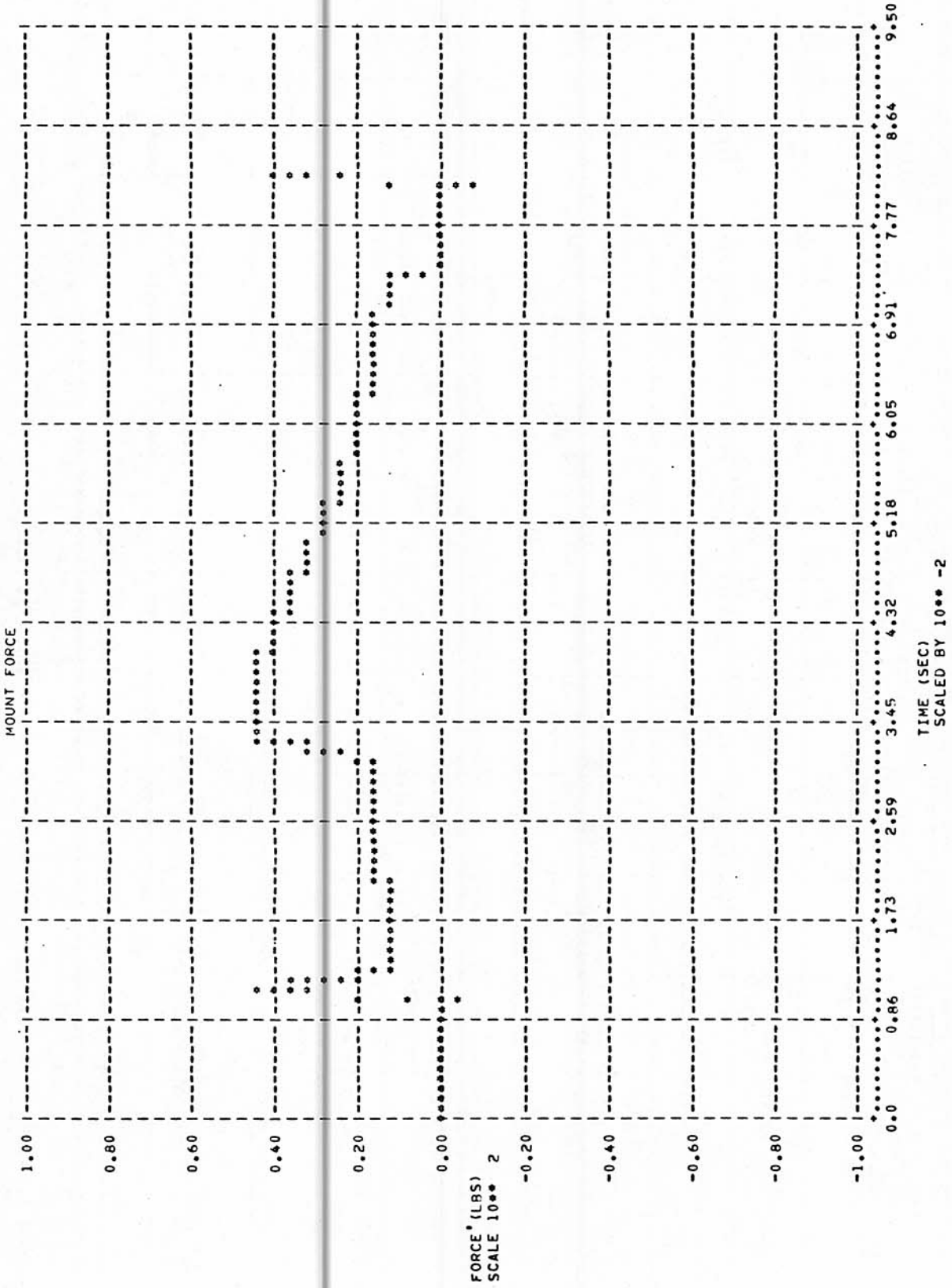
NOTE: BOLT, BOLT CARRIER, AND BUFFER DISPLACEMENTS ARE WITH RESPECT TO THE GUN AND NOT THE INERTIAL REFERENCE FRAME

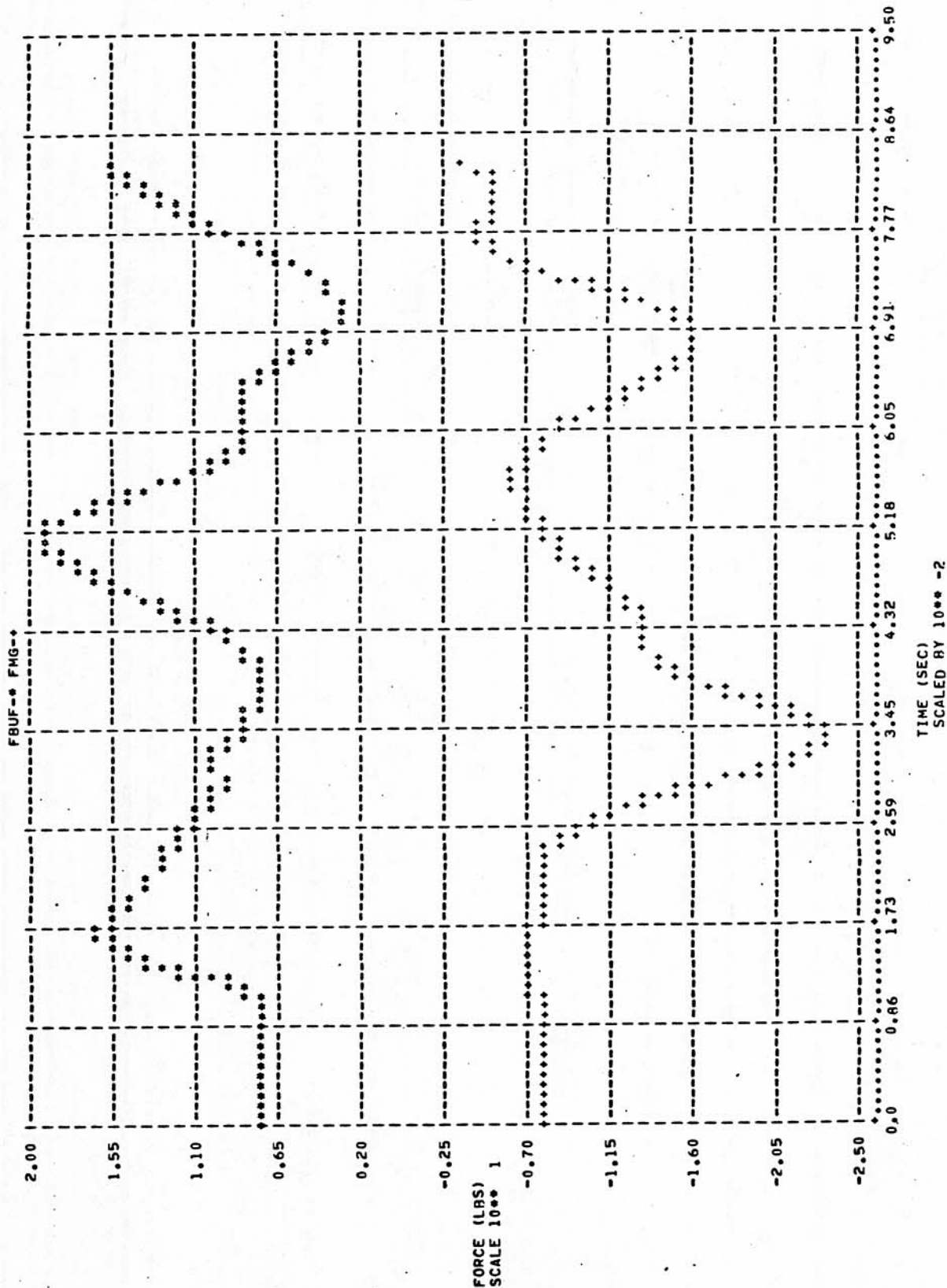
ABSOLUTE MAXIMUMS OF INDIVIDUAL FORCES
 MAX FB= 0.21848D 04 0.26717D 01 0.19305D 03 0.25568D 00 0.43364D 03 0.34970D 02
 0.227636D 02 0.34686D 03 0.69098D 01 0.51978D 03 0.93983D 00 0.17100D-03
 MAX FBC= 0.25568D 00 0.43364D 03 0.34970D 02 0.34686D 03 0.22636D 02 0.31211D 03
 0.37294D 00 0.51655D 02 0.51978D 03 0.26717D 01 0.12059D 01 0.64493D-03
 MAX FMC= 0.43385D 02 0.26717D 01 0.19305D 03 0.37294D 00 0.23069D 02 0.32500D 03 0.26675D 03
 0.47628D 02 0.32591D 03 0.31983D 00 0.67889D-02 0.26717D 01 0.12059D 01 0.32797D 00 0.69098D 01
 MAX FBUF= 0.31211D 03 0.32991D 03 0.19207D 02 0.58960D-04 0.16385D 03 0.32797D 00
 FMC1=-0.42921D 03 FMC2= 0.0
 ZCAP= 3.76849016FT (DEFLECTION OF PROJECTILE)
 X(L1)= 0.00879073RAD (GUN ROTATION)
 X(L1)= 0.29159483RAD/SEC (ROTATIONAL VELOCITY OF THE GUN)

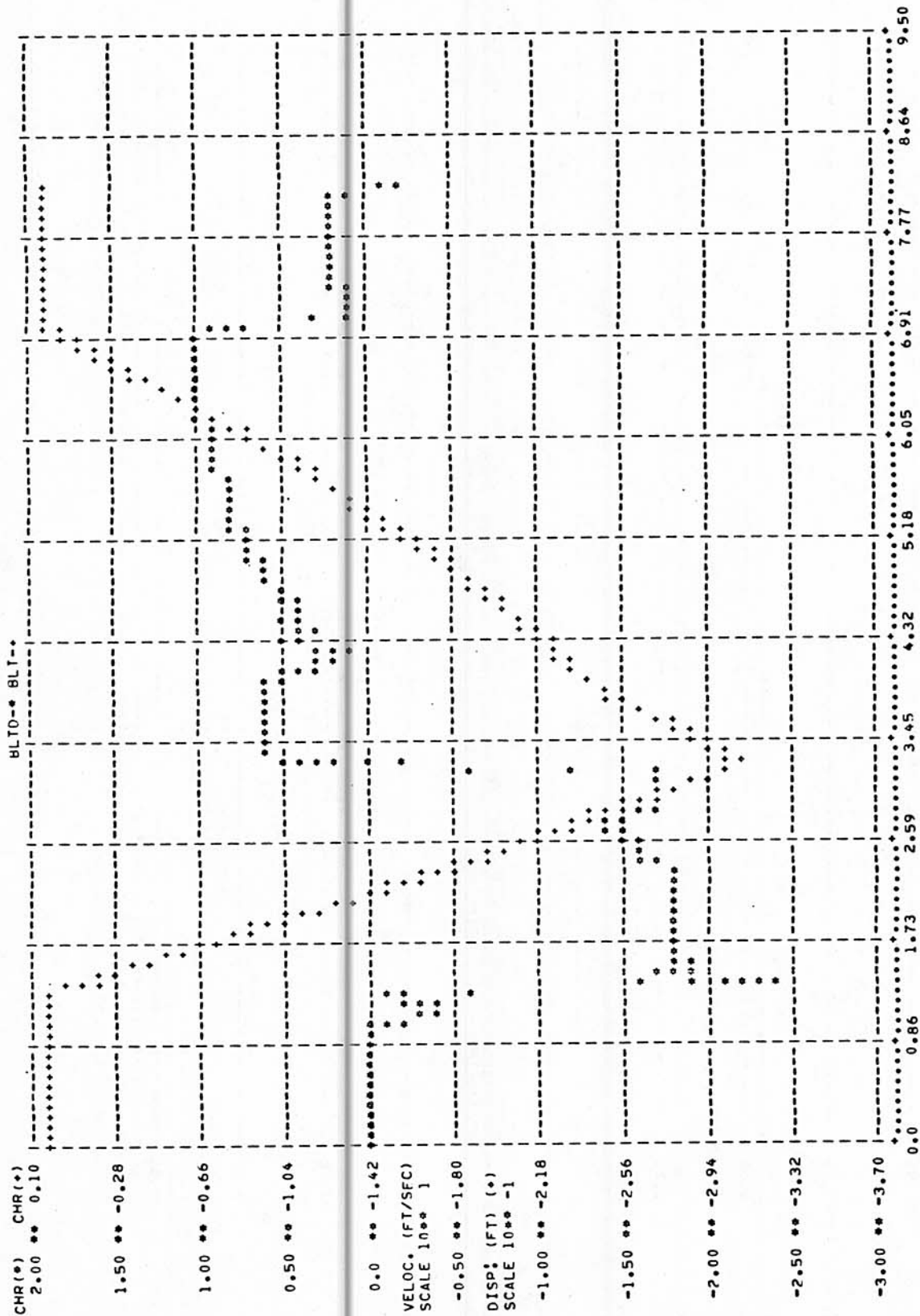
BUFW-BUFF

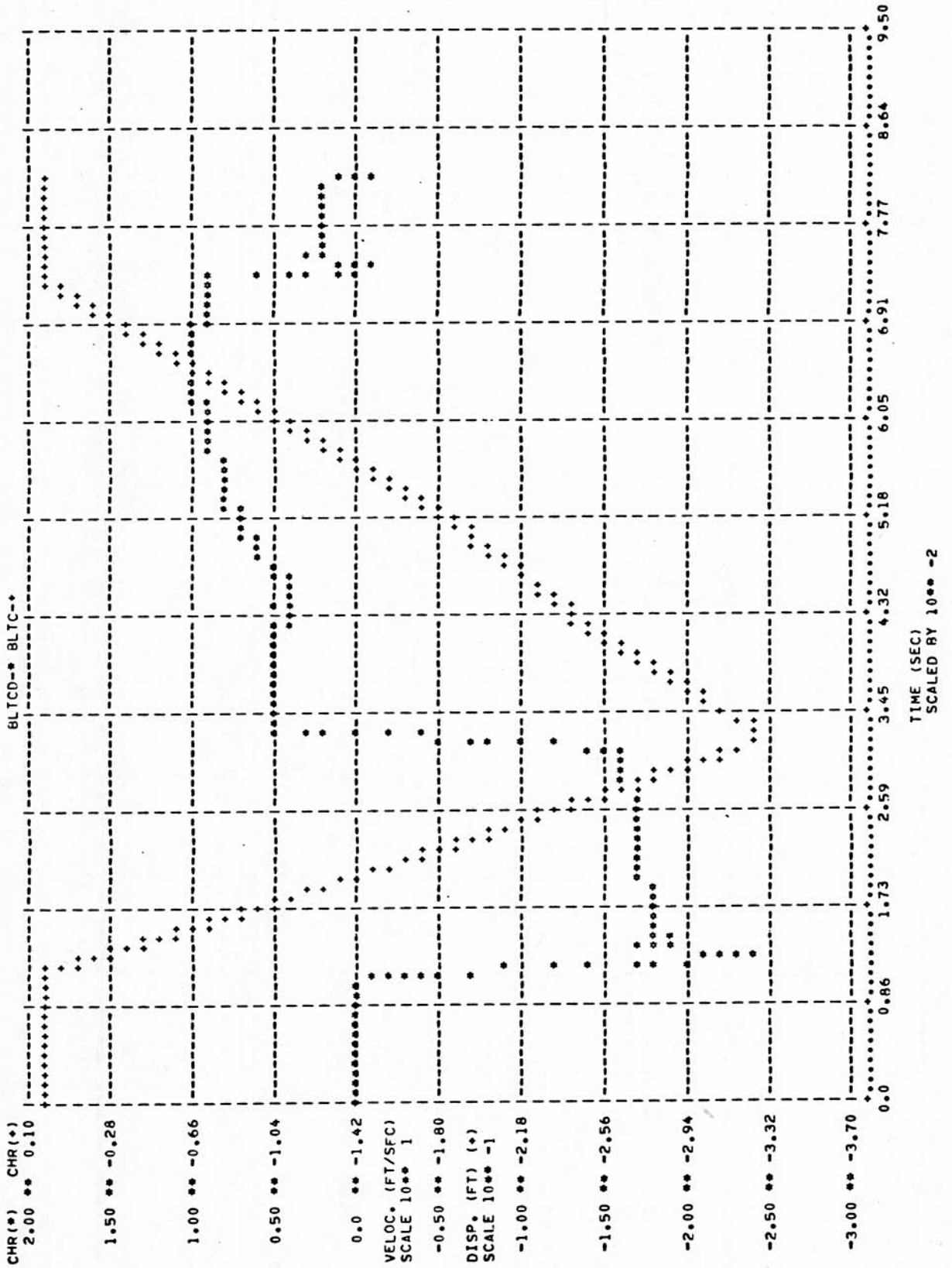


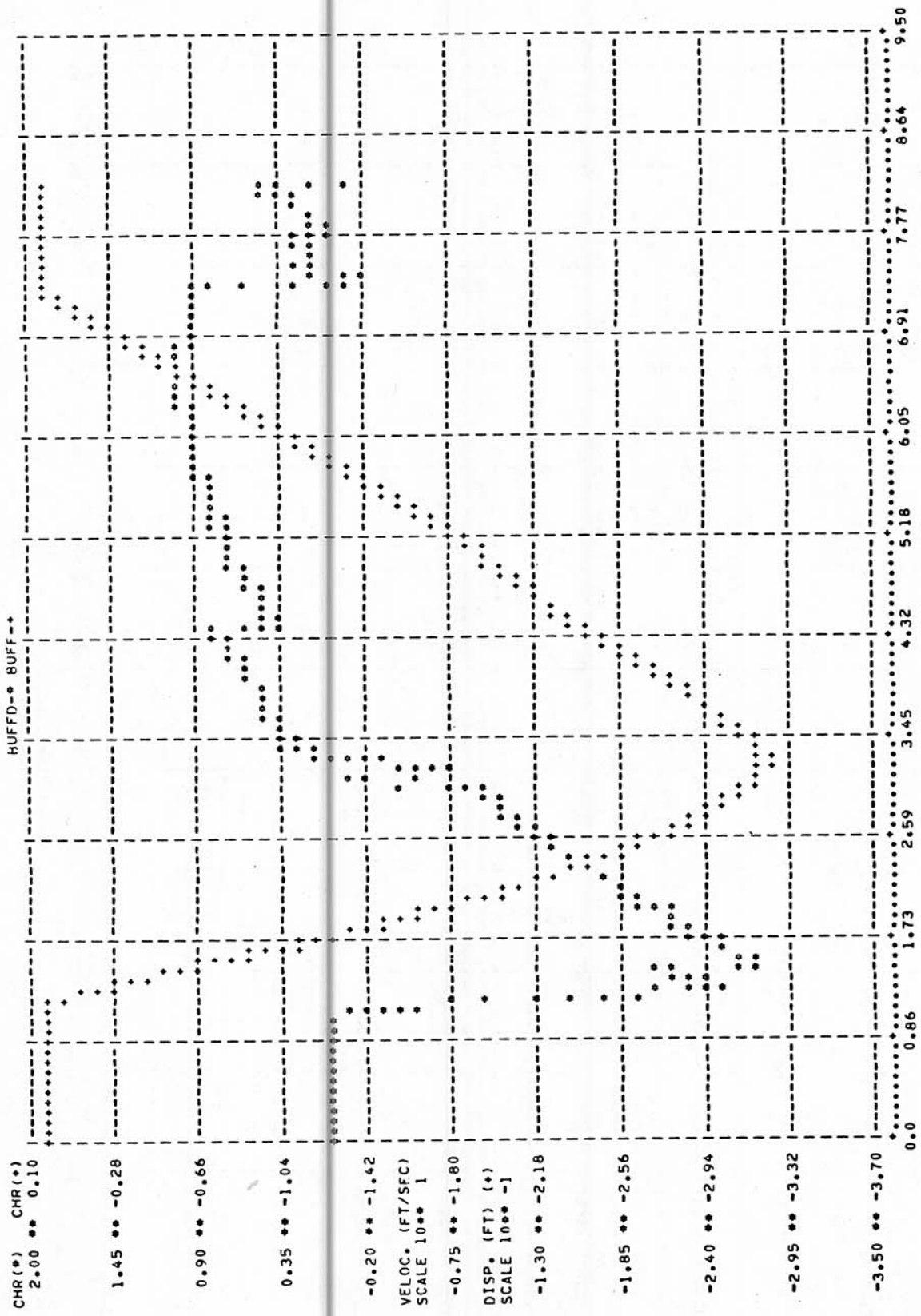
TIME (SEC)
SCALED BY 10** -2

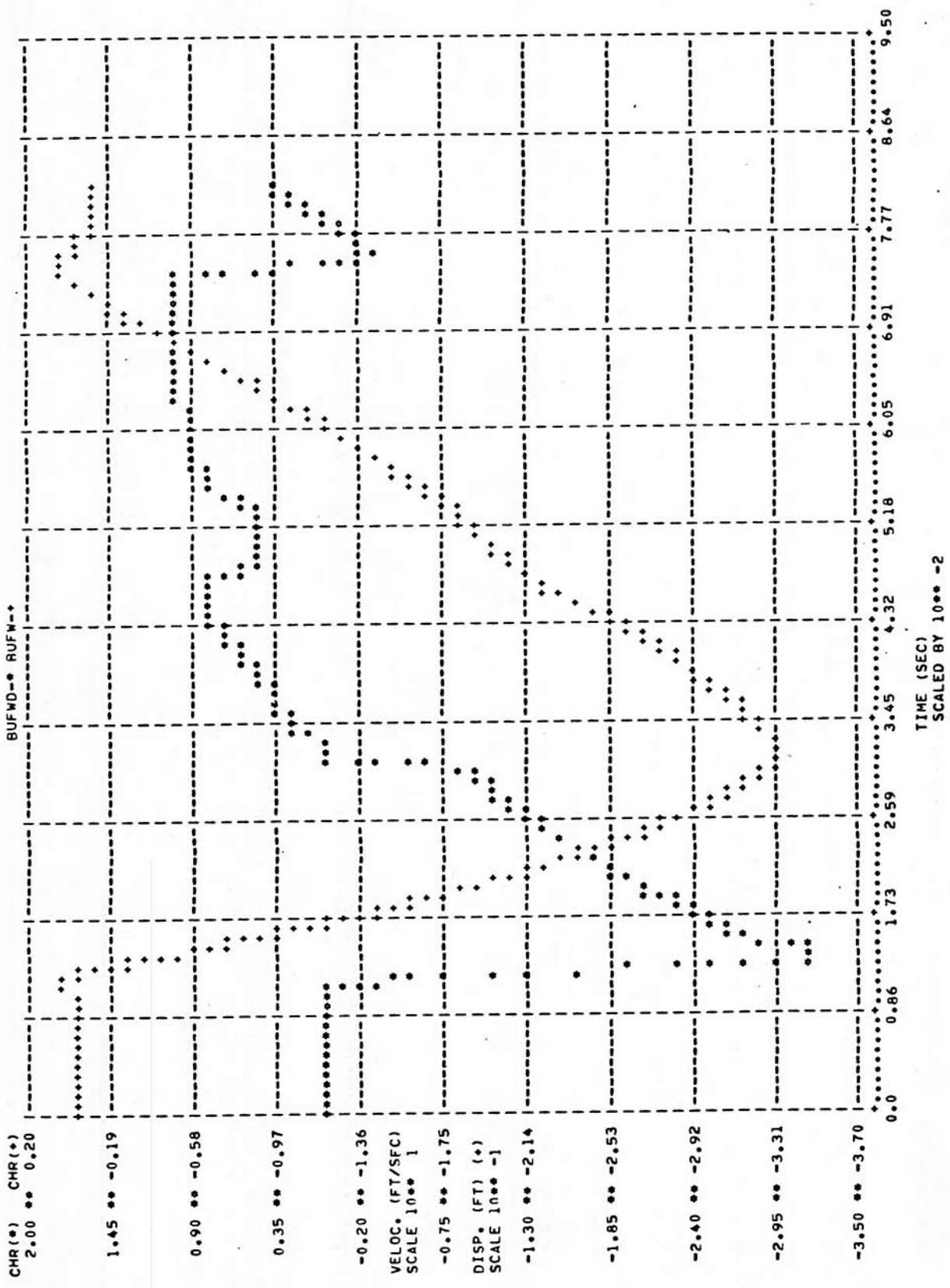


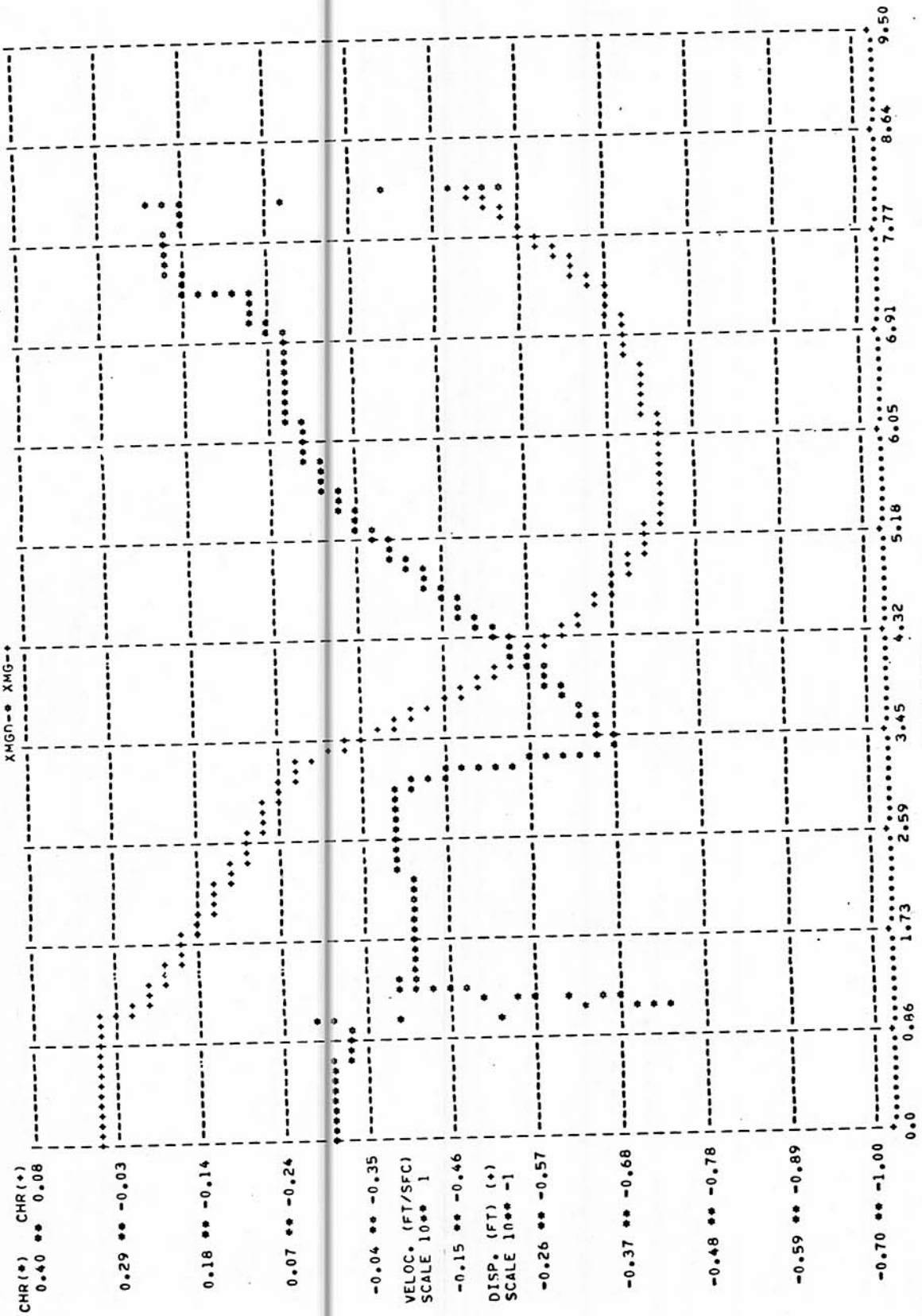


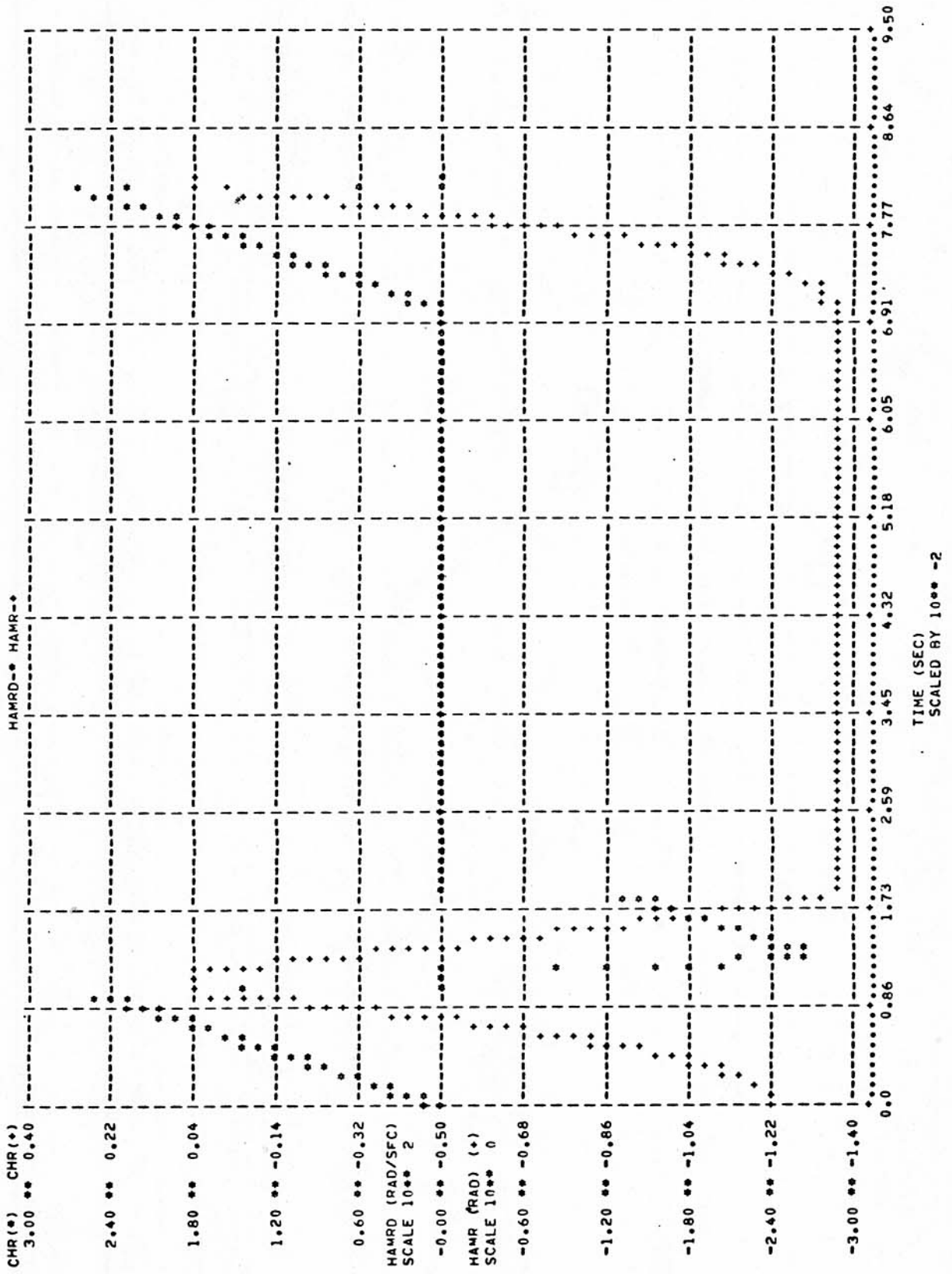












APPENDIX L

COMPUTER PROGRAM FOR SPRING SURGE AND SAMPLE OUTPUT


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FRUF=FN0T
CALL RUFF(RUFKON,RUFCON,RUFM)
PRINT 72, USK, DSC, DSM
72 FORMAT(4X,'A.F T E R A D J U S T M E N T',//3X,'K',13X,'C',13X,
1,'4/23F14.5)
INTEGPR=2 JPL0T(101,51)
CALL PLOT0(JPLOT)
RFAL A4(15), A2(15), A3(15), A4(15), A5(15), A6(15), A7(15)
RFAL A8(15)
RFAD 2,(A4(I),I=1,6)
DO 12 I=1,6
A2(I)=A4(I)
A3(I)=A4(I)
A4(I)=A4(I)
A5(I)=A4(I)
A6(I)=A4(I)
A7(I)=A4(I)
12 A4(I)=A4(I)
CALL PLOT1(A4)
CALL PLOT1(A2)
CALL PLOT1(A3)
CALL PLOT1(A4)
CALL PLOT1(A5)
CALL PLOT1(A6)
CALL PLOT1(A7)
CALL PLOT1(A4)
RFAL TITLE(5), VLAB(4), HLAB(4)
RFAD 9,TITLE
RFAD 9,VLAB
RFAD 9,HLAB
9 FORMAT(5A4)
INTEGPR=2 CH1(1),CH2(12),CH3(13),CH4(14),CH5(15),CH6(16),
* CH7(17),CH8(18)
INTEGPR=2 JPLOT(101,51)
CALL PLOT0(JPLOT)
RFAL HRR(4,15),A(15)
RFAD 2,(HRR(I,I),I=1,6)
DIMENSION IDSPLT(44)
RFAD 3,(IDSPLT(I),I=1,44)
3 FORMAT(4I1)
INTEGPR=2 DSCHAK(44),'A','B','C','D','E','F','G','H','I','J','K',
* 'L','M','N','O','P','Q','R','S','T','U','V','W','X','Y','Z',1,
* 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,22,23,24,25,26,27,28,29,30,31,32,33,34,35,36,37,38,39,40,41,42,43,44,
* 45,46,47,48,49,50,51,52,53,54,55,56,57,58,59,60,61,62,63,64,65,66,67,68,69,70,71,72,73,74,75,76,77,78,79,80,81,82,83,84,85,86,87,88,89,90,91,92,93,94,95,96,97,98,99,100,101,102,103,104,105,106,107,108,109,110,111,112,113,114,115,116,117,118,119,120,121,122,123,124,125,126,127,128,129,130,131,132,133,134,135,136,137,138,139,140,141,142,143,144,145,146,147,148,149,150,151,152,153,154,155,156,157,158,159,160,161,162,163,164,165,166,167,168,169,170,171,172,173,174,175,176,177,178,179,180,181,182,183,184,185,186,187,188,189,190,191,192,193,194,195,196,197,198,199,200,201,202,203,204,205,206,207,208,209,210,211,212,213,214,215,216,217,218,219,220,221,222,223,224,225,226,227,228,229,230,231,232,233,234,235,236,237,238,239,240,241,242,243,244,245,246,247,248,249,250,251,252,253,254,255,256,257,258,259,260,261,262,263,264,265,266,267,268,269,270,271,272,273,274,275,276,277,278,279,280,281,282,283,284,285,286,287,288,289,290,291,292,293,294,295,296,297,298,299,300,301,302,303,304,305,306,307,308,309,310,311,312,313,314,315,316,317,318,319,320,321,322,323,324,325,326,327,328,329,330,331,332,333,334,335,336,337,338,339,340,341,342,343,344,345,346,347,348,349,350,351,352,353,354,355,356,357,358,359,360,361,362,363,364,365,366,367,368,369,370,371,372,373,374,375,376,377,378,379,380,381,382,383,384,385,386,387,388,389,390,391,392,393,394,395,396,397,398,399,400,401,402,403,404,405,406,407,408,409,410,411,412,413,414,415,416,417,418,419,420,421,422,423,424,425,426,427,428,429,430,431,432,433,434,435,436,437,438,439,440,441,442,443,444,445,446,447,448,449,450,451,452,453,454,455,456,457,458,459,460,461,462,463,464,465,466,467,468,469,470,471,472,473,474,475,476,477,478,479,480,481,482,483,484,485,486,487,488,489,490,491,492,493,494,495,496,497,498,499,500,501,502,503,504,505,506,507,508,509,510,511,512,513,514,515,516,517,518,519,520,521,522,523,524,525,526,527,528,529,530,531,532,533,534,535,536,537,538,539,540,541,542,543,544,545,546,547,548,549,550,551,552,553,554,555,556,557,558,559,560,561,562,563,564,565,566,567,568,569,570,571,572,573,574,575,576,577,578,579,580,581,582,583,584,585,586,587,588,589,590,591,592,593,594,595,596,597,598,599,600,601,602,603,604,605,606,607,608,609,610,611,612,613,614,615,616,617,618,619,620,621,622,623,624,625,626,627,628,629,630,631,632,633,634,635,636,637,638,639,640,641,642,643,644,645,646,647,648,649,650,651,652,653,654,655,656,657,658,659,660,661,662,663,664,665,666,667,668,669,670,671,672,673,674,675,676,677,678,679,680,681,682,683,684,685,686,687,688,689,690,691,692,693,694,695,696,697,698,699,700,701,702,703,704,705,706,707,708,709,710,711,712,713,714,715,716,717,718,719,720,721,722,723,724,725,726,727,728,729,730,731,732,733,734,735,736,737,738,739,740,741,742,743,744,745,746,747,748,749,750,751,752,753,754,755,756,757,758,759,760,761,762,763,764,765,766,767,768,769,770,771,772,773,774,775,776,777,778,779,780,781,782,783,784,785,786,787,788,789,790,791,792,793,794,795,796,797,798,799,800,801,802,803,804,805,806,807,808,809,810,811,812,813,814,815,816,817,818,819,820,821,822,823,824,825,826,827,828,829,830,831,832,833,834,835,836,837,838,839,840,841,842,843,844,845,846,847,848,849,850,851,852,853,854,855,856,857,858,859,860,861,862,863,864,865,866,867,868,869,870,871,872,873,874,875,876,877,878,879,880,881,882,883,884,885,886,887,888,889,890,891,892,893,894,895,896,897,898,899,900,901,902,903,904,905,906,907,908,909,910,911,912,913,914,915,916,917,918,919,920,921,922,923,924,925,926,927,928,929,930,931,932,933,934,935,936,937,938,939,940,941,942,943,944,945,946,947,948,949,950,951,952,953,954,955,956,957,958,959,960,961,962,963,964,965,966,967,968,969,970,971,972,973,974,975,976,977,978,979,980,981,982,983,984,985,986,987,988,989,990,991,992,993,994,995,996,997,998,999,1000,1001,1002,1003,1004,1005,1006,1007,1008,1009,1010,1011,1012,1013,1014,1015,1016,1017,1018,1019,1020,1021,1022,1023,1024,1025,1026,1027,1028,1029,1030,1031,1032,1033,1034,1035,1036,1037,1038,1039,1040,1041,1042,1043,1044,1045,1046,1047,1048,1049,1050,1051,1052,1053,1054,1055,1056,1057,1058,1059,1060,1061,1062,1063,1064,1065,1066,1067,1068,1069,1070,1071,1072,1073,1074,1075,1076,1077,1078,1079,1080,1081,1082,1083,1084,1085,1086,1087,1088,1089,1090,1091,1092,1093,1094,1095,1096,1097,1098,1099,1100,1101,1102,1103,1104,1105,1106,1107,1108,1109,1110,1111,1112,1113,1114,1115,1116,1117,1118,1119,1120,1121,1122,1123,1124,1125,1126,1127,1128,1129,1130,1131,1132,1133,1134,1135,1136,1137,1138,1139,1140,1141,1142,1143,1144,1145,1146,1147,1148,1149,1150,1151,1152,1153,1154,1155,1156,1157,1158,1159,1160,1161,1162,1163,1164,1165,1166,1167,1168,1169,1170,1171,1172,1173,1174,1175,1176,1177,1178,1179,1180,1181,1182,1183,1184,1185,1186,1187,1188,1189,1190,1191,1192,1193,1194,1195,1196,1197,1198,1199,1200,1201,1202,1203,1204,1205,1206,1207,1208,1209,1210,1211,1212,1213,1214,1215,1216,1217,1218,1219,1220,1221,1222,1223,1224,1225,1226,1227,1228,1229,1230,1231,1232,1233,1234,1235,1236,1237,1238,1239,1240,1241,1242,1243,1244,1245,1246,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2247,2248,2249,2250,2251,2252,2253,2254,2255,2256,2257,2258,2259,2260,2261,2262,2263,2264,2265,2266,2267,2268,2269,2270,2271,2272,2273,2274,2275,2276,2277,2278,2279,2280,2281,2282,2283,2284,2285,2286,2287,2288,2289,2290,2291,2292,2293,2294,2295,2296,2297,2298,2299,2300,2301,2302,2303,2304,2305,2306,2307,2308,2309,2310,2311,2312,2313,2314,2315,2316,2317,2318,2319,2320,2321,2322,2323,2324,2325,2326,2327,2328,2329,2330,2331,2332,2333,2334,2335,2336,2337,2338,2339,2340,2341,2342,2343,2344,2345,2346,2347,2348,2349,2350,2351,2352,2353,2354,2355,2356,2357,2358,2359,2360,2361,2362,2363,2364,2365,2366,2367,2368,2369,2370,2371,2372,2373,2374,2375,2376,2377,2378,2379,2380,2381,2382,2383,2384,2385,2386,2387,2388,2389,2390,2391,2392,2393,2394,2395,2396,2397,2398,2399,2400,2401,2402,2403,2404,2405,2406,2407,2408,2409,2410,2411,2412,2413,2414,2415,2416,2417,2418,2419,2420,2421,2422,2423,2424,2425,2426,2427,2428,2429,2430,2431,2432,2433,2434,2435,2436,2437,2438,2439,2440,2441,2442,2443,2444,2445,2446,2447,2448,2449,2450,2451,2452,2453,2454,2455,2456,2457,2458,2459,2460,2461,2462,2463,2464,2465,2466,2467,2468,2469,2470,2471,2472,2473,2474,2475,2476,2477,2478,2479,2480,2481,2482,2483,2484,2485,2486,2487,2488,2489,2490,2491,2492,2493,2494,2495,2496,2497,2498,2499,2500,2501,2502,2503,2504,2505,2506,2507,2508,2509,2510,2511,2512,2513,2514,2515,2516,2517,2518,2519,2520,2521,2522,2523,2524,2525,2526,2527,2528,2529,2530,2531,2532,2533,2534,2535,2536,2537,2538,2539,2540,2541,2542,2543,2544,2545,2546,2547,2548,2549,2550,2551,2552,2553,2554,2555,2556,2557,2558,2559,2560,2561,2562,2563,2564,2565,2566,2567,2568,2569,2570,2571,2572,2
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RR(I,R) = A(8)
RR(I,15) = A(15)
15 CONTINUE
INTEGR=2 KPL0T(101,51)
CALL PLOT0(KPLOT)
KFAL AR2(15), AB3(15), AB4(15), AH5(15), AB6(15), AB7(15)
KFAD 2, (AH2(I), I=1,6)
DO 61 I=1,6
AR3(I)=AB2(I)
AR4(I)=AB2(I)
AR5(I)=AR2(I)
AR6(I)=AB2(I)
AR7(I)=AR2(I)
61 CONTINUE
CALL PLOT1(AR2)
CALL PLOT1(AH3)
CALL PLOT1(AR4)
CALL PLOT1(AH5)
CALL PLOT1(AB6)
CALL PLOT1(AH7)
KFAL TITL3(5), VLAB3(4), HLAB3(4)
KFAD 9, TITL3
KFAD 9, VLAB3
KFAD 9, HLAB3
COILS=COILS
100 CONTINUE
NTIME=NTIME+1
IF ( NTIME .LT. NTHRN ) GO TO 150
NTIME=0
CALL XNOW(T,XOP,XOPD,XCORD,YCORD)
CALL SPRING(XDS,XDOTDS,XMG,XBUF(1),XMGD,XDBUF(1),FBUF)
150 CONTINUE
CALL RUFFER(H,XBUF,XDBUF,XRUFDEL,FBUF,XMG,XOP,XMGD,XOPD,FNOT)
T=T+H
NO*PLT=NO*PLT+1
IF ( NOWPLT .LT. NIMPLT ) GO TO 60
NOWPLT=0
PX1=XHUF(1)+H,0
PX2=XHUF(2)+H,0-0.3775
PX3=XHUF(3)+H,0-0.3775-(0.684)*H,0
PX4=XHUF(4)+H,0-0.3775-(0.684)*H,0
PX5=XHUF(5)+H,0-0.3775-(0.684)*H,0
PX6=XHUF(6)+H,0-0.3775-(0.684)*H,0
PX7=XHUF(7)+H,0-0.3775-(0.684)*H,0-0.9565
PX8=PX1-H,0
CALL PLOT2(IPL01,CH1,AA1,T,PX1,0)
CALL PLOT2(IPL01,CH2,AA2,T,PX2,0)
CALL PLOT2(IPL01,CH3,AA3,T,PX3,0)
CALL PLOT2(IPL01,CH4,AA4,T,PX4,0)
CALL PLOT2(IPL01,CH5,AA5,T,PX5,0)
CALL PLOT2(IPL01,CH6,AA6,T,PX6,0)
CALL PLOT2(IPL01,CH7,AA7,T,PX7,0)
CALL PLOT2(IPL01,CH8,AA8,T,PX8,0)
DO 25 I=1,44
IF ( ICSPLI(I) .EQ. 0 ) GO TO 25
DO 26 J=9,15
FU=FU+I
XC0IL = AUS(I) + (COILS + 1.0 - FLOATT)*USIL/COILS
CALL PLOT2(JPLOT,BSCHAR(I),A,I,XCOIL,0)
DO 27 J=9,15

```

```

BRQ(I,J) = A(J)
27 CONTINUE
25 CONTINUE
PY2=PX1-PX2
PY3=PX1-PX3
PY4=PX1-PX4
PY5=PX1-PX5
PY6=PX1-PX6
PY7=PX1-PX7
CALL PLOT2(KPLOT,CH2,AB2,T,PY2,0)
CALL PLOT2(KPLOT,CH3,AB3,T,PY3,0)
CALL PLOT2(KPLOT,CH4,AB4,T,PY4,0)
CALL PLOT2(KPLOT,CH5,AB5,T,PY5,0)
CALL PLOT2(KPLOT,CH6,AB6,T,PY6,0)
CALL PLOT2(KPLOT,CH7,AB7,T,PY7,0)
WRITE (4) T,PY2,PY3,PY4,PY5,PY6,PY7
NRCDS=NRCD5+1
60 CONTINUE
IF ( T.LI. TEND ) GO TO 100
CALL PLOT3(IPLOT,AA1,TITLE,VLAH,HLAB,1)
CALL PLOT3(IPLOT,AA2,TITLE2,VLAB2,HLAB2,1)
CALL PLOT3(IPLOT,AB2,TITLE3,VLAB3,HLAB3,1)
CALL EXIT
END)
SUBROUTINE SPRNGO(N,FL,IL,XUP,XMG,K,C,M,INCR,X,XDOT,F1,XOPDOT)
MFAL K,M,INCR,IL
DIMENSION A(44),XDOT(44)
DO 10 I=1,N
X(I)=0.0
XDOT(I)=0.0
MFALN=N
CAPDEL=FL/(REALN*1.0)
DFL=IL/(REALN*1.0)
DEL1=XOP+(CAPDEL-DEL)
DEL2=XMG-(CAPDEL-DEL)
K=K*(MFALN*1.0)
C=C*(MFALN*1.0)
M=M/MFALN
ALPHA=2.*C
BETA=2.*K
POWER=-ALPHA/(2.0*M)
ETEM=EXP(POWER*INCR)
IF ((BETA/M)-(ALPHA/(2.0*M))**2)61,62,63
63 IFLAG=1
ROOT=SQRT((BETA/M)-(ALPHA/(2.0*M))**2)
COST=COS(ROOT*INCR)
SINT=SIN(ROOT*INCR)
PRINT 903
903 FORMAT(20X,'SPRING IS UNDER DAMPED')
GO TO 60
62 IFLAG=0
PRINT 902
902 FORMAT(20X,'SPRING IS CRITICALLY DAMPED')
GO TO 60
61 IFLAG=-1
ROOT=SQRT((ALPHA/(2.0*M))**2-(BETA/M))
ETEM=EXP(-ROOT*INCR)
ETEMN=EXP(-ROOT*INCR)
PRINT 901
901 FORMAT(20X,'SPRING IS OVER DAMPED')
60 CONTINUE

```

```

F1=K*(XOP-X(1)-DEL1)-C*(XOPDOT-XDOT(1))
N1=N-1
HRETURN
ENTRY SPRING(X,XDOT,XMG,XOP,XMGDOT,XOPDOT,F1)
IF (IFLAG)601,602,603
603 CONTINUE
H=K*(X(2)+XOP+CAPDEL-DEL-DEL1)+C*(XDOT(2)+XOPDOT)
COEF=X(1)-H/BETA
X(1)=ETERM*(COEF*CUSTM+(XDOT(1)+ALPHA*COEF/(2.*M))*SINTM/ROOT)
1 +H/BETA
XDOT(1)=ETERM*(XDOT(1)+COSTM-SINTM*(ALPHA*XDOT(1)+2.*BETA*COEF)/
1 (2.*M*ROOT))
IF (N.EQ.2) GO TO 200
DO 100 I=2,N1
H=K*(X(I-1)+X(I+1))+C*(XDOT(I-1)+XDOT(I+1))
COEF=X(I)-H/BETA
X(I)=ETERM*(COEF*CUSTM+(XDOT(I)+ALPHA*COEF/(2.*M))*SINTM/ROOT)
1 +H/BETA
100 XDOT(I)=ETERM*(XDOT(I)+COSTM-SINTM*(ALPHA*XDOT(I)+2.*BETA*COEF)/
1 (2.*M*ROOT))
200 CONTINUE
H=K*(X(N-1)+XMG-CAPDEL-DEL-DEL2)+C*(XDOT(N-1)+XMGDOT)
COEF=X(N)-H/BETA
X(N)=ETERM*(COEF*CUSTM+(XDOT(N)+ALPHA*COEF/(2.*M))*SINTM/ROOT)
1 +H/BETA
XDOT(N)=ETERM*(XDOT(N)+CUSTM-SINTM*(ALPHA*XDOT(N)+2.*BETA*COEF)/
1 (2.*M*ROOT))
GO TO 9999
602 CONTINUE
H=K*(X(2)+XOP+CAPDEL-DEL-DEL1)+C*(XDOT(2)+XOPDOT)
COEF=X(1)-H/BETA
X(1)=ETERM*(COEF-(POWER*COEF-XDOT(1))*INCR)+H/BETA
XDOT(1)=ETERM*(XDOT(1)-POWER*COEF)*(1.+POWER*INCR)+POWER*COEF
IF (N.EQ.2) GO TO 202
DO 102 I=2,N1
H=K*(X(I-1)+X(I+1))+C*(XDOT(I-1)+XDOT(I+1))
COEF=X(I)-H/BETA
X(I)=ETERM*(COEF-(POWER*COEF-XDOT(I))*INCR)+H/BETA
XDOT(I)=ETERM*(XDOT(I)-POWER*COEF)*(1.+POWER*INCR)+POWER*COEF
202 CONTINUE
H=K*(X(N-1)+XMG-CAPDEL-DEL-DEL2)+C*(XDOT(N-1)+XMGDOT)
COEF=X(N)-H/BETA
X(N)=ETERM*(COEF-(POWER*COEF-XDOT(N))*INCR)+H/BETA
XDOT(N)=ETERM*(XDOT(N)-POWER*COEF)*(1.+POWER*INCR)+POWER*COEF
GO TO 9999
601 CONTINUE
H=K*(X(2)+XOP+CAPDEL-DEL-DEL1)+C*(XDOT(2)+XOPDOT)
COEF1=(XDOT(1)+(XDOT(1)+H/HFTA)-COEF1)
COEF2=(X(1)-H/HFTA)-COEF1
X(1)=ETERM*(COEF1+ETERM*COEF2*ETERM)+H/BETA
XDOT(1)=ETERM*(COEF1+ETERM*(XDOT(1)+POWER*COEF2*ETERM*(XDOT(1)-POWER)
1 )
IF (N.EQ.2) GO TO 201
DO 101 I=2,N1
H=K*(X(I-1)+X(I+1))+C*(XDOT(I-1)+XDOT(I+1))
COEF1=(XDOT(I)+(XDOT(I)+H/HFTA)-COEF1)/(2.*M*ROOT)
COEF2=(X(I)-H/HFTA)-COEF1
X(I)=ETERM*(COEF1+ETERM*COEF2*ETERM)+H/BETA
101 XDOT(I)=ETERM*(XDOT(I)+ETERM*(XDOT(I)+POWER*COEF2*ETERM*(XDOT(I)-POWER)
1 )
201 CONTINUE

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```

H* K*(X(N-1)*XMG-CA*DEL+DEL-DEL2)+C*(XDOT(N-1)*XMGDOT)
COEF1=(XDOT(N)*(XDOT-POWER)*(X(N)-H/HFTA))/(2.*XROOT)
COEF2=(X(N)-H/BETA)-COFF1
X(N)=ETERM*(COEF1*ETERM+COEF2*ETERM)+H/BETA
XDOT(N)=ETERM*(COEF1*ETERM+(XDOT+POWER)-COEF2*ETERM*(XDOT-POWER)
1 )
9999 F1=-K*(XOP-X(1))-DEL1-C*(XOPDOT-XDOT(1))
F2=-K*(XMG-X(N))-DEL2-C*(XMGDOT-XDOT(N))
RETURN
END
SUBROUTINE XNOW(T,XOP,XOPD,X,Y)
DIMENSION X(20), Y(20)
I=1
CALL LINEAR(T,X,Y,XOP,I)
XOPD=(Y(I+1)-Y(I))/(X(I+1)-X(I))
RETURN
END
SUBROUTINE LINEAR (A,X,Y,VV,I)
DIMENSION X(20),Y(20)
2 IF(A-X(I)) 3,1,1
1 I=I+1
GO TO 2
3 I=I-1
VV=Y(I)*(A-X(I+1))/(X(I)-X(I+1))+Y(I+1)*(A-X(I))/(X(I+1)-X(I))
RETURN
END
SUBROUTINE HUFF(BK,BC,BM)
DIMENSION BK(4), HC(4), BM(3)
HFAL K(9), KON(9), C(9), CON(9), M(7)
M(1)=BK(1)
M(7)=BK(3)
KON(1)=HK(1)
CON(1)=HC(1)
KON(7)=HK(4)
CON(7)=HC(4)
KON(8)=BK(3)
KON(9)=HK(3)
CON(9)=HC(3)
CON(8)=HC(3)
DO I I=2,6
M(I)=BK(I)
KON(I)=HK(I)
CON(I)=HC(I)
1 CON(I)=HC(I)
RETURN
ENTRY RUFFER(INCH,X,XDOT,DEL,FBUF,XMG,XOP,XMGD,XOPD,FN01)
HFAL INCH
DIMENSION X(7), XDOT(7), DEL(9)
FOFT=FRUF-FWOT
CALL ZFRU(K,C)
IF( XOP-X(1) ) .LE. DEL(1) ) K(1)=KON(1)
IF( XOP-X(1) ) .LE. DEL(1) ) C(1)=CON(1)
IF( X(1)-X(2) ) .LE. DEL(2) ) K(2)=KON(2)
IF( X(1)-X(2) ) .LE. DEL(2) ) C(2)=CON(2)
IF( X(7)+DEL(4) ) .LE. X(1) ) K(8)=KON(8)
IF( X(7)+DEL(4) ) .LE. X(1) ) C(8)=CON(8)
IF( XMG-X(1) ) .GE. DEL(9) ) K(9)=KON(9)
IF( XMG-X(1) ) .GE. DEL(9) ) C(9)=CON(9)
ALPHA=C(1) + C(2) + C(4) + C(4)
HFTA = K(1) + K(2) + K(4) + K(4)
POWERK=-ALPHA/DEL(1)
ETERM=EXP(POWERK*INCH)

```

```

IF ( BETA/M(I) - (ALPHA/(2.*M(I)))**2) 11.12.13
11 ROOT=SQRT((ALPHA/(2.*M(I)))**2 - BETA/M(I))
ETERMP=EXP(ROOT*INCH)
ETERMNE=EXP(-ROOT*INCH)
H=FOOT-K(I)*K(9)*(XMG-DEL(9))+C(9)*XOPD +K(2)*(DEL(2)*X(2))+C(2)*
1 XDOT(2)+K(9)*(XMG-DEL(9))+C(9)*XMGD +K(8)*(X(7)+DEL(8))+C(8)*
2 XDOT(7)
COEFF1=(XDOT(1)+(ROOT-POWER)*(X(1)-H/HFTA))/(2.*ROOT)
COEFF2=X(1)-H/HFTA-COEF1
X(1)=ETERM*(COEFF1+ETERMP*(ROOT+POWER)-COEFF2+ETERM*(ROOT-POWER)
1 )
GO TO 20
12 HFTA=HFTA*10.**(-5)
13 ROOT=SQRT(HFTA/M(I) - (ALPHA/(2.*M(I)))**2)
COSTM=COS(ROOT*INCH)
SINTM=SIN(ROOT*INCH)
H=FOOT-K(I)*K(9)*(XMG-DEL(9))+C(9)*XOPD +K(2)*(DEL(2)*X(2))+C(2)*
1 XDOT(2)+K(9)*(XMG-DEL(9))+C(9)*XMGD +K(8)*(X(7)+DEL(8))+C(8)*
2 XDOT(7)
COEF=X(1)-H/HFTA
X(1)=ETERM*(COEF*COSTM+(XDOT(1)+ALPHA*COEF/(2.*M(I)))*SINTM/ROOT)
1 +H/HFTA
XDOT(1)=ETERM*(XDOT(1)*COSTM-SINTM*(ALPHA*XDOT(1)+2.*HETA*COEF)
1 / (2.*M(I)*ROOT)
20 CONTINUE
DO 30 I=2,6
CALL ZFRO(K,C)
IF( X(I)-X(I) ) .LE. DEL(I) ) K(I)=KON(I)
IF( X(I)-X(I) ) .LE. DEL(I) ) C(I)=CON(I)
IF( X(I)-X(I+1) ) .LE. DEL(I+1) ) K(I+1)=KON(I+1)
IF( X(I)-X(I+1) ) .LE. DEL(I+1) ) C(I+1)=CON(I+1)
ALPHA=C(I)+C(I+1)
HFTA=X(I)+X(I+1)
POWER=-ALPHA/(2.*M(I))
ETERM=EXP(POWER*INCH)
ETERMNE=EXP(-POWER*INCH)
IF ( HFTA/M(I) - (ALPHA/(2.*M(I)))**2) 21.22.23
21 ROOT=SQRT((ALPHA/(2.*M(I)))**2 - HETA/M(I))
ETERMP=EXP(ROOT*INCH)
ETERMNE=EXP(-ROOT*INCH)
H=K(I)*K(9)*(X(I)-DEL(I))+C(I)*XDOT(I-1)+K(I+1)*(X(I+1)+DEL(I+1))
1 +C(I+1)*XDOT(I+1)
COEF1=(XDOT(I)+(ROOT-POWER)*(X(I)-H/HFTA))/(2.*ROOT)
COEFF2=X(1)-H/HFTA-COEF1
X(1)=ETERM*(COEFF1+ETERMP*(ROOT+POWER)-COEFF2+ETERM*(ROOT-POWER)
1 )
XDOT(1)=ETERM*(COEFF1+ETERMP*(ROOT+POWER)-COEFF2+ETERM*(ROOT-POWER)
1 )
GO TO 30
22 HFTA=HFTA*10.**(-5)
23 ROOT=SQRT(HFTA/M(I) - (ALPHA/(2.*M(I)))**2)
COSTM=COS(ROOT*INCH)
SINTM=SIN(ROOT*INCH)
H=K(I)*K(9)*(X(I)-DEL(I))+C(I)*XDOT(I-1)+K(I+1)*(X(I+1)+DEL(I+1))
1 +C(I+1)*XDOT(I+1)
COEFF=X(1)-H/HFTA
X(1)=ETERM*(COEF*COSTM+(XDOT(1)+ALPHA*COEF/(2.*M(I)))*SINTM/ROOT)
1 +H/HFTA
XDOT(1)=ETERM*(XDOT(1)*COSTM-SINTM*(ALPHA*XDOT(1)+2.*HETA*COEF)
1 / (2.*M(I)*ROOT)
30 CALL ZFRO(K,C)

```

```

IF ( X(6)-X(7) .LE. DEL(7) ) K(7)=KON(7)
IF ( X(6)-X(7) .LE. DEL(7) ) C(7)=CON(7)
IF ( X(7)+DEL(8) .LE. X(1) ) K(8)=KON(8)
IF ( X(7)+DEL(8) .LE. X(1) ) C(8)=CON(8)
ALPHA=C(7)+C(8)
BETA=K(7)+K(8)
POWER=-ALPHA/(2.*M(7))
ETERM=EXP(POWER*INCH)
IF ( HFTA/M(7) - (ALPHA/(2.*M(7)))**2 - BETA/M(7))
31 ROOT=SQRT((ALPHA/(2.*M(7)))**2 - BETA/M(7))
ETERM=EXP(ROOT*INCH)
ETERMN=EXP(-ROOT*INCH)
H=K(7)*(X(6)-DEL(7))+C(7)*XDOT(6)-K(8)*(DEL(8)-X(1))+C(8)*XDOT(1)
COEF1=(XDOT(7)+H*ROOT*POWER)*(X(7)-H/HFTA)/(2.*ROOT)
COEF2=X(7)-H/HFTA-COEF1
X(7)=ETERM*(COEF1+ETERM*COEF2+ETERMN)+H/BETA
XDOT(7)=ETERM*(COEF1+ETERM*COEF2+ETERMN*(ROOT-POWER)
1 )

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31

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GO TO 40
32 HFTA=BETA*10.**(-5)
33 ROOT=SQRT(BETA/M(7) - (ALPHA/(2.*M(7)))**2)
COSTM=COS(ROOT*INCH)
SINTM=SIN(ROOT*INCH)
H=K(7)*(X(6)-DEL(7))+C(7)*XDOT(6)-K(8)*(DEL(8)-X(1))+C(8)*XDOT(1)
COEF=X(7)-H/BETA
X(7)=ETERM*(COEF+COSTM*(XDOT(7)+ALPHA*COEF/(2.*M(7)))+SINTM/ROOT)
1 *H/BETA
XDOT(7)=ETERM*(XDOT(7)+COSTM-SINTM*(ALPHA*XDOT(7)+2.*BETA*COEF)
1 / (2.*M(7)*ROOT)
/ 40 CONTINUE
RETURN
END)
SUBROUTINE ZERO(X,Y)
DIMENSION X(9), Y(9)
DO 1 I=1,9
X(I)=0.
Y(I)=0.
RETURN
END)

```

/ 40 CONTINUE

```

//60.SYSIN 00 *
20
0.0 0.0 9.7 18.6 11.6 21.2 13.1 23.0
16.2 26.7 28.1 45.0 38.1 60.0 45.6 72.0
52.1 80.5 69.1 97.9 71.1 47.3 73.1 96.9
74.1 96.6 87.1 103.1 103.1 73.5 117.6 60.0
129.1 49.0 138.1 39.0 173.1 0.0 99990000. 0.0
10. 0.0 32.0 -123.14 0.20 .00001 50. 0.
50. 0. 50. 0. 0.0 0.0 0.0 0.0 0.13
.0002570 .0001057 .00005449
32.0 0.0 0.0
13.8125
44 0.0 12.2 8.16 0.0003364
10
1
0.0 0.2
RUFFLE MOTION
DISPLACEMENT
TIME
0.0 0.2 -0.5 9.50 0.3 0.0

```

//60.SYSIN 00 *

0.0	0.0	9.7	18.6	11.6	21.2	13.1	23.0
16.2	26.7	28.1	45.0	38.1	60.0	45.6	72.0
52.1	80.5	69.1	97.9	71.1	47.3	73.1	96.9
74.1	96.6	87.1	103.1	103.1	73.5	117.6	60.0
129.1	49.0	138.1	39.0	173.1	0.0	99990000.	0.0
10.	0.0	32.0	-123.14	0.20	.00001	50.	0.
50.	0.	50.	0.	0.0	0.0	0.0	0.13
.0002570	.0001057	.00005449					
32.0	0.0	0.0					
13.8125							
44	0.0	12.2	8.16	0.0003364			
10							
1							
0.0	0.2	-2.	8.0	0.0	0.0		
RUFFLE MOTION							
DISPLACEMENT							
TIME							
0.0	0.2	-0.5	9.50	0.3	0.0		

```

1 1 1 1 1 1 1
SPRING MOTION
DISPLACEMENT
TIME
0.0 0.0 0.0 5.0 0.0 0.0
RELATIVE MOTION
DISPLACEMENT
TIME
/*
//FT04F001 DD USN=6TAPE,UNIT=231,SPACE=(32*(1000,100)),
// DCH=(RECFM=VHS,LRECL=32,BLKSIZE=324),DISP=(NEW,DELETE)
/*

```


HUFFER MOTION

DISPLACEMENT	MULT. BY 10** 0	0.00	0.02	0.04	0.06	0.08	0.10	0.12	0.14	0.16	0.18	0.20
8.00												
7.00												
6.00												
5.00												
4.00												
3.00												
2.00												
1.00												
-0.00												
-1.00												
-2.00												

SPRING MOTION

DISPLACEMENT	MULT. BY 10** 0	0.00	0.02	0.04	0.06	0.08	0.10	0.12	0.14	0.16	0.18	0.20
		TIME										
9.50												
8.50												
7.50												
6.50												
5.50												
4.50												
3.50												
2.50												
1.50												
0.50												
-0.50												

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Technical Report # R-TR-75-010

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Theory and Application of Mathematical Modeling of
Shoulder-Fired Weapons
Part I of II, Final Report

Prepared By: Paul E. Ehle and Albert E. Rahe
Security Class. (of this report): Unclassified
Technical Report R-TR-75-010

243 Pages, Incl Figures

UNCLASSIFIED

1. M16 Rifle
 2. Dynamics
 3. Simulation
 4. Man Weapon
 5. Kinematics
 6. Mathematical Model
- I. Paul E. Ehle and Albert E. Rahe
II. Rock Island Arsenal
III. Research Directorate
General Thomas J. Rodman Laboratory
Rock Island Arsenal

DISTRIBUTION

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These allow a total of eleven degrees of freedom plus a nearly unlimited number of degrees of freedom for the drive spring. To provide accuracy data, one rotational degree of freedom is allowed for weapon pitch motion. Expressions are derived for the many diverse forces acting on the masses.

The resulting equations are solved on an IBM 360/65 digital computer. A sensitivity analysis is conducted, and the results are compared with those from a similar model of the XM19 Rifle. The greatest differences between the M16A1 and the XM19 sensitivities were found to be the effect of the ignition delay and the drive spring on cycle time and the mounting conditions on accuracy.

The methods developed during model construction are applicable to the modeling of many other weapons. Part II of this two-report series is an application of these techniques to the XM19 Rifle.

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